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## Introduction

In recent years the use of timber as construction material has become quite common in all Europe. This is in general a preferred choice in all those applications where the structure has to fit well in the surrounding landscape. Moreover different studies demonstrated that wood in general has a good thermal and acoustic properties. Moreover timber elements are eco-friendly: they fit very well in all those applications where the issues of pollution are sensitive or needs an *ad hoc* design approach. Thus, there is a lot of application of timber all around the world for the so called "green constructions".

In this work a particular branch of timber constructions is discussed and analyzed: the pedestrian bridges. The motivations behind this work can be summarized in two main aspects:

- this research field is quite young and different research groups are working within the footbridges area. In particular several works in the literature are focused on comfort, vibrations control, and so forth...;
- 2) the design of pedestrian bridges is analyzed in this work to better identify the dynamics of these structures, trying to identify a new design scheme for those loads that affect the behaviour of these particular structures, the so called Human Induced Loads (HIL).

The development of new techniques and construction materials, as for example the glued laminated timber (GLT) within the field of timber bridges, allows the designers to conceive footbridges with span of 100 m long or more, "without" problems in terms of safety but leading to specific performance considerations.

In this thesis, the *in situ* measured structural responses of two existing timber footbridges are reported, analyzed and discussed. These records are used in order to better understand the dynamic behaviour of this type of structures. An attempt to

develop a numerical model able to simulate the interaction among the structure and the human induced loads in a realistic manner is the next step. Finally different vibrations control solutions, under wind and pedestrian loads, are numerically implemented and their performance is discussed.

The thesis is organized in seven chapters plus four appendixes. The topics of the single chapters are:

- Chapter 1 State of the art for the design of footbridges: the main recommendations and prescriptions within the field of the pedestrian bridge are also considered;
- Chapter 2 Design requirements and performance satisfaction: code and literature reviews concerning the dynamic of the bridge and the comfort of the footbridges are provided;
- Chapter 3 Numerical modelling: the numerical models of the case studies are reported;
- *Chapter 4* Action modelling: the loads assigned in the numerical simulations are presented;
- *Chapter 5* Model order reduction (MOR): the theory behind MOR is briefly presented and discussed. Moreover the main results achieved by applying this technique are emphasized;
- *Chapter* 6 Control solutions: the vibration control solutions implemented are introduced and their effect evaluated;

Chapter 7 – Conclusions.

## Chapter 1 State of the art for the design of footbridges

In last years, developments of new high strength materials and construction technology, have introduced a growing trend towards the construction of lightweight and slender pedestrian bridges (see [1]-[3] among others). This slenderness pointed out how these structures are quite sensitive to dynamic aspects. This is mainly due to their reduced mass that, during the action of the dynamic forces, can increase the amplitude of the vibrations.

Probably the most world-wide famous case, within the footbridges in which the vibrations caused several to the users, is the "Millennium Bridge" in London (Figure 1.1).



Figure 1.1. The Millennium Bridge.

As well documented, during the opening celebration day, the bridge was crossed by about 90000 people, with up to 2000 people simultaneously on the bridge (resulting in a maximum density between 1.3 and 1.5 persons per square metre) [4]. Unexpected excessive lateral vibrations of the bridge occurred showing as this structure was suffering a lack of stiffness. Excessive vibration did not occur continuously, but built up when a large number of pedestrians were on the affected spans of the bridge and

decayed when the number of people on the bridge reduced, or the persons stopped their walking. This can be explained by a phenomenon known as *lock-in* effect: that pedestrian(s), disturbed by the vibration of the footbridge, tends to synchronize its (their) step with the natural frequency of the bridge. In the case of the Millennium bridge the problem was solved by a retrofit adopting 37 fluid-viscous dampers to control the horizontal movements and 52 tuned mass dampers to control the vertical movement.

The increase of vibration problems in modern footbridges shows that footbridges should be no longer designed for static loads only (as required by the Italian code [5]), but their dynamic behaviour has also to be accounted by the designer. Indeed the lateral vibration problems of the Millennium Bridge is not so unusual; in fact recent studies (see [6]-[7] among others) showed as any bridge with lateral frequency modes of less than 1.3 Hz, and sufficiently low mass have the same phenomenon with sufficient pedestrian loading<sup>1</sup>.

## 1.1 Scientific papers

Different research group all around the world are currently working in one or more fields related with footbridges: design, monitoring of existing structures, serviceability evaluation (comfort evaluation), structural control, Human Induced Vibration and/or Loads (HIV – HIL) and so forth...

The purpose of this section is to give a general review, but it is worth noticing that the attention is only focused on recent reviews and contributions on this theme to international conferences. From the references lists of them it is possible to find all the precedent literature in this field.

### 1.1.1 Footbridge Dynamics

With reference to the structural opportunities offered by the use of timber as a construction material, an overview is in [6]. In this paper the authors analyses the main typologies of structural scheme for timber footbridges giving a general review of the traditional past applications and suggestions for possible developments in this field.

For the evaluation of the serviceability conditions for a pedestrian bridge, the reader is referred to reference [7]. The authors of this review give an evaluation of the

<sup>&</sup>lt;sup>1</sup> The greater the number of people, the greater the amplitude of the vibrations

methodology proposed by the recent European guidelines HiVoSS (for steel structures) and the French guidelines Sétra, widely applied in practice for comfort assessment: the discussion is based on a selection of eight slender footbridges. As a result, the above guidelines are highly sensitive to small variations in the predicted natural frequencies. Based on a series of in situ experimental investigations, the author of [8] presents a new vibration comfort criteria for footbridges (Figure 1.1.1).

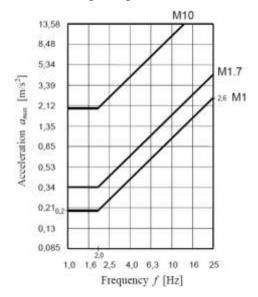


Figure 1.1.1. Vibration comfort criteria for footbridges in case of human induced vibrations [8].

Moving to the monitoring of timber bridges, different papers have been published. Indeed timber is very sensitive to the environmental changes, e.g. humidity, temperature and so on (see [9] and [10] among others). In reference [11] an *ah doc* system of devices for the monitoring of the footbridge is designed with the intent of evaluating how the changes in the environment conditions can affect the modal response of the structure. The duration of the data acquisition (1 year) and the number of sensors employed are significant aspects of the report in [12]. In particular the monitoring system is made of 18 tri-axial accelerometers, 9 at each side of the deck, a temperature sensor and an anemometer located along the deck (Figure 1.1.2). From these data the changes of the dynamic properties due to the temperature variation were analyzed. Moreover different studies on the comfort were developed on the basis of real data.

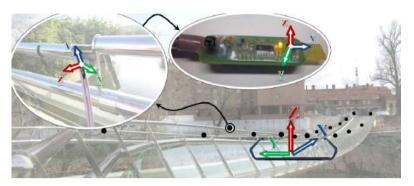


Figure 1.1.2. Monitoring system of the Pedro Gómez Bosque footbridge [12].

The investigation of the aspects related to the particular shape of the footbridge, is reported in [13]. The footbridge shows a ring form with outer radius of about 40 m, service width of 4 m, raised about 5,5 m over the crossroad level (Figure 1.1.3).



Figure 1.1.3. On the left: general view of the footbridge. On the right: testing under human induced vibration [13].

The proper design of the monitoring system is investigated in [14]. The authors addresses the layout optimization of a Wireless Sensors Network for a footbridge monitoring system considering both the network energy consumption and the sensor placement quality, in order to best capture the footbridge properties. This work shows as a proper design (calibrated for that particular structure) is needed in order to achieve the best result from a monitoring campaign.

Moving to the structural control of footbridges, several papers can be found in the literature (see [15] and [16] among others). An example of passive control is presented in [17]. This paper describes a study of the human induced dynamic effects on the new footbridge over the Ave River, in Santo Tirso, Portugal. The study outlines the need of installing a system of two tuned mass dampers (TMD), with total mass of 2100kg, in order to mitigate the induced vibrations. A new set of tests on the system mounting the devices assess the effectiveness of the mitigation system in increasing the comfort level.

This experience pointed out as the role of the damping is essential to mitigate the vibration; in particular shows as the measured damping ratio is usually below the one theoretically estimated. Thus, it is important to be very carefully in the estimation of the damping. In [18] the authors implemented five MR-TMD control devices at the middle span of the footbridge to be controlled under wind pressure. In particular a semi-active strategy is adopted. The controllable Coulomb force is adjusted in a manner that its value does not exceed the nominal maximum force of the MR damper but it is higher than the nominal minimum force of the MR damper. The results show that the semi-active control have a better control, in terms of acceleration performance, compared with the classical passive control solution.

A further paper covers the active structural control is reported in [19]. The design and experimental implementation of a passive tuned mass damper (TMD) and of an active mass damper (AMD) to mitigate human induced vibrations on an in-service footbridge is discussed. The TMD is designed based on the  $H_{\infty}$  control theory. The AMD consists of a proof - mass actuator controlled by a strategy based on acceleration feedback with a phase-lag network. The main features of the control strategy are summarized in Figure 1.1.4. As shown in this scheme the output of the system is the structural acceleration since it is convenient to measure it. The authors conclude with one remark: compared with passive control solutions, the cost of an active solution is much higher. In the particular case of the two strategies studied in [19], the authors estimated that the implementation of the AMD solution (i.e. AVC hardware and installation) costs about three or four times more than the passive solution (TMD).

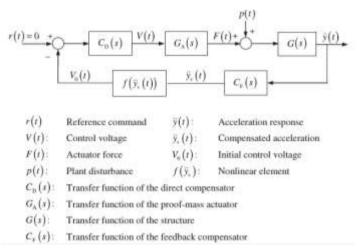


Figure 1.1.4. Active control scheme [19].

#### 1.1.2 Human Induced Vibration (HIV)

Consider now the topic of the Human Induced Vibration (HIV) [20]. The paper [21] deals with the possibility of introducing simplified procedures to evaluate the maximum dynamic response of footbridges due to a realistic loading scenario. Thus, a new nondimensional approach is introduced to identify the essential non-dimensional parameters governing the dynamic behaviour under different loadings. Finally two simplified procedures based on the definition of two coefficients, the Equivalent Amplification Factor (EAF) and the Equivalent Synchronization Factor (ESF), are proposed with the aim of assessing the vibration serviceability of a footbridge without the need of numerical analyses. The coefficient EAF is defined as the ratio between the maximum dynamic response to a realistic loading scenario and the maximum dynamic response to a single resonant pedestrian, while ESF is the ratio between the maximum dynamic response to a realistic loading scenario and the maximum dynamic response to uniformly-distributed resonant pedestrian loadings. Moreover in [22], the human walking is modelled using the random nature of the dynamic load due to pedestrians walk. The probabilistic approach allows the designer to account for the variation of the human walking force, due to the variation of the pedestrian weight, the step frequency, the step length and the dynamic load factors (DLF), and so forth..., on the dynamic response of the footbridges.

### **1.2 Recommendations and prescriptions**

Timber footbridges with long span, and more generally all slender bridges, are sensitive to vibrations, thus dynamic considerations have to be taken into account during the design [12].

These sources of vibrations can be due to pedestrian traffic, and in particular by a crowd of pedestrians walking along the deck in resonance with the bridge, or in extreme case by rescue vehicles (e.g. ambulance) that may cross the bridge in emergency cases [22]. The general design rules regarding vibrations in timber footbridges reported in Eurocode 5 ([23]), state that a bridge should be designed in a such a way that the loads acting on the bridge do not result in uncomfortable vibrations for the pedestrian.

The basic question is: "*how is possible to define a level of vibration comfortable or uncomfortable?*" As an example in Eurocode 0 ([24]) acceleration limits (Table 1.2.1) regarding pedestrian induced vibrations are proposed, but no methods are given for the evaluation of the dynamic behaviour. The code states that it is a designer's

responsibility to make reasonable assumptions during the design stage and analysis to guarantee that the proposed limits are fulfilled.

Acceleration limit [m/s2]	Direction and occurrence	
0.7	Vertical acceleration	
0.2	Lateral acceleration, normal use	
0.4	Lateral acceleration, exceptional crowd conditions	

Table 1.2.1. Acceleration limits in Eurocode 0 [24].

The experience of vibrations is highly individual because it is difficult to specify suitable regulations for vibrations in lightweight bridges (see [25] among others). In fact the perception of the vibration depends on visual and acoustic cues, individual's position (moving, standing, sitting or lying), preoccupation and knowledge, health conditions on that particular day, and so on. All these aspects can modify the occupant awareness about vibration perception increasing/decreasing the whole comfort level. Moreover the perception seems to be related to the human's activities, in particular as pointed out in [26] for tall buildings, "the least tolerant appears to be those people doing officework, with apartment dwellers seemingly more tolerant and tower-top diners even more".

The studies carried out in [27] and [28] show the peak of tolerable acceleration for people in function of the frequency. In particular these studies cover only the frequency range above to 1Hz (Figure 1.2.1).

In Figure 1.2.2 the tolerable peak of accelerations in function of low value of frequency (f<1Hz) a reported following the experiences reported in [29]-[31].

Similar conclusions are achieved in [32] and summarized in Figure 1.2.3.

In the particular case of wind-induced motion of tall buildings, the study reported in [33] suggested that the root-mean-square acceleration should not exceed 5 mg for a return period of 6 years to avoid significant vacancy of an office building.

The research carried out by [34] found that the difference between individuals' perception tends to be more significant than the effects of body positions, indicating the acceleration value of 4 mg for a perception level, and 2 mg as a disturbing level for the frequency of 0.13 Hz.

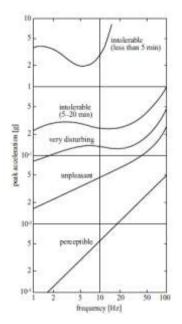


Figure 1.2.1. Peak of tolerable acceleration for high range of frequencies (f >> 1 Hz) ([27] and [28]).

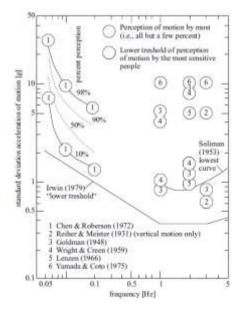


Figure 1.2.2. Peak of tolerable acceleration for low range of frequencies (f < 1 Hz). Values are plotted in terms of root-mean-square of the acceleration. To compare these result with the ones proposed in Figure 1.2.1, multiply these values by  $2^{0.5}$  ([29]-[31]).

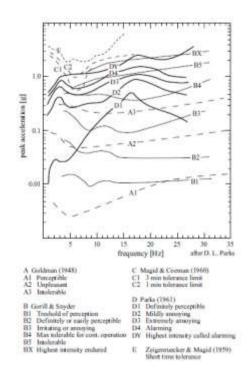


Figure 1.2.3. Summary of human exposure to acceleration. Peak accelerations versus frequencies [32].

Based on the experience matured by [35], the standard ISO6897 [36] proposes the following expression to indicate the level of human occupancy comfort in terms of the root-mean-square acceleration  $[m/s^2]$ :

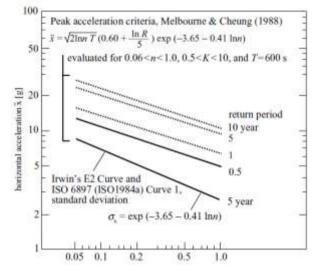
$$\sigma_a = e^{-3.56 - 0.41 \ln(n)}, \tag{1.1}$$

*n* being the frequency with an approximately normal distribution [Hz], as a satisfactory magnitude indicating a level of acceleration at which "*about 2% of the occupants will comment adversely*".

Further studies summarized in [26] develop a formulation to evaluate the peak acceleration in the horizontal plane  $a_h$  [m/s<sup>2</sup>] for the return period *R* [year] (less than 10 years), due to wind loads as:

$$a_h = \sqrt{2\ln(nT)} \left[ 0.68 + \frac{\sqrt{\ln(R)}}{5} \right] e^{-3.5 - 0.41\ln(n)},$$
 (1.2)

where *n* is the frequency [Hz] and *T* the period of wind storm (T=600 sec as suggested in [26]).



Equations (1.1) and (1.2) are graphically presented in Figure 1.2.4.

Figure 1.2.4. Peak acceleration criteria proposed by [35] and [26].

The above acceptable criteria regarding the human tolerable vibrations (comfort) may be classified as "physiological" criteria. More details about this topics, in particular related to how evaluate the comfort level of footbridges, will be presented and deeply analyzed in section 2.2.

More in general for all the structures the acceptance criteria for vibrations of civil engineering structures should be considered from various aspects. Indeed the structure should, or better must not collapse maintain its structural integrity during the "vibration". Secondly the vibration could produce any mechanical problems such as overstressing, malfunctioning and so forth..., that sometimes could be produce future troubles (e.g. fatigue problems) or worst hide problems that compromise the global structural safety. As for the comfort criteria is not so easy produce an unified code (see [37]-[39] among others), anyway different guidelines and/or code are proposed in literature. An example as given in Table 1.2.2 [40].

$A$ contained limit $[m/a^2]$	Peak velocity limits		
Acceleration limit [m/s <sup>2</sup> ]	Frequency range [Hz]	Velocity [mm/s]	
Industrial buildings	$f \leq 10\ 20$	20	
Residential buildings	$10 < f \le 50 \ 15 + f/2$	15 + f/2	
Vulnerable buildings	$50 < f \le 100 \ 30 + f/5$	30 + f/5	

Table 1.2.2. German Standard DIN 4150 [40].

Another chart showing the safety limits of structural vibration, and based on the study by [41], is presented in Figure 1.2.5.

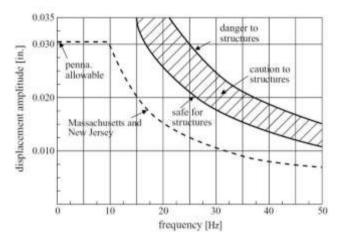


Figure 1.2.5. Structural safety criteria for frequencies range up to 50 Hz [41].

A general review of the existing criteria is proposed in [42] and summarized in Table 1.2.3.

An unified peak acceleration criteria taking account all the above presented study, is introduced by [43] and summarized<sup>2</sup> in Figure 1.2.6. This discomfort criterion approximately corresponds to the curve labelled as "unpleasant" in Figure 1.2.1, the

<sup>&</sup>lt;sup>2</sup> As specified by the author the proposed criteria is only a rough approach in order to evaluate the discomfort of the structure, and cannot be applied blindly to all structures.

peak acceleration criteria for R = 5 years in Figure 1.2.4 and about the middle line of the "Chen and Robertson bounds" (Figure 1.2.2) for the lower frequency range.

Structures	Acceptance acceleration/velocity level	Comments
Pedestrian structures	$a \le 5 - 10\%$ g	Normally the lower value does not produce discomfort
Office buildings	$a \le 2\%$ g	DIN and BS may yield quite different values
Gymnasia (sport halls)	<i>a</i> ≤ 5 − 10% g	Gymnasia $a \le 5 - 10\%$ g The higher value recommended only if the acoustic effect is small, and only participants are on or near the vibrating floor
Dancing and concert halls	$a \le 5 - 10\%$ g	The same as for gymnasia
Factory floors	$v \le 10$ mm/s	Stricter bounds required for high-quality production factories

Table 1.2.3. Structural criteria: overall acceptance.

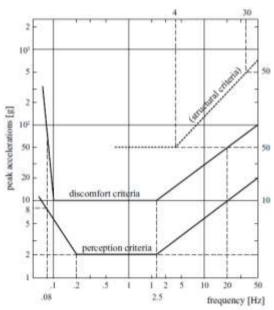


Figure 1.2.6. General peak acceleration criteria [43].

Concluding, usually the discomfort criteria are much stricter than the structural criteria and hence become more critical for structural design purposes. Thus, great care should be taken when the required services of the structure are particularly sensitive.

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# Chapter 2 Design requirements and performance satisfaction

As already observed in the previous chapter, one of the main problems when one works on footbridges, is not the safety aspect, but rather the limitation of the vibration in order to achieve a good result in term of comfort for the user.

This means that footbridges should be no longer designed for static loads only, but also accounting for the dynamic behaviour during the design stage, without forgetting the rule of thumb: "*the more slender structures become, the more attention must be paid to vibration phenomena*".

One of the main and simplest way if one wants to take into control the vibration of a given footbridge is that one has to keep the natural frequency within a suitable range as will be better analyzed and discussed in section 2.2 (see [1]-[2] among others). To fulfil such a requirements, one could be forced to a special footbridge design; in fact very slender and/or lightweight structures, suspension bridges and so forth... could be unable to satisfy these requirements. Not only natural frequencies but also mass, damping, deck loading (pedestrian loads) determines the dynamic behaviour of the footbridge.

These bridges are usually characterized by the lowest value of the natural frequencies very close to the external excitations like that of the pedestrian loads; thus the risk of resonance is quite great. Vibrations of footbridges may occur in vertical and horizontal directions. Even the torsion of the bridge deck is sometimes excited.

Vibrations of footbridges may lead to serviceability problems, affecting the comfort of the crossing pedestrians. Collapse or even damage due to human induced dynamic forces are quite unlikely to occur. For these kinds of structures, the pedestrian induced load<sup>3</sup> is probably the main source of vibration of footbridges, together with the wind loads.

<sup>&</sup>lt;sup>3</sup> Pedestrian loading is by nature unsteady, transient and waddling in a small range of excitation frequency.

Another dynamic loading on footbridges is intentional excitation by people that are jumping on the spot, bouncing, swaying body horizontally, shaking stay cables etc... In that case, the comfort is certainly not fulfilled but the structure should preserves its integrity under these extreme loads must be considered.

Summarizing, in general the assessment of human-induced vibrations needs to be considered by the designer during the design to evaluate that:

- 1) vibrations due to pedestrian traffic are acceptable for the users (comfort aspects),
- the footbridge does not collapse when subjected to intentional excitation (safety aspects due to extreme loads).

Once proved that the dynamic behaviour of the bridge due to expected loads is evaluated and fulfilled the required comfort, the footbridge can be designed and constructed. If the vibration behaviour does not satisfy some comfort criteria, changes in the design or damping devices should be considered.

Concluding, a well done footbridge design should consider all these factors in order to match together structural (both from static and dynamic point of view) and architectural aspects with the comfort requirements.

## 2.1 Dynamics of the footbridges design and structural health monitoring

The purpose of this section is to recall the main features and concepts within the dynamic of the structures (see [3]-[5] among others) and the structural health monitoring (see [6]-[8] among others).

#### 2.1.1 Governing relations

Fort the general formulation of the dynamic problem the reader will be referred to [9]-[14] among others.

Most of the structural dynamic properties can be highlighted studying the so called simple oscillator. The simple oscillator (discrete system) consists of mass m, connected to a support by a linear spring of stiffness k and a linear damper of viscosity c, on which is acting an external force F(t). It is supposed that the device moves only in one direction; therefore one has only one degree of freedom (1-DOF) defined by the position x(t) of its mass (Figure 2.1.1).

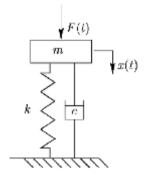


Figure 2.1.1. Schematic representation of a 1-DOF system.

The dynamic equation of equilibrium can be written as [9]:

$$m\ddot{x}(t) + c\dot{x}(t) + kx(t) = F(t), \qquad (2.1)$$

or, introducing the following quantities:

$$\omega_0 = \sqrt{\frac{k}{m}} = 2\pi f_0$$

$$\xi = \frac{c}{2\sqrt{km}}$$
(2.2)

in the new form:

$$\ddot{x} + 2\xi\omega_0\dot{x} + \omega_0^2 x = \frac{F}{m}.$$
 (2.3)

where the dependence on the time is made implicit for simplicity,  $f_0$  is the natural frequency of the oscillator  $(Hz)^4$  and  $\xi$  the (non-dimensional) critical damping ratio. It is important to underline that the following developments are based on the hypothesis of linear damping:  $F_{damping}(x, \dot{x}) \propto \dot{x}(t)$  (viscous, with a damping force proportional to speed [14]). This is the assumption selected by most footbridge designers and engineers. Assume that the external excitation can be represented as a harmonic function expressed as:

$$F(t) = F_s \sin(\omega t). \tag{2.4}$$

<sup>&</sup>lt;sup>4</sup> Or equivalently  $\omega_{\theta}$  the natural pulsation of the oscillator (rad/sec).

Then it is possible to evaluate the "static" response of the oscillator under the action of an equivalent force of amplitude equal to  $F_s$ ; that is:

$$x_{static} = \frac{F_0}{k} = \frac{F_0}{\omega_0^2}.$$
 (2.5)

From this it is possible to determine the maximum dynamic response of the system by simply multiplying the static response by the dynamic amplification parameter  $A(\Omega)$ :

$$x_{max} = x_{static} A(\Omega), \qquad (2.6)$$

where  $\Omega = \omega/\omega_0$  is the relative pulsation (or reduced pulsation), and the dynamic amplification parameter can be represented as a function of the relative pulsation and the critical damping ratio:

$$A(\Omega) = \frac{1}{\sqrt{(1 - \Omega^2)^2 + 4\xi^2 \Omega^2}}.$$
 (2.7)

These parameter may be represented by a set of curves parameterized by  $\xi$  as reported in Figure 2.1.2.

These curves show a peak for the value of  $\Omega_R = \sqrt{1 - 2\zeta^2}$  characterising the resonance<sup>5</sup> and therefore corresponding to the resonance frequency:

$$f_R = \omega_R / 2\pi \Leftrightarrow \omega_R = \omega_0 \sqrt{1 - 2\xi^2}.$$
 (2.8)

It is worth underlining that the resonance phenomena for a damped structure occurs when the frequency of the external excitation equals the resonance frequency; that is  $\omega = \omega_R$ .

Since the structural damping for the usual civil structures is weak ([9] and [13]), i.e.  $\xi \ll 1$ , one may consider that resonance occurs when:

$$\omega = \omega_0 \approx \omega_R \Leftrightarrow \Omega_R = 1 \to A \left( \Omega_R = 1 \right) \approx \frac{1}{2\xi}.$$
 (2.9)

<sup>&</sup>lt;sup>5</sup> Resonance is the tendency of a system to oscillate with greater amplitude at some frequencies than at others. At these frequencies, even small periodic driving forces can produce large amplitude oscillations.

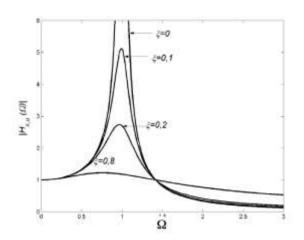


Figure 2.1.2. Dynamic parameter trend for specific value of the critical damping.

The above properties, even if introduced for a *1*-DOF, show that designing the structures only taking into account the maximum intensity ( $F_s$ ) of the acting load can result to be unsatisfactory. In fact the resonance phenomenon can generate higher displacements/ stresses even if the intensity of the loads is lower. Thus, for example, the force  $F_1(t) = F_s \sin(\omega_1 t)$  can generate displacements or stresses much lower than a load with an amplitude 10 times weaker,  $F_2(t) = [F_s/10] \sin(\omega_2 t)$ , only because  $F_2(t)$  has a frequency much closer to the resonance frequency of the structure. Moreover the resonance amplification is directly related to damping.

The most common sources of damping are:

- 1) the internal damping linked with the material itself; its value is mainly linked to the temperature and the frequency of excitation;
- 2) the damping by friction (Coulomb)  $F_{damping}(x, \dot{x}) = \mu[sign(\dot{x})]$ : it is mainly linked to the construction joints between different elements.

Just to illustrate how the damping acts on the response of the oscillator, consider the so called free vibration: assume that the external force is equal to zero over the time (F(t)=0). Thus, re-writing equation (2.3) one achieves:

$$\ddot{x} + 2\xi \omega_0 \dot{x} + \omega_0^2 x = 0. \tag{2.10}$$

The generic solution of the above differential equation can be expressed as:

$$x(t) = X_0 e^{(rt)}, (2.11)$$

being *r* the solution of the characteristic equation:

$$r^{2} + (2\xi\omega_{0})r + \omega_{0}^{2} = 0.$$
(2.12)

 $\xi \ge 0$  characterized the so called dissipative system. The different rates are so classified depending on the damping ratio as:

- 1)  $0 \le \xi < 1$  (the typical case for civil structures);
- 2)  $\xi = 1;$
- 3)  $\xi > 1$ .

The response of the 1-DOF system is summarized in Figure 2.1.3.

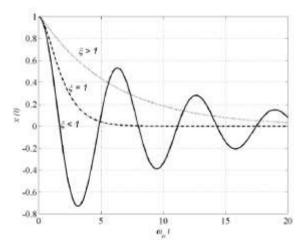


Figure 2.1.3. Free oscillation of a 1-DOF system with different value of damping.

For the applications of this thesis damping is modelled by viscous damping. This may be phisically interpreted as a force that opposes the velocity of the structure. It is interesting to note that sometimes (as for some models of the pedestrian load), the forces "agree" with the velocity. This "agreement" generates "negative" value of damping, which leads to oscillations of the pedestrian bridges that become greater and greater (Figure 2.1.4). Thus, instability problems could appear and could lead, in the worst hypothesis, to the collapse of the construction.

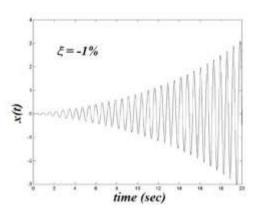


Figure 2.1.4. Oscillation of a 1-DOF system – divergences induced by a "negative" damping.

A properly evaluation of the damping is therefore a key point ([15] and [16]).

Two methods are here briefly presented ([9], [14] and [16]):

1) *logarithmic decrement method*: the damping may be evaluated knowing the period  $T_a$  corresponding the interval between two distinct peaks, and the

decrement 
$$\delta = \log\left(\frac{x(t_1)}{x(t_2)}\right) \approx \frac{2\pi\xi}{\sqrt{1-\xi^2}}$$
. Assuming  $\xi \ll 1$ , one obtains:  $\xi \approx \frac{\delta}{2\pi}$  (Figure 2.1.5).

2) bandwidth method: the damping may be evaluated by the equation  $\Delta \Omega = \Omega_1 - \Omega_2 \approx 2\xi \sqrt{1 - \xi^2} \approx 2\xi \text{ when } \xi \ll 1, \text{ where the bandwidth } \Delta \Omega \text{ is}$ such that  $|A(\Omega_{1,2})| = \frac{|A(\Omega)|_{\text{max}}}{\sqrt{2}}$  (Figure 2.1.6).

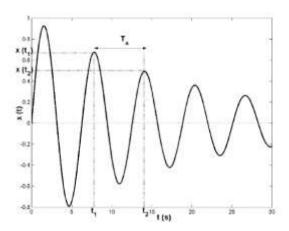


Figure 2.1.5. Representation of the "logarithmic decrement method".

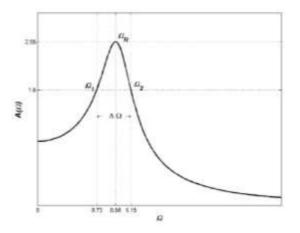


Figure 2.1.6. Representation of the "bandwidth method".

In order to generalize the problem to the case of a discrete n-DOF system, system in Figure 2.1.7 is considered. It sees n masses connected in series by springs and dampers. The displacement of the generic mass is then dependent on the displacement of another mass; in other word one can say that the n masses are coupled through springs and dampers.

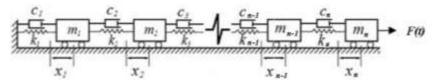


Figure 2.1.7. Schematic representation of a *n*-DOF system.

Let's  $x_i(t)$  the position of the *n*-th mass where i=1,2,...,n. Then, the dynamic equation of equilibrium becomes [9]:

$$\mathbf{M}\ddot{X} + \mathbf{C}\dot{X} + \mathbf{K}X = F,\tag{2.13}$$

where:

•  $X = \begin{bmatrix} X_1 & X_2 & \cdots & X_n \end{bmatrix}^{-1}$  is the vector of DOFs (size *n*); •  $\mathbf{M} = \begin{bmatrix} m_1 & \cdots & 0 \\ \vdots & \ddots & \vdots \\ 0 & \cdots & m_n \end{bmatrix}$  is the mass matrix (size *n* by *n*). Dealing with the

hyphotesis of lumped masses (discrete approach) this matrix assumes the classical diagonal form because the accelerations of a lumped mass produces inertial forces only in correspondance of the concentration points;

• 
$$\mathbf{C} = \begin{bmatrix} c_{11} & \cdots & c_{1n} \\ \vdots & \ddots & \vdots \\ c_{n1} & \cdots & c_{nn} \end{bmatrix}$$
 is the damping matrix (size *n* by *n*);

• 
$$\mathbf{K} = \begin{bmatrix} k_{11} & \cdots & k_{1n} \\ \vdots & \ddots & \vdots \\ k_{n1} & \cdots & k_{nn} \end{bmatrix}$$
 is the stiffness matrix (size *n* by *n*);

•  $F = [f_1 \ f_2 \ \cdots \ f_n]^{-1}$  is the vector of the external forces acting on the system (size *n*).

in which the dependence on the time is neglected for simplicity.

The mass and stiffness matrices can be evaluated starting from energy considerations [9] as follows:

• the mass matrix is associated with the kinetic energy *T* of the system, as the sum of the kinetic energies of each mass; that is:

$$T = \frac{1}{2} \sum_{i=1}^{n} \left( m_i \dot{x}_1^2 \right) = \frac{1}{2} \dot{X}^T \mathbf{M} \dot{X}; \qquad (2.14)$$

• the stiffness matrix is associated with the strain energy *J* of the system, that is the sum of the elastic potential energy of each spring:

$$J = \frac{1}{2} \left[ k_{11} x_1^2 + k_{12} \left( x_2 - x_1 \right)^2 + \ldots + k_{nn} x_{n-1}^2 \right] = \frac{1}{2} X^T \mathbf{K} X; \qquad (2.15)$$

These two matrices are both symmetric, moreover the mass matrix is positive definite (because the kinetic energy is always positive by definition), while the stiffness matrix is positive semi-definite because of definition.

For *n*-DOF systems it is useful to introduce the *modal superposition* approach [14]. Consider the free movement case, thus the equation of motion (2.13) can be re-written as:

$$\mathbf{M}\ddot{X} + \mathbf{K}X = 0, \tag{2.16}$$

where the generic vector solution may be represented as:

$$X = \phi_0 \,\mathrm{e}^{rt},\tag{2.17}$$

in which  $\phi_0$  is a vector of constant to be determinate, of size *n*. Substituting (2.17) into (2.16) one reach:

$$\left(\mathbf{K} + r^2 \mathbf{M}\right) \phi_0 = 0. \tag{2.18}$$

The above equation admits advantageous solutions only if the determinant of the system is equal to zero; that is only if:

$$\det\left(\mathbf{K}+r^{2}\mathbf{M}\right)=0.$$
(2.19)

For a classical *n*-DOF system the above system has 2*n* solutions:  $\pm i\omega_k$ , k = 1, 2, ..., n where  $\omega_k$  represents the *k*-th natural pulsation of the system. With each index *k*, there is an associated vector called modal vectors of the system  $\phi_k$  such that the response of the whole system can be re-written as:

$$X(t) = \sum_{k=1}^{n} \phi_k \left( \alpha_k \, \mathrm{e}^{-i\omega_k t} + \beta_k \, \mathrm{e}^{+i\omega_k t} \right). \tag{2.20}$$

The components of the vector  $\phi_k$  are linked, thus usually these modal vectors are defined after normalization. The most frequent choices are here summarized:

- one of the components is made equal to 1.
- each mode is unitary; i.e.  $|\phi_i| = 1$ ;

• satisfy the requirement:  $\phi_i^T \mathbf{M} \phi_i = 1$ .

Summarizing, the natural pulsations  $\omega_i$  are the square roots of the natural positive values of the matrix  $\mathbf{M}^{-1}K$  evaluable solving the equation  $\det(\mathbf{K} - \omega_i^2 \mathbf{M}) = 0$ . The natural forms of a discrete system are the natural vectors  $\phi_i$  associated with it. The couple  $(\omega_i, \phi_i)$  is called a natural vibration mode where i=1,2,...,n. In other words for a linear system a mode is one particular solution of the free non-damped vibration, in which all the DOFs components are synchronous<sup>6</sup>:

$$X(t)_{\text{mode 0}} = \phi_0 e^{i\omega_0 t}$$
 (2.21)

The above natural vibration modes are orthogonal in relation to the mass matrix and in relation to the stiffness matrix [9]; that is:

$$\begin{aligned} \phi_i^T \mathbf{M} \phi_j &= m_i \delta_{ij} \\ \phi_i^T \mathbf{K} \phi_j &= k_i \delta_{ij} \end{aligned}, \quad \forall i, j$$

$$(2.22)$$

in which  $\delta_{ij}$  is the classical Kronecker symbol, while  $m_i$  and  $k_i$  represent, respectively, the so called generalised mass and stiffness associated to *i*-th mode. Their values will depend on the normalization used to define the natural vectors as described above.

To the case of non-damped forced vibration, the dynamic equilibium (2.13) become as:

$$\mathbf{M}\mathbf{X} + \mathbf{K}\mathbf{X} = F. \tag{2.23}$$

To solve directly the above equation needs to evaluate the inverse of the mass matrix that could be costly, from the calculation point of view, if the number of DOFs is high. Thus, it is standard practice to apply the modal superposition approach. This way is quite useful because it is able to unlinks the equations of motion that compose the general equations, significantly reducing the calculation operations. The principle behind this approach is to introduce a reference change re-writing the equation of motion in the new base system obtained by a linear combination of the natural vectors  $\phi_i$  the so called modal base. Thus, the response X(t) of the system represented by equations (2.23) may be expressed as:

$$X(t) = \mathbf{\phi}q(t) = \sum_{i=1}^{n} \phi_i q_i(t), \qquad (2.24)$$

<sup>&</sup>lt;sup>6</sup> That is they reach their maxima and their minima at the same time.

in which:

- $\phi = [\phi_1 \quad \phi_2 \quad \cdots \quad \phi_n]$  is a matrix (size *n* by *n*) where each columns contain the natural vector  $\phi_i$ ;
- q(t) is the vector the modal variables, the new variables obtained re-writing the problem in the new reference system.

Having in mind the orthogonality of the modes, see (2.22), and the generic solution in (2.24), pre-multiplying equation (2.23) by  $\phi^{T}$  one obtains:

$$\boldsymbol{\phi}^{T}\mathbf{M}\boldsymbol{\phi}\ddot{q}(t) + \boldsymbol{\phi}^{T}\mathbf{K}\boldsymbol{\phi}q(t) = \boldsymbol{\phi}^{T}F(t) \rightarrow \tilde{\mathbf{M}}\ddot{q}(t) + \tilde{\mathbf{K}}q(t) = \boldsymbol{\phi}^{T}F(t)$$
(2.25)

or in extended form:

$$\begin{bmatrix} \ddots & 0 & 0 \\ 0 & m_i & 0 \\ 0 & 0 & \ddots \end{bmatrix} \ddot{q}(t) + \begin{bmatrix} \ddots & 0 & 0 \\ 0 & k_i & 0 \\ 0 & 0 & \ddots \end{bmatrix} q(t) = \mathbf{\phi}^T F(t).$$
(2.26)

where  $\tilde{\mathbf{M}}, \tilde{\mathbf{K}}$  are respectively the diagonalized mass and stiffness matrices. This allows one to get a system of unlinked differential equations as:

$$\ddot{q}(t) + \omega_i^2 q(t) = p_i(t), \qquad (2.27)$$

where the vector  $p_i(t) \triangleq \frac{\phi^T F}{m_i}$  contain the component per unit of generalized mass of

the force vector F(t), and it is called the modal contribution factors of the applied force vector.

The possibility of writing the general problem expressed by equation (2.23) in diagonalized form should be achieved in order to decrease the computational cost as observed. The presence of damping matrix **C** makes the problem a little bit more complicated. In fact, as far as the damping matrix **C** is a general matrix, it is not possible to state that this matrix will also be diagonalized by the modal superposition approach. However, there are some cases where this property does apply. The most remarkable case when a matrix **C** is transformed into a diagonal matrix by the modal approach is the one called "proportional damping". In fact when the structure under study is not to much dissipative and the natural frequencies are clearly separated, it can be shown that is reasonable ([9] and [14]) to define the damping matrix as a linear combination of the mass and stiffness matrices; that is:

$$\mathbf{C} = \sum_{j=1}^{N} \alpha_{j} \mathbf{M} \left( \mathbf{M}^{-1} \mathbf{K} \right)^{j-1}, \quad \alpha_{j} \in \mathbb{R}.$$
(2.28)

Assuming in the above equation *N*=2 one achieves:

$$\mathbf{C} = \alpha_1 \mathbf{M} + \alpha_2 \mathbf{K}, \tag{2.29}$$

the so called Rayleigh's damping matrix, where  $\alpha_1$  and  $\alpha_2$  are two constant parameters called Rayleigh's coefficients, whose values define the entity of damping actions in the system, that in the International System have the physical dimension of  $s^{-1}$  and s respectively.

Since both the mass and stiffness matrices may be diagonalized applying the modal approach (see equation (2.23)), is simple to observe as the damping matrix expressed by will also be diagonalized by following the same approach; indeed:

$$\boldsymbol{\phi}^{T} \mathbf{C} \boldsymbol{\phi} = \boldsymbol{\phi}^{T} \left( \alpha_{1} \mathbf{M} + \alpha_{2} \mathbf{K} \right) \boldsymbol{\phi} = \dots = \alpha_{1} \mathbf{\tilde{M}} + \alpha_{2} \mathbf{\tilde{K}}.$$
(2.30)

Thus, the *n* equations of motion of the generic system described in (2.23) are decoupled. Then one considers each single equation as represented by a *1*-DOF system as follows<sup>7</sup>:

$$\ddot{q}(t) + 2\xi_i \omega_i \dot{q}(t) + \omega_i^2 q(t) = p_i(t), \qquad (2.31)$$

where  $\xi_i = \frac{\gamma_i}{2\omega_i m_i}$  in which  $\gamma_i = \alpha_1 m_i + \alpha_2 k_i$ . In practice, each critical damping ratio  $\xi_i$  must be experimentally recalibrated and/or evaluated. By the way it is usual to consider that the value of the critical damping ratio does not depend on the mode under consideration. This allows one to assume  $\xi_i = \xi$ , where the constant value  $\xi$  is fixed

experimentally or by the regulations [36].

In conclusion the principle of modal superposition allows one to treat the generic n-DOF system as the "sum" of n simple oscillators (Figure 2.1.8), once a new reference frame has been introduced.

<sup>&</sup>lt;sup>7</sup> This formulation can be seen as a generalization of equation (2.27).

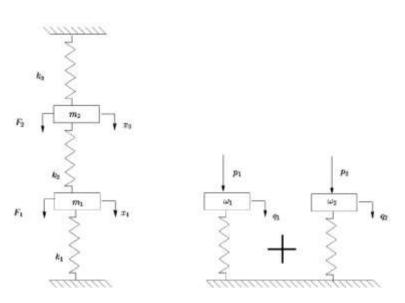


Figure 2.1.8. Modal superposition approach applied to a 2-DOF system.

#### 2.1.2 Structural Health Monitoring

Another important aspect for civil engineering structures (e.g. bridges, strategic buildings and so forth... but in particular footbridges), is the maintenance, and/or the health monitoring issue. It is important indeed to maintain at least, and when possible improve, the quality and the level of serviceability of the structural system for the user. To achieve this purpose one needs to "know" the lifecycle performance of the structure to ensure long service life and durability. A useful tool to be used for this scope is the so called **Structural Health Monitoring** (SHM). The SHM process involves the observation of a system over time using sampled dynamic response measurements from several sensors installed along sensitive points of the structure. From the analysis and study of these measurements, or in other words from the study of the on field dynamic behaviour of the structure, the health of the system can evaluated, pointing out any eventual damage or problem for the safety.

As proposed by [17] five axioms may be formulate for structural health monitoring that are:

• Axiom 1. The assessment of damage requires a comparison between two system states.

- Axiom 2. The identification of the existence and location of damage can be done in an unsupervised learning mode, but the type of damage and the damage severity can only be identified in a supervised learning mode.
- Axiom 3. Without intelligent feature extraction, the more sensitive a measurement is to damage, the more sensitive it is to changing operational and environmental conditions.
- Axiom 4. There is a trade-off between the sensitivity to damage of an algorithm and its noise rejection capability.
- Axiom 5. The size of damage that can be detected from changes in system dynamics is inversely proportional to the frequency range of excitation.

In the civil engineering area SHM requires to take account for numerous non linearity and not well known "boundary" condition of the problem, as in the case of soil structure interaction. Thus, the SHM key of success is enclosed in the possibility to collect high quality data combined with realistic identified models and deterioration laws [12]. The final output of a SHM could be the reliability index, the safety level, a graphic symbol or any other output depending on the owner [18]. An example of a SHM architecture applied to a bridge case is reported in Figure 2.1.9. Another example can be found in [19].

In the last years different approaches were proposed for health monitoring (see [20]-[24] among others). It is important that the data collected during the experimental campaign by accurately stored in order to, first of all, create a database for the structure, and apply new future algorithms when available to compare the structural behaviour between the "old" and "new" data analysis approach. By the way, it is important to note that to achieve a whole computerized SHM system able to make decision about the structural health monitoring by itself is far away.

It is clear now as SHM provides the opportunity to quantify the condition of the structure and gives the guidelines for making decisions about the structure (Figure 2.1.10), since the dynamic characteristic contains information on the global structural condition as well as on local phenomena [13].

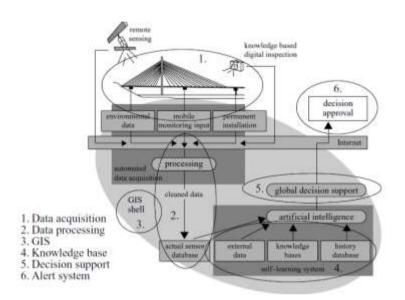


Figure 2.1.9. SHM architecture applied to a bridge [12].

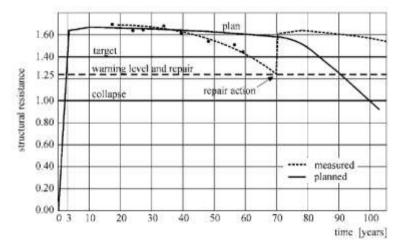


Figure 2.1.10. Example of proposed rehabilitation planning based on the Ultimate Limit Strength [13].

Three different mains topics could be taken into account when one wants to perform a SHM:

- 1) responsibility;
- 2) economy;
- 3) curiosity.

From the above-mentioned motivations it is necessary to offer services on increasing quality levels depending on the structure under study and so to the depths of investigation that one wants to perform. Usually five level of investigation (Figure 2.1.11) are introduced within bridges field according to [12]:

- Level 1 Rating: this represents the basic/conventional assessment of the structure starting with a visual field inspection that provides a subjective impression of the condition of the structure.
- Level 2 Condition assessment: rough visual field inspection after which is possible to decide whether the conventional approach is satisfactory or an extended one is needed. Moreover this inspection will determine the type and quantity of instrumentation that will be used on field.
- Level 3 Performance assessment: provides additional indicators for the assessment and will demonstrate the performance of the structure. It requires several synchronous devices to perform the monitoring.
- Level 4 Detail assessment and rating: establish an analytical model representing the structure. The numerical model will be compared with the result collected during the monitoring campaign(s).
- Level 5 Lifetime prediction: the last step regards the lifetime evaluation (see Figure 2.1.12 as proposed in [25]). To achieve a serious lifetime prediction the records available have to be long enough to cover, at least, three relevant cycles for the structure (e.g. usually at least three years).

To achieve the above purpose, a selection of suitable data has to be made. In particular one can have four different levels of monitoring,, that are:

- spot monitoring: should comprise a very quick measurement campaign with a few simple to handle sensors only. It provides only general condition information of the structure in order to create a ranking;
- 2) *periodic monitoring*: repeated campaigns on a structure after a specified time interval (e.g. seasonal, monthly campaign and so forth...);
- 3) *permanent monitoring*: continuously monitoring over the time. This approach allows a very detailed assessment about the structure;

 online monitoring: observation and assessment allows warning through electronic media<sup>8</sup>. Decisions might be taken by the computer based on the measurement data.

With reference to above classifications the monitoring carried out in this work, and related to the two footbridges object of study, can be defined as **periodic monitoring** – **level 3**.

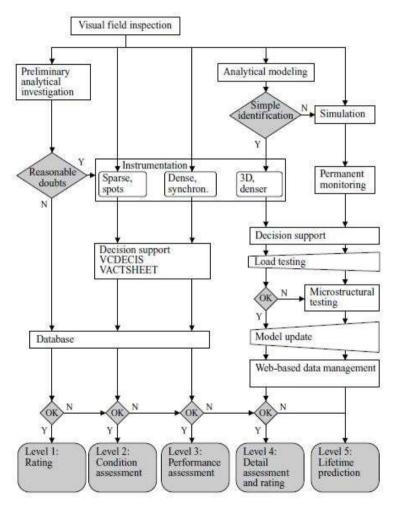


Figure 2.1.11. SHM for bridge. Typical flowchart [16].

<sup>&</sup>lt;sup>8</sup> These alert systems will only be applied at extremely critical structures.

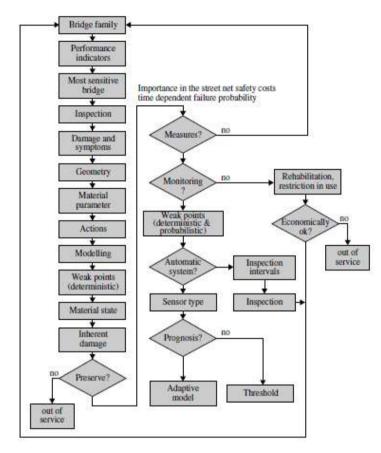


Figure 2.1.12. SHM for bridge. Typical flowchart [25].

Different typologies of devices/sensors may be used to acquire data from field as summarised in Table 2.1.1 [12], while usually the transmission of the data is entrusted to two technologies: either wired or wireless solution [26].

In conclusion, the monitoring system [27] should be considered and implemented by the designer at the design stage. Indeed the designer knows very well which could be the sensitive cross-sections of the structure, so he/she can suggest the best position for the sensors along the structure in order to optimize the quality cost trade-off. When the SHM is implemented within the design stage, all the infrastructures (e.g. power supply devices, cable, and so on) necessary to perform the monitoring may be located in an optimal position reducing once again the global operational cost.

Measured Value	Sensor Type	Mourarement Range (MR)	Resolution	Linearty	Supply	Frognorey Range	Influences
Displacement	LVDT	1-50 mm	1 % MR	0.1% MR	AD/DC	0-100 Hz	stress ferces
	LVDT caliper Triangulation sensor Cable extension transcheer	1-50 mm 2-200 mm 50-40000 mm	1 % MR 1 % MR	0.1% MR 0.3% MR 0.05% MR	ACDC DC DC	0-50 Hz 0-10 kHz 0-500 Hz	measurement frequency reflecting surfaces, soil cross fires
ndimisen	Bubble level Pendubari	±10 grade ±1 grade	1 % MR 1 % MR	0.1 grade 0.05% MR	DC DC	0.5Hz	vibrations
Settlement	Hydrostatic leveling system PSD	0-00 mm 0-00 mm	0.01 mm	< 1% MR	2 2	static 0-500 Hz	atmosph press., temp gradients meisture
Strain	Strain ganges Fiber Bragg graftugs Fabry-Perot fiber sensor SOFO system Optical string	1-10000 µmbm + 10000 µmbm + 5000 µmbm 0.5% sensor length 0.5% sensor length	1 juntin 1 juntin 1 juntin 2 juntin	<115 < 176	DC/AC Laser light Broathand white light Laser light Laser light	0-100 kHz 0-100 kHz 0-1 kHz static 0-100 Hz	AT, headbage AT, transverse stress AT fruttsverse stress
Acceleration	Prezedoctine aerosoria MEMS-A6-40 B12 (diffarential choke)	+100g +1g +20g	10 µg 5 µg	2 <u>2</u> 	2 2 2	0.1-2000 Hz 0-250 Hz 0-100 Hz	fixing
Whenting velocity	Geophone Laser vibrometer	100 mm/s 10 m/s	5 µm/s 1 µm/s	~14 ~15	DC/AC	4-1000 Hz 1-20000 Hz	fixing
Temperature	Thermocouples Pr100	-185 to 300-C -200 to 600-C	50 µ V/K 400 µ V/K	121	Я		leads. tempenature gradients

Table 2.1.1. Typical sensors employed during a SHM [12].

#### 2.2 Comfort for footbridges: codes and literature overview

The classical design of the pedestrian bridges allows to work within a static context. It is therefore quite frequent, that after the construction of the bridge, the crossing pedestrians feel vibrations. In other words these structures tend to be more sensitive to dynamic forces induced by pedestrians, resulting in vibrations of the bridge deck. These vibrations can reach high levels, especially when the walking step of the pedestrians is close to the natural frequency of the bridge, resulting in situations in which the users feel uncomfortable or even unsafe. This aspect is very important for the Serviceability Limit State (SLS) evaluation of footbridges. As above suggested, the classical static approach is not enough to guarantee an adequate degree of comfort, and a more sophisticated dynamical approach seems to be more effective during the design stage.

Within the contest of the comfort evaluation for the footbridges (and in general for all those structures sensitive to vibrations problems), different approaches or guidelines are proposed in the last years (see [28]-[30] among others). But an unified code that establishes "how" to evaluate these aspects has not yet been coded, and these topics are still in a developing stage. This lack of regulation gives to the designers a sort of freedom during the design. Nevertheless it is of vital importance that the bridge has to get a minimum level of comfort in agreement at least with the client or owner requirements. Thus the question "Will the footbridge meet the comfort criteria when vibrating?" plays an important role in the design process.

As observed in section 1.2 the concept of comfort is directly correlated to human perception and so it is not so simple to identify a general and unified definition of "comfort level".

In the following the more interesting guidelines and codes which could be taken into account during the design stage, are presented and discussed. More details are reported in *Appendix C*.

The first code to be proposed is the **Eurocode** [31]. The criteria on which this code is based, is to provide the maximum level of acceleration to ensure the minimum level of comfort. In all those cases in which this minimum level is no longer satisfied, the code prescribes the installation of *ad hoc* devices (e.g. dampers, TMD and so forth...) to reduce the level of vibration on the bridge. More in detail Eurocode 0 - Appendix 2 [31], states that the pedestrian comfort criteria must be defined in terms of maximum acceleration in any part of the bridge. The maximum values of acceleration recommended are:

• 0.7 m/s<sup>2</sup> for the vertical vibrations;

- 0.2 m/s<sup>2</sup> for horizontal vibration, in conditions of normal use;
- 0.4 m/s<sup>2</sup> for horizontal vibration, in exceptional conditions of the crowd.

In addition, the code states that the comfort assessment must be carried out if the natural frequencies of the bridge are less than:

- 5 Hz for vertical vibrations;
- 2.5 Hz for the horizontal transverse and torsional vibrations.

In Eurocode 1 - Part 2 [32] the traffic load models for the design of road bridges, rail and pedestrian walkways are defined. In particular in Section 5.7, the dynamic models to be applied for pedestrian loads are discussed. Moreover it is underlined as, depending on the dynamic characteristics of the structure, the main natural frequencies of the bridge should be evaluated by an appropriate numerical model. Furthermore in the SLS verification of vibration must be considered those forces exerted by pedestrians which have a frequency equal to a natural frequencies of the structure. Ad hoc model of these forces should be proposed by the designer, while the code does not give more details about "how" to evaluate the human induced loads. Eurocode 5 – Part 2 deals with the design of wooden structures [2], and in particular covers the design of timber bridges. This code requires the calculation of the maximum acceleration in the presence of small groups or crowds of persons both in the vertical and horizontal (i.e. in transversal direction compared with the deck axis) direction, where the maximum acceptable accelerations are the same reported in Eurocode 0 and above summarized. The code states once again that the assessment should be made for those structures that have natural frequencies lower than 5 Hz and 2.5 Hz, respectively, for the vertical and horizontal modes.

Another code is the **ISO 10137** [33]. This code provides a series of principles for the prediction of the vibrational characteristics of a structure during the design stage. To assess the effects of the vibrations is important to refer to the type of vibration source, its location and the receiving element. The assessment of the response is evaluated by a numerical model is needed. The code gives to the designer the choice of the comfort of level to be considered, but states that the pedestrian bridges must be so that the amplitude of the vibrations do not cause discomfort to users. In the absence of more definitive data, the code suggests the following maximum level of vibrations:

• in the vertical direction (z-axis): the vibrations should not exceed those obtained by a multiplying factor of 60 to the relevant base curve represented in Figure 2.2.1 when one or more persons still standing on the walkway have

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f

to be accounted for (such as the first scenario), in which case a multiplying factor of 30 should be applied.

horizontal vibrations induced by pedestrian traffic or wind should not exceed • 60 times the base curve for the horizontal direction (x and y- axis), see Figure 2.2.2.

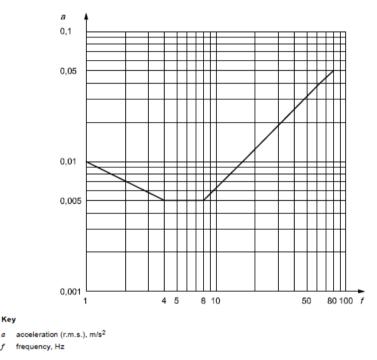


Figure 2.2.1. Building vibration z-axis base curve for acceleration (foot-to-head vibration direction) [33].

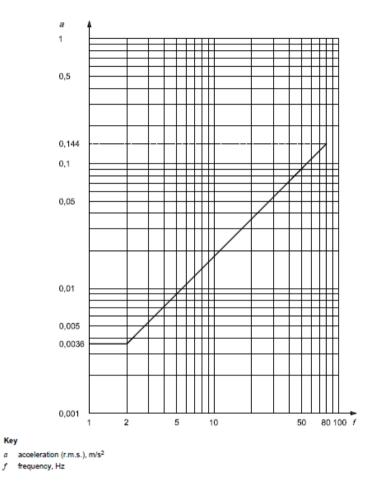


Figure 2.2.2. Building vibration z-axis base curve for acceleration (foot-to-head vibration direction) [33].

The **British Standard** BS5400, BD 29/04 and BD 37/01 [34] state that all the footbridges should satisfy the following minimum requirements:

- in the vertical direction the natural frequencies of the structure must have a value greater than 5 Hz;
- in the transverse horizontal direction the minimum frequency is fixed at 1.5 Hz.

If the fundamental frequency of vertical vibration, said  $f_0$ , however, is less than or equal to 5 Hz, the maximum vertical acceleration  $a_v$  (expressed as m/s<sup>2</sup>) of any part of the bridge has to be limited to:

$$a_v \le 0.5\sqrt{f_0}$$
 . (2.32)

The maximum vertical acceleration may be calculated either by simplified method or by rigorous method. For complex structures, the maximum vertical acceleration must be calculated assuming that the dynamic load induced by a single pedestrian is that due to a concetrated force *F*, which moves through the span of the bridge at a constant speed  $v_t = 0.9 f_0$  [m/s], where:

$$F = 180\sin(2\pi f_0 t). \tag{2.33}$$

If the fundamental frequency of the horizontal vibration is less than 1.5 Hz, great attention has to be paid to avoid the possibility to activate the lateral movement of the deck (the so called *lock-in* effects).The codes, however, do not propose any method to evaluate the maximum horizontal (transversal) acceleration.

The guidelines published by the **Fib** [35] provide an available tool for the design of the footbridges, in particular with reference to the assessment of the maximum acceleration to be compared with the comfort limit value. Following this guideline the maximum acceleration in  $[m/s^2]$  is defined as:

$$a = 0.6 \frac{\alpha G}{0.5M} \frac{\pi}{\delta} \left( 1 - e^{-n\delta} \right), \tag{2.34}$$

where G is the dead load of an ideal pedestrian (0.7 kN), M the mass in [t] of the equivalent *1*-DOF oscillator for single span,  $\delta$  the logarithmic decrement and n the number of cycles per span (span over step length).

The last document here analyzed is the **Technical guide Sétra-AFGC** [36]. Several aspects have to be considered by the designer during the design of the footbridge to avoid resonant phenomena in order to improve, or guarantee, an adequate level of comfort. Moreover this guideline can be used in order to evaluate the comfort of existing bridges. Operationally the proposed method can be summarized in the following five steps:

- 1) step 1: determination of the footbridge class;
- 2) *step 2*: choice of comfort level;
- 3) *step 3*: evaluation of the dynamic footbridge properties (if needed);

- 4) *step 4*: dynamic evaluation (if needed);
- 5) *step 5*: modification of the structural design to fulfil the comfort requirement (if needed).

The methodology of working is summarized in the flowchart in Figure 2.2.3.

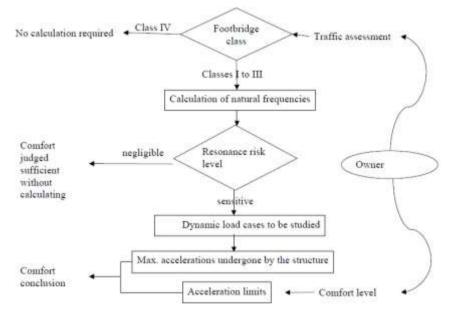


Figure 2.2.3. Methodology flowchart proposed by Sétra-AFGC [36].

### 2.3 Pedestrian Timber Bridge: a new construction way. Introduction to the structures under study

Pedestrian bridges are becoming a common trend in the design. In particular in the last years thanks to the developments of new [37] high strength materials within the field of wood (e.g. Glue Laminated Timber (GLT) elements), in many installations regarding the construction of footbridge, timber is selected as main resistant material.

Part strictly structural reasons based on the strength of wooden materials, timbers elements are valuable for both economics and environmental aspects because they are eco-friendly materials. For this reason they are very appreciated in all those installation where the bridge as to fit well with the surrounding environment.

In this work two footbridges located in two different Alpine villages are considered as case studies.

The first one, called "Trasaghis" footbridge (Figure 2.3.1), is located in the Trasaghis Municipality not far Udine city (North-east of Italy). Even if the bridge looks like the cable-stayed type, in this structural solution the cables are replaced by oblique steel tubular elements, i.e. stays, of external diameter 45.75 cm and thickness 1.42 cm. The replacing of the cables by the stays, allowed the designer to save money invested in the realization of an *ad hoc* protection system (larch plank and metal cladding) for the two main GLT beams. From geometric point of view the span is about 83 m long and 3.22 m wide; the pillar shown a height of about 15 m and the double-beam deck width is 4 m, of which 3.22 m represents the free crossing width [38]. The whole static scheme sees two pillars located on both sides (Figure 2.3.2), while the resistant scheme of the deck is obtained linking the two main glued laminated timber (GLT) arcuate beams, of rectangular cross-section of 20 by 194.1 cm, by H-shape steel tubular elements (Figure 2.3.3).



Figure 2.3.1. "Trasaghis" footbridge.



Figure 2.3.2. "Trasaghis" footbridge. View of the pillars.



Figure 2.3.3. View from the bottom of the "Trasaghis" footbridge on the left. Zoom on the structural resistant scheme of the deck on the right.

The second footbridge under study, called "Tesa" footbridge, is shown in Figure 2.3.4. It is located in Farra d'Alpago Municipality not far from Belluno city (Figure 2.3.5). The static scheme of the bridge is inspired by the classic cable-stayed solution [39]. Totally 16 steel cables (of diameter 44 mm and 32 mm) are employed to carries the two main curved GLT beams of rectangular cross-section 18 by 123.8 cm. The two antennas are 16 m high. From a geometric point of view, the span length is about 110 m subdivided into three segments of length 22.5 m, 65 m (in the middle) and 22.5 m, respectively; while the width of the deck is about 2.5 m. The resistant scheme of the deck is realize by linking these two main beams with steel tubular elements of high strength, assuming the classical "U-shape" cross-section (Figure 2.3.7).



Figure 2.3.4. General view of the "Tesa" footbridge.



Figure 2.3.5. "Tesa" footbridge. Satellite view of the location.



Figure 2.3.6. View of the pillar on the left; detail of the connection between cable and deck on the right.



Figure 2.3.7. View of the structural scheme of the "Tesa" footbridge deck.

Both the footbridges mentioned above were experimentally and numerically studied in this thesis. In particular:

- "Trasaghis" footbridge: has been subject to an extensive in situ monitoring campaign carried out by the author with the aim to identify its dynamic behaviour, in order to implement, subsequently, different numerical structural control systems as presented in *Chapter 6*;
- "Tesa" footbridge: has been subject to an extensive in situ monitoring campaign. Even if this bridge has been studied from a numerical point of view, in this case study the attention of the author is addressed to the definition, implementation and validation of an alternative model for the pedestrians (crowd) loads based on a stochastic approach, as discussed in section 4.2.

More details about the two structures under study are given in *Appendix A* (section A.1 and section A.2, respectively). For the numerical models and the achieved results the reader is referred to *Chapter 3* from to *Chapter 6*.

To conclude this introduction of timber bridges, in some countries different timber solutions are employed not only for "pedestrian" application but also as classical road bridges. In the following pictures (from Figure 2.3.8 to Figure 2.3.12) a few examples of road timber bridges spread all around the world, are presented. They are taken from the proceeding of the last international conference in timber bridges ICTB2013 held in Las Vegas last October, 2013.

The first one, see Figure 2.3.8, is located in Norway and it is a classical arch bridge [40].



Figure 2.3.8. Tynset bridge, Norway [40].

Another example, see Figure 2.3.9, is represented by the Cosumnes River Bridge located near Sacramento, California (USA). This is a classical truss bridge [41].



Figure 2.3.9. Cosumnes River Bridge, California (USA) [41].

An example of a five-span timber structure is the Goodpasture Covered Bridge (see Figure 2.3.10) crossing the McKenzie River in eastern Lane County, Oregon (USA) [42].



Figure 2.3.10. Goodpasture Covered Bridge. Lane County, Oregon (USA) [42]. a) lateral view; b) frontal view.

Another example, see Figure 2.3.11, of arch timber bridge is here below presented is located close to the town Gislaved (Sweden) and it is one of the infrastructures belonging to Highway 27 [43].



Figure 2.3.11. Goodpasture Covered Bridge. Lane County, Oregon (USA) [43].

To conclude this part in the next picture the oldest road timber bridge in Skelleftea (Sweden), see Figure 2.3.12, completed in 1737 is reported [44].



Figure 2.3.12. Lejonstroms Bridge. Skelleftea, Sweden [44].

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# Chapter 3 Experimental campaign and numerical modeling

In this chapter the numerical models of the two pedestrian bridges under study (section 2.3) are presented. For more details about the pedestrian bridges the reader is referred to *Appendix A*.

All the structural models are developed within the theory of Finite Element Model (FEM) [1]. The code adopted is developed by MSC Marc software-house. In particular the graphical interface called MENTAT2010 [2] is used. This software allows one a complete solution (pre-processing solution and post-processing solution) for implicit nonlinear Finite Element Analysis (FEA). It allows one to accurately simulate the response of structures under static, dynamic and multi-physics loading scenarios. Marc's versatility in modelling nonlinear material behaviours and transient environmental conditions makes it an efficient tool for the design of structural systems. Moreover it allows the user to define the *ad hoc* functions for boundary conditions, loads configuration and so forth...

### 3.1 Experimental campaign

In this section the experimental campaigns carried out in situ for the two structures under study are presented. It is worth underlining that these tests are a must in view of defining more realistic numerical models to represent the dynamic behaviour of the footbridges. In particular the calibration and validation of the FE models will be entrusted to the comparison of the responses of the footbridges under wind loads acquired in situ, with the responses achieved by the implemented numerical models. Indeed, in this particular application, the loads associated to the wind can be regarded as "ambient vibration" or "background noise". During the experimental campaigns, long acquisitions of the structural response of the bridges under "low-wind-velocity" were collected for this purpose.

## 3.1.1 "Tesa" footbridge experimental campaign

On November 7, 2013 an experimental campaign was performed with the aim of the acquisition of the bridge dynamic response (i.e. the acceleration components of relevant points along the footbridge deck) under different loading conditions [3]-[4].

The data acquisition is tailored to standard tri- and uni- axial *Kinemetrics EPISENSOR* accelerometer (Figure 3.1.1), while the data transmission is entrusted to wireless transceiver [5] in order to cover all the entire span length of the bridge, without the need for the intermediate storage stations. An example of these devices is reported in Figure 3.1.2.



Figure 3.1.1. Example of the tri-axial accelerometer posed over the deck.



Figure 3.1.2. Example of the wireless station unit (WSU) employed for the data acquisition-transmission.

During the in situ tests, a total of six accelerometers is deployed; in particular:

- two uni-axial accelerometers anchored to the "internal-main" cables, labelled as WSa5\_ch1 and WSa5\_ch2;
- four tri-axial devices labelled as WSa*i*, *i*=1,..,4, where *i* represents the *i*-th accelerometers, along the centreline of the deck in *ad hoc* positions. More details are shown in Figure 3.1.3.

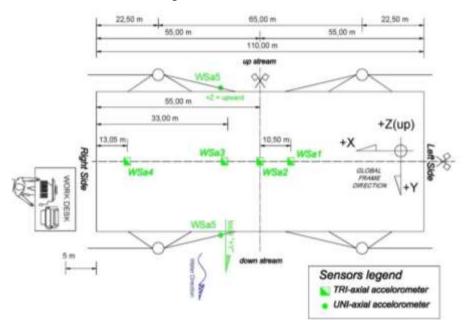


Figure 3.1.3. Sensors equipment configuration during the in situ tests.

Figures 3.14, 3.15 and 3.16 show the positions of the sensors WSa1, WSa2 and WSa3.



Figure 3.1.4. Sensors WSa1 during the in situ tests.



Figure 3.1.5. Sensors WSa2 during the in situ tests.



Figure 3.1.6. Sensors WSa3 during the in situ tests.

Two groups of tests are carried out during the experimental campaign:

- *test A environmental loads*, i.e. wind (the bridge location is often interested by strong winds) used as "calibration" test for the instrumentations, and for the first "validation/calibration" of the numerical model;
- test B running and walking along the deck. Different schemes of "human induced loads" (HIL) are defined. For the walking loads, three different cases are analyzed: a round trip along the bridge of one person (test B1), four persons (test B2 see Figure 3.1.7) and six persons (test B3). For the running configuration, the case of a round trip of one person is considered (test B4). More details will be presented in section 4.3.2 and in Appendix D discussing the validation of the model for the crowd load proposed in this thesis.

All the accelerations are acquired with a sampling frequency of 100 Hz.

An example of the accelerations acquired by WSa2 along the vertical axis Z for *test A* and for *test B2* is reported in Figure 3.1.8 and Figure 3.1.9 respectively.



Figure 3.1.7. "Tesa" footbridge – picture during test *B2*.

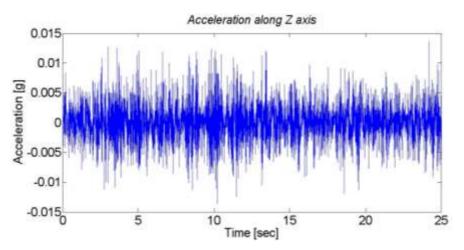


Figure 3.1.8. Acceleration along the Z axis recorded by WSa2 – duration 25 sec – sampling rate 100 Hz – November 7, 2013 @ 2.12 pm – air temperature 15°C.

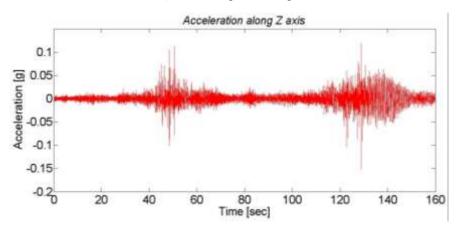


Figure 3.1.9. Acceleration along Z axe recorded (160 sec) by WSa2 – sampling rate 100 Hz – November 7, 2013 @ 2.55 pm – air temperature 15°C.

All the recorded data are analyzed following four different approaches:

- 1) standard Fast Fourier Transform (FFT);
- 2) Short Time Fast Fourier Transform (STFFT) ([6]-[7] among others);
- 3) Wavelet algorithm (WA) ([8]-[9] among others);
- 4) Stochastic Subspace Identification theory (SSI) ([10]-[12] among others).

All the calculations are performed within the MatLAB environment [13]; the SSI analysis is carried out by using the MACEC toolbox of MatLAB [14]. In particular the

methodologies STFFT, WA and SSI were used for better identify the interaction behaviour between the pedestrian and the bridge as better described in *Chapter 4*.

As an example, the results achieved applying the different methodologies described above for the acceleration acquired by WAa1 during the *test* A along Y transversal axis are represented from Figure 3.1.12 to Figure 3.1.12. The results are summarized and compared in Table 3.1.1.

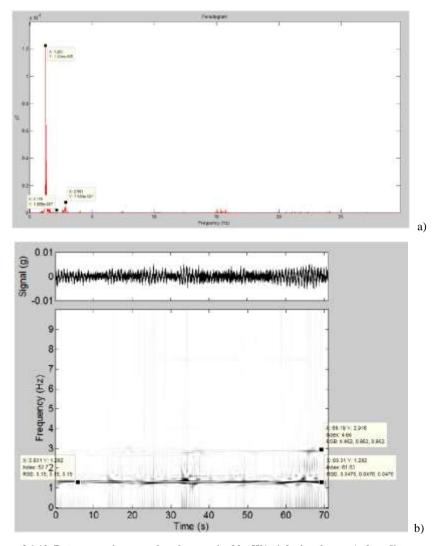


Figure 3.1.10. Data processing – acceleration acquired by WAa1 during the *test A* along *Y* transversal axis. a) FFT; b) STFFT.

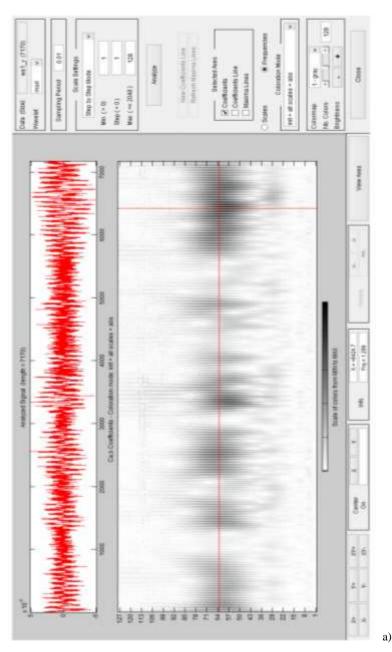


Figure 3.1.11. Data processing – acceleration acquired by WAa1 during the *test A* along *Y* transversal axis. a) WA.

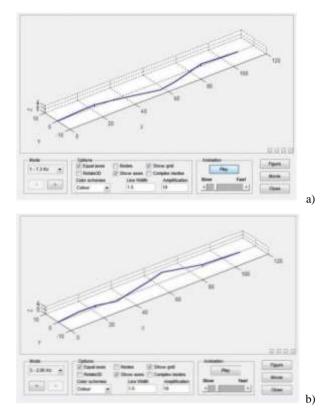


Figure 3.1.12. Data processing by using SSI – acceleration acquired by WAa1 during the *test A* along *Y* transversal axis. a) mode I (*f*=1.30 Hz); b) mode III (*f*=2.85 Hz).

Table 3.1.1. "	1 669 1	100101000 =	treatiencies	range trom	data anaiveic
1 abic 3.1.1.	I USA	TOOLDITUGE	ii cquencies	range nom	uata analysis.

Algorithm	Frequency [Hz]	damping [-]
FFT	1.283	-
	2.901	
STFFT	2.900	-
	1.300	
WA	2.900	-
SSI	1.311	1.84 %
~	2.850	3.163 %

## 3.1.2 "Trasaghis" footbridge experimental campaign

Two experimental campaigns were planned. The accelerations in previously selected points of the bridge under environmental loads were collected. Two challenges were met:

- 1) replace the standard wired acquisition system;
- 2) create a database of data to develop a (future) reliable process to clean the effects of current environment conditions, that might afflict different experimental campaigns.

About the first aspect the classical cable solution for the data acquisition is able to cover distance up to about 30 m. Being the span of the bridge about 85 m long, to overcome this experimental gap without use of signal repetition bridge, the transmission of the data was entrusted to a wireless technology (Figure 3.1.13) as illustrated in [5] and [16].



Figure 3.1.13. The tri-axial accelerometer connected with a wireless sensor unit (WSU) posed over the deck.

To acquire the acceleration standard tri- and uni-axial *Kinemetrics EPISENSOR* accelerometers were employed (Figure 3.1.14). In total six accelerometer were used: four tri-axial and 2 uni-axial.



Figure 3.1.14. Example of the Kinemetric EPISENSOR uni- and tri- axial accelerometer employed during the experimental campaigns.

The first set of experiments was carried out on November 30, 2012. The average temperature of the air along the data acquisition was about 10 °C.

The second campaign of experiments was carried out on May 12, 2013. Two sets of data from this test are reported:

- the first one was collected in the morning (9.30 a.m. temperature of air about 15.5 °C);
- the second one at noon (12:00 a.m. temperature of the air about 25 °C).

This test repetition was planned for investigating changes in the structure behavior related to environment condition variations.

The configuration of the sensors for both the campaigns is given in Figure 3.1.15.

The acquired data are then elaborated using the standard approach proposed by the classical Fast Fourier Transformation (FFT). The related acceleration periodograms for the components along the transversal (Y) and vertical (Z) axes are given in Figure 3.1.16 and Figure 3.1.17. As an example the data acquired by the device in the position labelled *P#3* are reported.

The analysis of these periodograms shows the same ranges for the frequencies. In particular the experimental results confirmed the low sensitivity of the dynamic response to temperature variations, i.e., the prevalence of the steel skeleton on determining the footbridge vibration. A summary in reported in Table 3.1.2.

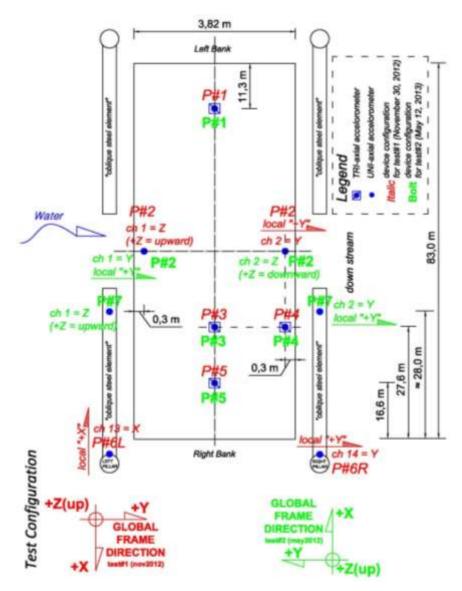


Figure 3.1.15. Sensors configuration for the two experimental campaign (November 30, 2012 and May 12, 2013).

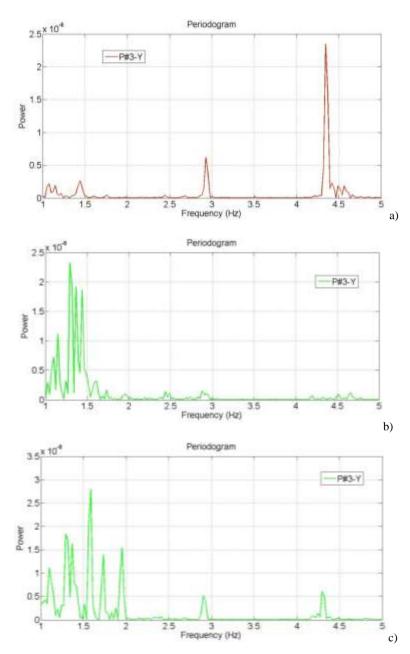


Figure 3.1.16. Periodograms of the signals recorded at position 3 (P#3): from the top to bottom, one sees the elaboration of the signals (along direction Y) collected in November, in the early morning of May 12, 2013 and in the late morning of the same day. The number of points in the temporal window of the signal is 2170.

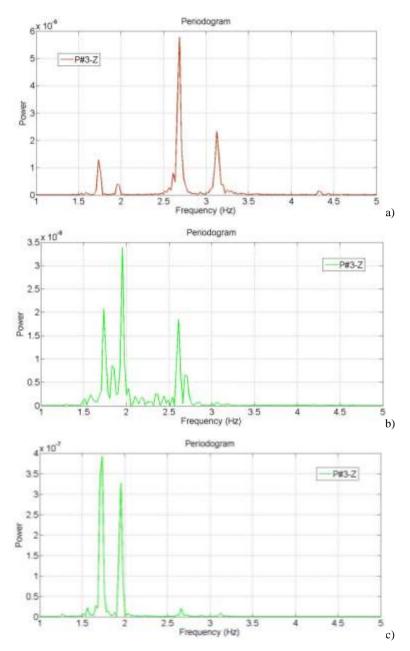


Figure 3.1.17. Periodograms of the signals recorded at position 3 (P#3): from the top to bottom, one sees the elaboration of the signals (along direction Z) collected in November, in the early morning of May 12, 2013 and in the late morning of the same day. The number of points in the temporal window of the signal is 2170.

Set ID	Frequency [Hz]	Oscillation plane
1	1.05	X - Z
2	1.40	X - Y
3	1.75	Ζ
4	1.96	<i>X</i> - <i>Y</i> & <i>X</i> - <i>Z</i>
5	2.55	Ζ

Table 3.1.2. Frequency ranges "Trasaghis" footbridge - main results.

From the analysis of the signal recorded during these tests different frequencies peaks appear in the range under about  $1.40 \pm 1.50$  Hz. Numerical investigation and focused analysis on the acquired data on field showed as these frequencies can be associated to the dynamic behaviour of the stay and the deck-stay connection (Figure 3.1.18 and Figure 3.1.19). More details are provided in section 3.3.2.

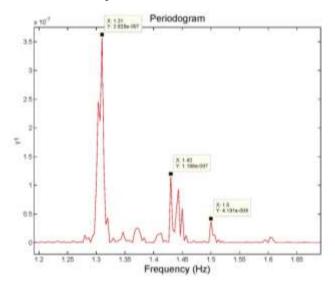


Figure 3.1.18. Periodograms of the signals recorded at position 7 (P#7): FFT along direction Y.

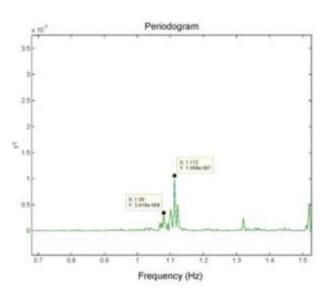


Figure 3.1.19. Periodograms of the signals recorded at position 7 (P#7): FFT along direction Z - data collected in May 12, 2013.

The measurements of the campaign carried out in May, 2013 were collected during two different tests: in the early morning the first, and at noon the second one. These tests are justified by the wish of extracting the variations in the structure performance due to changes in the environment conditions. There are small discrepancies between the values recorded in the early morning and the late morning, but the deviation is within confidence intervals. In other words the second campaign allows one to confirm the ranges of frequency covered by the vibration of the different elements, even if some small deviations associated to the temperature variation, have to be emphasized.

## 3.2 Cable-stayed system: "Tesa" footbridge FE Model

In this section the description of the "Tesa" footbridge numerical model is given. The model is developed within the Marc-MENTAT2010 software environment [2].

#### 3.2.1 Design data and numerical model definition

The static scheme of the bridge is inspired by the classic cable-stayed solution. Two pillars 16 m height are located on both sides of the bridge. On them sixteen steel cables are hanged to carry the deck. In particular eight "internal-secondary" cables of diameter 32 mm, and other eight "external-main" cables of diameter 44 mm are adopted. From a

geometric point of view, the span length is about 110 m subdivided into three segments of length 22.5m, 65m (in the middle) and 22.5 m, respectively. The free crossing width of the deck is 3.2 m.

Following the technical documents, details, information and so forth... provided by the designer of the footbridge, a first "rough" numerical model is built.

The global reference frame adopted follows the next rule:

- *X* axis: along the bridge axis, i.e. parallel to the deck;
- *Y* axis: transversal axis, i.e. orthogonal to the deck;
- Z axis: vertical axis parallel to the gravity axis.

A general view of the model implemented in which is represented in Figure 3.2.1.

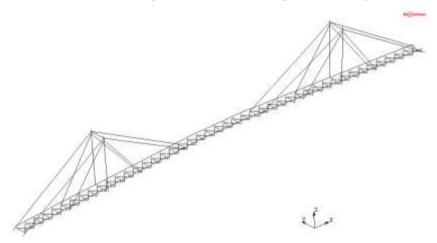


Figure 3.2.1. View of the numerical model implemented in MARC Mentat2010.

The model is defined by using the classical *beam* element. This class of elements is used to model the two mains GLT beams, the horizontal steel tubular element belonging to the U-shape resistant scheme of the deck, the four longitudinal timber beams that support the walking deck, the steel pillars. The cables are modeled using the standard equivalent stiffness technique; that is defining an equivalent value of the Young's modulus. The walking deck has not been modeled because it represents a non-structural elements. *Ad hoc* rigid links are used to connect the axes of the two main curved beams with the longitudinal beams to model the resistant scheme of the deck.

An overview of the geometry of the structural elements implemented in the FEM is summarized in Table 3.2.1.

ID	element type	size [mm]	note
GLT beam	beam	1238 by 18	solid cross-section
longitudinal timber beam	beam	163 by 100	solid cross-section
steel pillar	beam	radius = 228.6 - th. 12.5	tubular cross-section
cable	-	diameter: 32 and 44	no compression
mains steel braces	beam	diameter: 25	solid cross-section
U-shape steel elements	beam	100 by 200 – th. 5	tubular cross-section

Table 3.2.1. "Tesa" footbridge - numerical model: mains features.

A summary of the model properties in given in Table 3.2.2.

Table 3.2.2. "Tesa" footbridge - numerical model: main features.

ID	TOT [num]	
nodes	610	
beam	837	
rigid link	90	

The materials properties are summarized in Table 3.2.3. Due to the complexity of the problem, for all structural elements the standard isotropic behavior is adopted according to the structural code [17].

All the non-structural elements masses have been evaluated following the technical documents and implemented in the numerical model, in order to ensure a consistent dynamic response of the model.

Table 3.2.3. "Tesa" footbridge - numerical model: materials properties.

	Timber – BS14 & BS16	Cable ¢32	Cable <i>\$44</i>	Steel - S355J0
mass density [Kg/m <sup>3</sup> ]				
ρ	500.00	5620.00	8260.00	7850.00
Young's Modulus [N/m <sup>2</sup> ]				
E	1.10+10	1.65 + 11	1.65 + 11	2.06 e+11
Poisson's Coefficients [-]				
V	0.10	0.30	0.30	0.30

The following kinematics boundary are introduced:

- for the base of the pillars and for the stays connected with the ground all the degree of freedom are assumed fixed; that is  $D_x=D_y=D_z=R_x=R_y=R_z=0$ ;
- to model the presence of the *neoprene* support below the mains GLT beams the following degree of freedom are assumed fixed; that is  $D_y=D_z=R_z=0$ .

More details about the kinematics boundary conditions are summarized in Figure 3.2.2 and Figure 3.2.3.

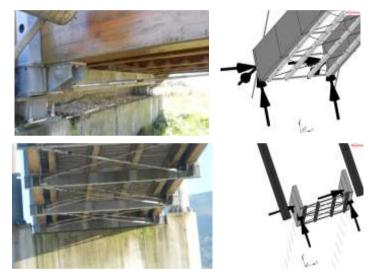


Figure 3.2.2. Detail of the kinematics boundary condition below the mains GLT beams. On the left picture from field; on the right the implemented boundary on the numerical model.



Figure 3.2.3. Detail of the kinematics boundary condition at the base of the pillars. On the left picture from field; on the right the implemented boundary on the numerical model.

Finally an *ad hoc* pre-strain is given to the "external" cables to achieve the final geometrical configuration of the bridge (Figure 3.2.4).

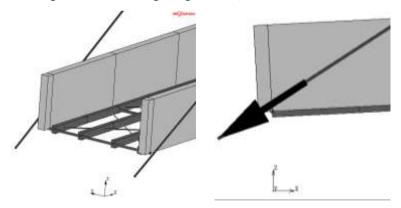


Figure 3.2.4. Detail of the kinematics boundary condition at the base of the pillars. On the left picture from field; on the right the implemented boundary on the numerical model.

To conclude this part, some plots of the FEM are reported in the figures from 3.2.5 to 3.2.8.

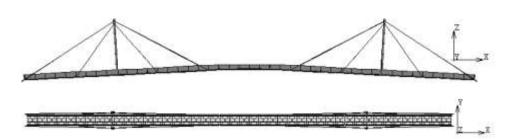


Figure 3.2.5. Above: lateral view of the numerical model; below: view from the above.



Figure 3.2.6. Frontal view of the numerical model.



Figure 3.2.7. 3-D view of the numerical model.

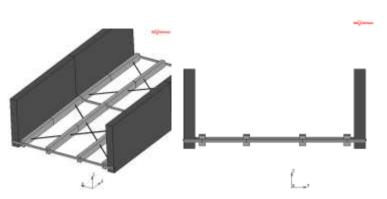


Figure 3.2.8. View of the deck. On the left 3-D view; on the right cross-section view.

In order to take into account the geometric non linearity due to the presence of the cables, the "large strain" option has been activated in the software configuration.

It is worth underlining that, even if the whole general dynamic problems is not linear due to the geometric non linearity introduced by the cables, all the dynamic analysis are conducted within small oscillations around the deformed configuration of equilibrium achieved by the bridge.

#### 3.2.2 FE Model refinement and validation

In order to calibrate and validate the numerical model developed, all the data recorded on the field during *test* A (section 3.1.1) are analyzed following the standard Fast Fourier Transform (FFT) approach.

The results achieved for the acceleration acquired by WAa1 during the *test A* along the *Y* transversal axis are represented in Figure 3.2.9. The related results are summarized and compared in Table 3.2.4.

Based on the results achieved analysing the data acquired on field, the numerical model previously defined has been refined (e.g. the nodal mass associated to the non-structural elements has been modified) until the dynamic modal analysis associated to the FEM "match well" with the experimental evidence.

The comparison between the analysis of the data coming from the field and the refined numerical model is reported in Figure 3.2.10.

As observed previously even if this bridge has been subject to an extensive study both from the numerical point of view and from the analysis of the data acquired during the in situ monitoring campaign, the attention on the "Tesa" footbridge is mainly addressed to the definition, implementation and validation of a model for the pedestrian loads.

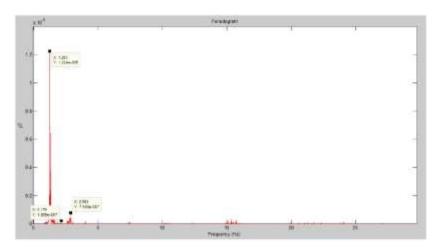


Figure 3.2.9. FFT – acceleration acquired by WAa1 during the *test A* along the *Y* axis. Table 3.2.4. "Tesa" footbridge – frequencies comparison.

Algorithm	Frequency [Hz]

Algorithm	Frequency [Hz]
FFT	1.283
ГГІ	2.901

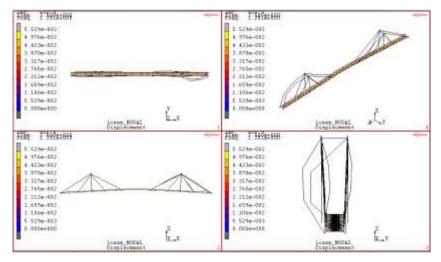


Figure 3.2.10. Mode I – prevalent motion along the transversal direction Y. FEM dynamic modal analysis performed in MARC Mentat2010 – f = 1.28 Hz.

## **3.3 FE Model of the "Trasaghis" footbridge**

In this section the description of the "Trasaghis" footbridge numerical model is given. The model is developed within the Marc-MENTAT2010 software environment [2].

#### 3.3.1 Project data and numerical model definition

Despite the bridge looks of the cable-stayed type, the static scheme sees a vertical antenna on both sides but the cables are replaced by tubular steel elements. This assumption during the design stage made quite complicated the definition and calibration of the numerical model as described later on in this section.

Starting for the technical documents provided by the designer of the footbridge, a first "rough" numerical model is built following all the design details. In general the span is 83 m and the double beam deck width is 4 m, of which 3.22 m represents the free crossing width for the pedestrians.

The reference global frame has been selected with the:

- *X* axis: along the bridge axis, i.e. parallel to the deck;
- *Y* axis: transversal axis, i.e. orthogonal to the deck;
- *Z* axis: vertical axis parallel to the gravity axis.

A general view of the model implemented in which is represented in Figure 3.3.1.

The model is composed by classical *beam* and *shell* elements. In particular the beam elements are used to model the H-shape steel tubular element of the deck, the five longitudinal timber beams that carry the walking deck, the steel tubular elements of the pillars and oblique stays and the deck braces; while the shell elements are employed to model the two mains GLT beams and walking timber deck. Moreover *ad hoc* rigid links are used to connect the walking deck with the longitudinal beams to model the real space between these elements. An overview of the geometry of the structural elements implemented in the FEM is summarized in Table 3.3.1.

The materials properties are summarized in Table 3.3.2. In particular for the timber elements (according to the structural code [17]), the orthotropic behavior is adopted to represent better their mechanical properties [18].

A summary of the main features of the FE model is given in Table 3.3.3.

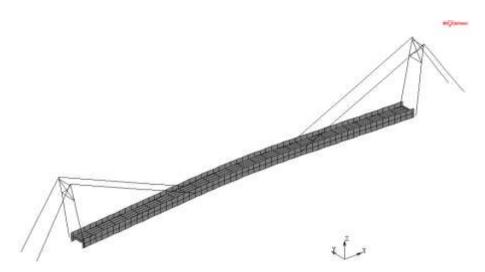


Figure 3.3.1. View of the numerical model implemented in MARC Mentat2010.

ID	element type	size [mm]	note
GLT beam	shell	2950 by 1000 – th. 200	-
walking deck	shell	2950 by 725 – th. 63	-
longitudinal timber beam	beam	163 by 120	solid cross-section
steel pillar	beam	radius = 228.6 - th. 16	tubular cross-section
steel stay	beam	radius = 136.5 - th. 8	tubular cross-section
mains steel braces	beam	80 by 10	solid cross-section
H-shape steel elements	beam	1200 by 200 – th. 63	tubular cross-section

Table 3.3.1. "Trasaghis" footbridge - numerical model: geometry of the structural elements.

	Timber - GL28c	Timber - GL24c	Steel - S355JR
mass density [Kg/m <sup>3</sup> ]			I
ρ	509.68	509.68	7645.26
Young's Modulus [N/m <sup>2</sup> ]			L
$E_{I}$	1.26e+10	1.16e+10	
$E_2$	3.90e+08	3.20e+08	2.10e+11
$E_3$	3.90e+08	3.20e+08	
Poisson's Coefficients [-]			
$V_l$	0.40	0.40	
$V_2$	0.40	0.40	0.30
V3	0.40	0.40	
Shear's Modulus [N/m <sup>2</sup> ]			
$G_{12}$	7.20+08	5.90+08	
$G_{23}$	7.20+08	5.90+08	8.10e+10
$G_{31}$	7.20+08	5.90+08	

Table 3.3.2. "Trasaghis" footbridge - numerical model: materials properties.

Table 3.3.3. "Trasaghis" footbridge - numerical model: main features.

ID	TOT [num]
nodes	724
beam	572
shell	352
rigid link	145

For the kinematics boundary conditions the following assumptions are assumed:

- for the base of the pillars and for the stays connected with the ground all the degree of freedom are assumed fixed; that is D<sub>x</sub>=D<sub>y</sub>=D<sub>z</sub>= R<sub>x</sub>=R<sub>y</sub>=R<sub>z</sub>=0;
- below each ends of the mains GLT beams a standard abutment is implemented to model the presence of the *neoprene* support;
- to represent the link located to each ends of the GLT beams between the foundation system and the GLT beam, the following kinematic boundary is implemented:  $D_x=D_y=D_z=R_y=R_z=0$ .

More details on the kinematics boundary conditions are summarized from Figure 3.3.2 to Figure 3.3.4.

Great care was paid to model the mass of all the non-structural elements because of their fundamental importance when performing dynamic analysis; i.e. dynamic modal and dynamic transient analysis. Thus, following the technical documents, all these mass are first valuated and then implemented within the numerical model as "lamped mass" along the deck.

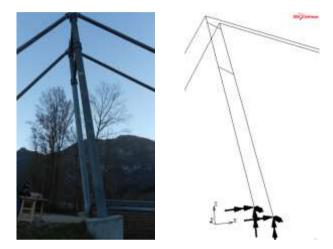


Figure 3.3.2. Detail of the kinematics boundary condition at the base of the pillars. On the left picture from field; on the right the implemented boundary on the numerical model.



Figure 3.3.3. Detail of the kinematics boundary condition at the base of the stays. On the left picture from field; on the right the implemented boundary on the numerical model.

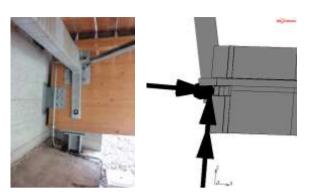


Figure 3.3.4. Detail of the kinematics boundary condition for the GLT beams. On the left picture from field; on the right the implemented boundary on the numerical model.

Great care was paid to model the damping of the structure. In particular the damping properties are fitted by identifying the Rayleigh's coefficients (see equation (2.29)) based on the first experimental campaign carried out in this bridge. The environmental test shown two peaks along Z axis in terms of frequencies; that is  $f_1 = 1.75$  Hz and  $f_2 = 2.67$  Hz. By applying the bandwidth method presented in section 2.1, the following damping value can be evaluated:  $\xi_1 = 0.0092$  and  $\xi_2 = 0.0056$  respectively for the frequencies  $f_1$  and  $f_2$ . Thus, the Rayleigh's coefficients can be evaluated as:

$$\begin{cases} \alpha_{1} = \frac{2\omega_{1}\omega_{2}\left(\xi_{1}\omega_{1} - \xi_{2}\omega_{2}\right)}{\omega_{1}^{2} - \omega_{2}^{2}}; \\ \alpha_{2} = \frac{2\left(\xi_{1}\omega_{1} - \xi_{2}\omega_{2}\right)}{\omega_{1}^{2} - \omega_{2}^{2}}; \end{cases}$$
(3.1)

where  $\omega_1 = 2\pi f_1 = 11.0$  rad/s and  $\omega_2 = 2\pi f_2 = 16.7$  rad/s. Substituting these values in the above equation ones obtains  $\alpha_{I} = 0.1681$  and  $\alpha_2 = 11.63e-5$ . These parameters are implemented within the code for the element that compose the two main GLT beams. This approach for the estimation of the modal damping properties is an alternative way which is often followed in the modelling process, especially when one preliminarily knows that are the most significant modes.

To verify the goodness of the implemented model, the maximum vertical displacement at the mid-span obtained by the simulation is compared with the one obtained during the loading test. The load test was carried out in order to simulate the uniformly distributed load required by the regulations [17] equal to 400 kg/m<sup>2</sup> (more details are given in section A.1). The result from the numerical analysis differs by about 1.50 cm from the

displacement recorded in field<sup>9</sup>, so the model can be considered validated from the static point of view.

To conclude this part some plots of the FEM are reported in figures 3.3.5 to 3.3.10.

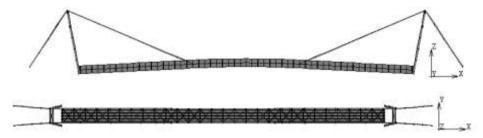


Figure 3.3.5. Above: lateral view of the numerical model; below: view from the above.



Figure 3.3.6. Frontal view of the numerical model.

<sup>&</sup>lt;sup>9</sup> About 13.2 cm evaluated by the FEM against 14.5 cm recorded during the loading test.

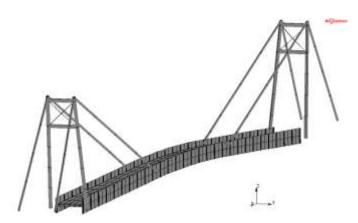


Figure 3.3.7. 3-D view of the numerical model.

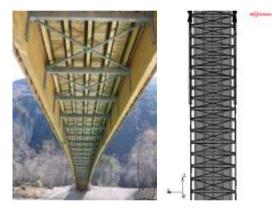


Figure 3.3.8. View from the bottom of the deck. On the left picture from field; on the right the implemented boundary on the numerical model.

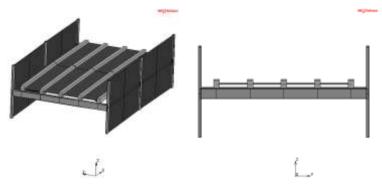


Figure 3.3.9. View of the deck. On the left 3-D view; on the right cross-section view.

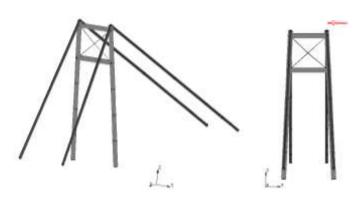


Figure 3.3.10. View of the steel pillars and stays systems. On the left 3-D view; on the right cross-section view.

#### 3.3.2 FE Model refinement and validation

Based on the main results of the experimental campaigns, in terms of signal analysis by the standard FFT transform, the numerical model described is then refined, calibrated and validated from dynamic point of view.

The behaviour of the link between the steel stay elements and the deck, and their role with respect to the whole dynamic response of the bridge, is not a priori known. For better investigate this aspect a preliminary numerical model (Figure 3.3.11) of the steel portal is implemented in MARC Mentat2010 [2]. All the elements touching the ground are regarded as fully fixed; that is both translations and rotations are assumed to be zero ([19] and [20]).

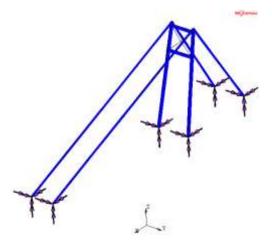


Figure 3.3.11. Partial numerical of the steel pillars and stays implemented in MARC Mentat2010.

The modal analysis of this partial model provides the range of frequencies that can be associated with the single components of the steel supporting system. Four situations are of interest for this study:

- the vibration of the longest elements (the ones linking the deck with the antenna), the vibration relevant frequency as *f*<sub>1</sub>;
- the vibration associated with an in-plane (transversal to the bridge axis) movement of the single antenna portal relevant frequency as  $f_2$ ;
- the vibration of the rear oblique elements of the antenna relevant frequency as *f*<sub>3</sub>;
- the vibration of the vertical columns forming the antenna relevant frequency as *f*<sub>4</sub>.

Of course, each of the frequency values introduced above, denotes the lower bound of a frequency band where several modes show the vibration of these elements. Indeed the vibration in two parallel elements can occur in different planes (either in-plane or out-of-plane) and can combine with different signs in the structural system giving rise to a multiplicity of modes. The values of  $f_1, \ldots, f_4$ , were then also calculated under different constraints at the points where the link with the deck will be realized (Table 3.3.4).

Table 3.3.4. Partial FEM -	<ul> <li>Frequency ranges for</li> </ul>	different boundary	conditions for the deck-stay link.

Boundary conditions for the deck-stay link	Frequency [Hz]			
	$f_{I}$	$f_2$	$f_3$	$f_4$
fixed ends	1.52-1.54	3.43	5.60-6.61	8.73
A - free vertical translation	1.52-1.54	3.43	5.60-6.61	8.73
B - free transversal translation	0.39-2.06	3.43	5.52-6.60	8.66
C - free rotation around the transversal axis	1.07-1.54	3.43	4.25-6.61	8.40
D – free rotation around the vertical axis	1.08-1.54	3.36	4.33-6.56	8.73
A+B	0.39-2.05	3.46	5.52-6.73	8.66
C+D	1.07-1.09	3.36	4.25-6.56	8.40
A+C+D	1.07-1.09	3.35	4.20-6.54	8.37

The recorded signals and the numerical model signals confirm the role of the long tubular stays in determining the vibration of the system at frequencies in the range from 1 to 2 Hz. Within this range, actually, the possible combinations of their vertical (in plane) movements correspond to the lowest frequency value, and those of their horizontal (out-of-plane) movements correspond to an intermediate frequency range, both resulting in modest movements of the deck. This different orthogonal behavior is due to the connections of the stays, which are represented by hinges in the vertical plane, but they can transfer the moments in the orthogonal plane. A further slightly higher frequency shows the movement of all the oblique elements in the same (transversal/out-of-plane) direction resulting in "a torsion" of the central third of the deck. A short summary of the frequencies of interesting for this study is given in Table 3.3.5.

Following the above observations, the numerical model of the whole bridge is so refined (e.g. the nodal mass associated to the non-structural elements has been modified) until the dynamic modal analysis associated to the FEM "match well" with the experimental evidence. An example of the validation is reported from Figure 3.3.12 to Figure 3.3.17. A dynamic modal comparison in terms of frequencies range for the main modes between the results is reported in Table 3.3.6. Some small discrepancies appear but, however, they can be assumed acceptable for the purpose of this work, taking also in account the complexity of the structural scheme of the bridge (i.e. the coupling between steel and wood in the resistant scheme of the deck; the steel stays, and so forth...).

Set ID	Frequency	Involved	Oscillation plane	
	[ <b>H</b> z]	Elements		
1	1.06	Tubular stay	X - Z	
2	1.41	Tubular stay	X - Y	
3	1.96	Tubular stay	<i>X</i> - <i>Y</i> & <i>X</i> - <i>Z</i>	

Table 3.3.5. Frequency ranges for the lowest modes of the "Trasaghis" footbridge.

Table 3.3.6. Frequencies percentage error. Comparison between "field" and "FEM" results.

ID	$\varepsilon = \left  \frac{f_{field} - f_{FEM}}{f_{FEM}} \right  x100$		
mode I	2.3 %		
mode II	2.8 %		
mode III	5.8 %		

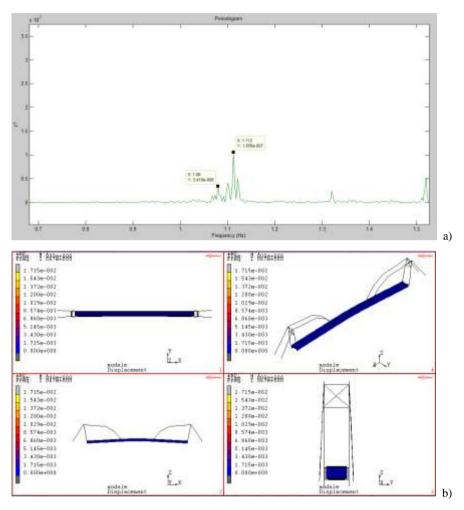


Figure 3.3.12. Mode I – local mode related to the prevalent vertical motion of the stays. a) FFT of the experimental data (Campaign of May, 2013 – position P#7) –  $f = 1.08\pm1.11$  Hz; b) FEM dynamic modal analysis performed in MARC Mentat2010 – f = 1.06 Hz.

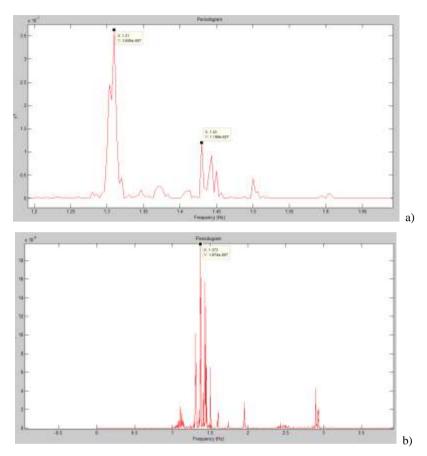


Figure 3.3.13. Mode II – local mode related to the prevalent horizontal/transversal motion of the stays.
a) FFT of the experimental data along *Y* (Campaign of May, 2013 – position *P#7*) – *f* = 1.31±1.43 Hz; b)
FFT of the experimental data along *Y* (Campaign of May, 2013 – position *P#1*) – *f* = 1.37 Hz.

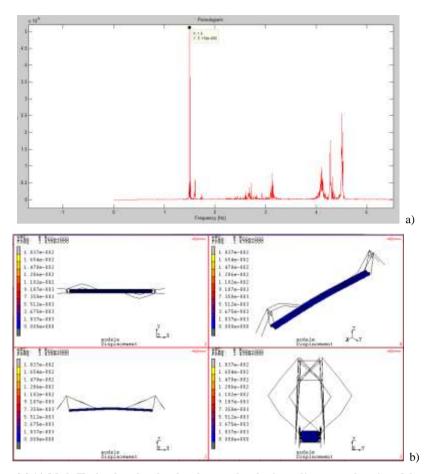


Figure 3.3.14. Mode II – local mode related to the prevalent horizontal/transversal motion of the stays. a) FFT of the experimental data along Z (Campaign of May, 2013 – position P#I) – f = 1.5 Hz; b) FEM dynamic modal analysis performed in MARC Mentat2010 – f = 1.41 Hz.

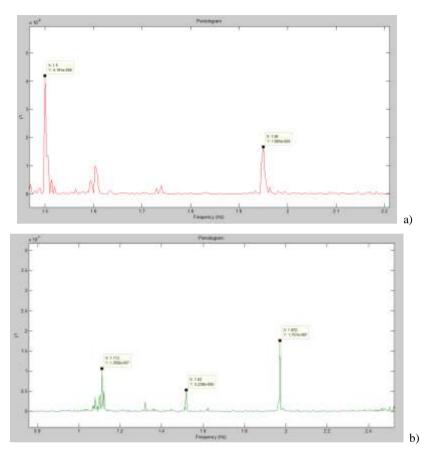


Figure 3.3.15. Mode III – global "pseudo" torsional mode of the deck. a) FFT of the experimental data along *Y* (Campaign of May, 2013 – position *P#7*) – f = 1.95 Hz; b) FFT of the experimental data along *Y* (Campaign of May, 2013 – position *P#1*) – f = 1.97 Hz.

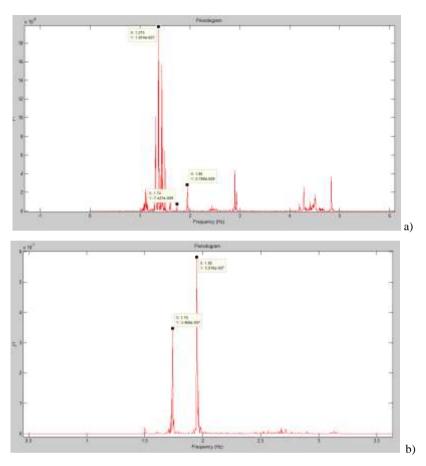


Figure 3.3.16. Mode III – global "pseudo" torsional mode of the deck. a) FFT of the experimental data (Campaign of May, 2013 – position P#3) –  $f = 1.74\pm1.95$  Hz; b) FFT of the experimental data along Y (Campaign of May, 2013 – position P#3) –  $f = 1.74\pm1.95$  Hz.

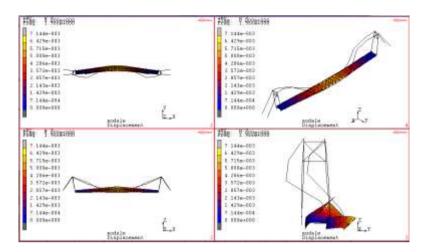


Figure 3.3.17. Mode III – global "pseudo" torsional mode of the deck. FEM dynamic modal analysis performed in MARC Mentat2010 – f = 1.96 Hz.

### 3.4 References

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# **Chapter 4** Modeling the main actions

As already pointed out in *Chapter 2*, when one works with the design and/or control of slender structures as pedestrian bridges, generally the main problems are related with the comfort of the users rather than with the strength of the construction materials. Thus an inaccurate modelling definition of the load(s) at the design stage could result dangerous in term of both resistance and comfort.

Thus, one of the main challenges for the designer is to identify the most suitable model of the *live load(s)*, to be applied on the structure in order to capture/identify the most reliable response of the structure under study. To achieve this purpose a pure theoretical definition of the loads could result unsatisfactory. The best result is achieved merging theoretical aspects with observations made in laboratory and/or in the field on a real case study.

An incorrect definition of the live loads during the design stage can be produces two concatenated effects in terms of comfort; they are:

- an ineffective performance with, in the worst case, a decrease in the achieved comfort level;
- a damage from the economic point of view. In fact, usually, in order to achieve a better level of comfort one needs to install *ad hoc* devices on the structure, i.e. TMDs, dampers and so forth..., that are expensive in term of both installation and maintenance.

In this chapter, models of the main loads to be considered in the design of pedestrian bridges are presented ([1] and [2]) for:

- 1) the wind load;
- 2) the Human Induced Loads (HIL).

For both of them one is looking for models which allows the simulation of time histories of random field realizations.

For the first one the procedure based on a stochastic simulation approach, comes from the literature [9] (see section 4.1). For the case of the human induced vibration a new model for the crowd load is discussed in section 4.2.

#### 4.1 Wind action

In previous chapter, the wind has been introduced as "ambient vibration" (or background noise) and it has been used for the calibration and validation of the numerical models of the two footbridges under study. In this section a more sophisticated mathematical model of the wind is introduced for the purpose of this thesis: i.e. implementation of different control solutions for the mitigation of the footbridge vibrations.

In wind engineering, Monte Carlo simulation is becoming one of the most important tool for the design of wind-excited structures [3]. The following formulas describe the procedure adopted in this work.

The wind velocity is usually idealized as the sum of two components: the first one, the "mean" part (U), is assumed to be constant along a given time interval; the second component, the "fluctuating" part (u), represents the atmospheric turbulence. It is usually modelled as a stationary zero mean Gaussian random process. To simulate the fluctuating part the "weighted amplitude waves superposition" (WAWS) method is adopted in this thesis [4]-[7].

Let  $\{O; x_1x_2x_3\}$  be the global Cartesian reference system with *O* lying on the ground, and the  $x_3$  axis parallel to the gravity direction (Figure 4.1.1). One also assumes, without less of generality, that **U** is parallel to the  $x_1$  axis. Within the above hypotheses one can write:

$$\mathbf{V}(\mathbf{x},t) = \mathbf{U}(x_3) + \mathbf{u}(\mathbf{x},t), \tag{4.1}$$

in which,

$$\mathbf{U}(x_3) = \frac{1}{k} u^* \ln\left(\frac{x_3}{z_0}\right), \text{ along } x_1 \text{ axis,}$$
(4.2)

where k=0.41 is the Von Karman constant [4],  $z_0$  the roughness length and  $u^*$  the shear velocity. An example [8] of the mean wind velocity profile with the height is reported in for three different scenario of roughness in Figure 4.1.2.

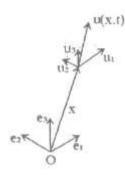


Figure 4.1.1. Global reference frame.

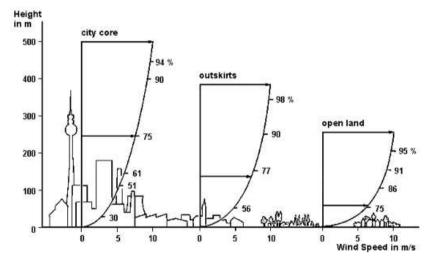


Figure 4.1.2. Example of "mean" wind velocity (U) trend for three different case of roughness [8].

Let  $\mathbf{x} = (x_1, x_2, x_3)$  be the vector position of any point *P* in the above frame, then one can introduce the fluctuating part of the wind velocity as  $\mathbf{u}(\mathbf{x}, t) = (u_1, u_2, u_3)$  where  $u_1, u_2, u_3$  are respectively the longitudinal, lateral and vertical components of turbulence as in Figure 4.1.1.

To introduce the turbulence field **u** as a zero mean, Gaussian, stationary random field, depending on time, one has to introduce the autocorrelation function  $R_{u_i u_j}(\mathbf{x}, \mathbf{x}', t, t')$  for each pair of points *P* and *P'* with absolute positions indicated by **x** and **x'**, respectively, in the above global frame. In particular one has:

$$R_{u_i u_j}\left(\mathbf{x}, \mathbf{x}', t, t'\right) = \mathbf{E}\left[u_i\left(\mathbf{x}, t\right), u_j\left(\mathbf{x}', t'\right)\right],$$
(4.3)
99

where  $\mathbf{E}[-]$  is the mean operator.

In the wind engineering field, it is reasonable assume **u** as an *ergodic* process; i.e. in other words, the statistical properties of the fluctuating part of the wind velocity could be deduced from a single, sufficiently long, sample of the process. By applying the Fourier transform of the cross power spectral density (CPSD) function between  $u_i$  and  $u_j$  the autocorrelation function is given as:

$$R_{u_{i}u_{j}}\left(\mathbf{x},\mathbf{x}',t,t'\right) = \int_{-\infty}^{+\infty} S_{u_{i}u_{j}}\left(\mathbf{x},\mathbf{x}',\omega\right) e^{i\omega\tau} d\omega, \qquad (4.4)$$

where  $\omega$  is the circular frequency,  $\tau = t - t'$  and *i* the imaginary unit.

The autocorrelation function depends on  $\tau$  (i.e. the process stationary in time) and on the positions **x** and **x'**. As a consequence the random field is not isotropic. Moreover it is important underline as this field is not homogeneous because the autocorrelation function depends separately from the height of the points **x**<sub>3</sub>, **x**<sub>3</sub><sup>'</sup> from the ground (with the zero level assumed as in Figure 4.1.1).

The model adopted in the following work describes the CPSD [6] in term of the autospectra  $S_{u_iu_j}(\mathbf{x},n)$ ,  $S_{u_iu_j}(\mathbf{x}',n)$  and coherence function  $Coh_{u_iu_j}(\mathbf{x},\mathbf{x}',n)$ , i, j = 1,2,3, in which  $n = \frac{\omega}{2\pi}$  represents the frequency.

Focusing the attention on the simulation of the fluctuating velocity in the plane  $(x_2x_3)$  it is reasonable to neglect the quadrature spectrum related to the imaginary part of the CPSD<sup>10</sup>.

By normalizing the auto-spectrum for the turbulence component  $u_i(\mathbf{x}, n)$ ,  $S_{u_i u_j}(\mathbf{x}, n)$ , one obtains:

$$\sigma_{u_i}^2\left(\mathbf{x}\right) = \int_{0}^{+\infty} S_{u_i u_j}\left(\mathbf{x}, n\right) dn, \qquad (4.5)$$

in which  $\sigma_{u_i}^2(\mathbf{x})$  represents the variance of the component  $u_i(\mathbf{x}, n)$ .

<sup>&</sup>lt;sup>10</sup> The quadrature spectrum becomes significant only for point located, far from each-others, along the wind direction  $x_l$ . So this aspect is not relevant for footbridges, and can be neglected without loos of generality.

Following the model proposed in [9] and [10] the auto-spectrum  $S_{u_i u_j}(\mathbf{x}, n)$  of the three components of the turbulence can be analytically given as:

$$\frac{nS_{u_{i}u_{j}}(\mathbf{x},n)}{\sigma_{u_{i}}^{2}(\mathbf{x})} = \frac{\lambda_{j}n(L_{u_{i}}(\mathbf{x})/\|\mathbf{U}(\mathbf{x})\|)}{(1+1.5\lambda_{j}n(L_{u_{i}}(\mathbf{x})/\|\mathbf{U}(\mathbf{x})\|))^{5/3}}, \quad (j = 1, 2, 3),$$
(4.6)

where  $L_{u_i}$  is the integral length scale of the turbulence component  $u_j$ , and  $\lambda_1 = 6.868$ ,  $\lambda_2 = 9.434$ ,  $\lambda_3 = 6.103$ .

The three components of the turbulence  $\mathbf{u}(\mathbf{x},t) = (u_1, u_2, u_3)$ , that are orthogonal to each other, are usually assumed statistically uncorrelated. Vice versa, the dependency of each pairs of parallel turbulence components in two distinct points of the space is described by the coherence function modelled as follows:

$$Coh_{u_{i}(\mathbf{x})u_{j}(\mathbf{x})}(\mathbf{x}, \mathbf{x}', n) = \\ = \exp\left[-\frac{n\sqrt{\left[2C_{j1}\left(x_{1}-x_{1}'\right)\right]^{2}+\left[2C_{j2}\left(x_{2}-x_{2}'\right)\right]^{2}+\left[2C_{j3}\left(x_{3}-x_{3}'\right)\right]^{2}}{\|\mathbf{U}(\mathbf{x})\|+\|\mathbf{U}(\mathbf{x}')\|}\right]}, \quad (j = 1, 2, 3),$$

$$(4.7)$$

where the coefficient  $C_{js}$  represents the exponential decay of the component  $u_j$  versus the longitudinal (s=1), lateral (s=2) and vertical (s=3) displacements respectively.

Once introduced the problem from a mathematical point of view, in order to obtain a simulated realization of the wind velocity field, one needs to discretize the spatial domain into k=1,2,3,...,N nodes which usually represent some significant points of the structure under study<sup>11</sup>. Assuming that  $\mathbf{x}^{(k)} = (x_1^{(k)}, x_2^{(k)}, x_3^{(k)})$  represent the position of the generic *k*-th node of the discretized space, (4.1) can be rewritten simply as:

$$\mathbf{V}^{k}\left(t\right) = \mathbf{V}^{k}\left(\mathbf{x}^{\left(k\right)}, t\right) = \mathbf{U}\left(\mathbf{x}^{\left(k\right)}\right) + \mathbf{u}\left(\mathbf{x}^{\left(k\right)}, t\right), \quad k = 1, 2, 3, ..., N,$$
(4.8)

where  $u_1^k, u_2^k, u_3^k$  are, once again, respectively the longitudinal, lateral and vertical components of turbulence in the *k*-th node of the simulation domain. Following the

<sup>&</sup>lt;sup>11</sup> Usually the same points corresponds to a subset of the nodes used in the numerical model (FEM).

above approach the time-dependent turbulence random field  $\mathbf{u}(\mathbf{x},t) = (u_1, u_2, u_3)$ , is transformed into a *3N*-variate stationary random process  $\underline{\mathbf{u}}(t)$ , where  $\underline{\mathbf{u}}(t)$  is a *3N*order vector containing the components  $u_1^{(k)}, u_2^{(k)}, u_3^{(k)}$  of the turbulence vector  $\mathbf{u}^{(k)}(t)$ .

The characterization of the  $\underline{\mathbf{u}}(t)$  process is complete introducing the power spectral density matrix (PSD) of order  $N \times N$  defined as:

$$S_{\underline{u}\underline{u}}(n) = \begin{pmatrix} S_{u^{(1)}u^{(1)}}(n) & \dots & S_{u^{(1)}u^{(N)}}(n) \\ \vdots & \ddots & \vdots \\ S_{u^{(N)}u^{(1)}}(n) & \dots & S_{u^{(N)}u^{(N)}}(n) \end{pmatrix},$$
(4.9)

where each component are defined by a matrix of order  $3 \times 3$ :

$$S_{u^{(h)}u^{(k)}}\left(n\right) = \begin{pmatrix} S_{u_{1}^{(h)}u_{1}^{(k)}}\left(n\right) & S_{u_{1}^{(h)}u_{2}^{(k)}}\left(n\right) & S_{u_{1}^{(h)}u_{3}^{(k)}}\left(n\right) \\ S_{u_{2}^{(h)}u_{1}^{(k)}}\left(n\right) & S_{u_{2}^{(h)}u_{2}^{(k)}}\left(n\right) & S_{u_{2}^{(h)}u_{3}^{(k)}}\left(n\right) \\ S_{u_{3}^{(h)}u_{1}^{(k)}}\left(n\right) & S_{u_{3}^{(h)}u_{2}^{(k)}}\left(n\right) & S_{u_{3}^{(h)}u_{3}^{(k)}}\left(n\right) \end{pmatrix}, \quad h, k = 1, 2, ..., N$$
(4.10)

The PSD in (4.9) is real<sup>12</sup>, symmetric and positive definite at each frequency value n. Introduce the two-side power spectral density matrix (2-DPSD) as follows:

$$G_{\underline{u}\underline{u}}(n) = \begin{cases} \frac{1}{2\pi} S_{\underline{u}\underline{u}}(n) \\ \frac{1}{2\pi} S_{\underline{u}\underline{u}}(n)^* \end{cases}, \tag{4.11}$$

where the operator \* indicates the complex conjugate.

Since  $\mathbf{u}(t)$  is *3N*-variate stationary random Gaussian process, it is possible to apply the Priestley spectral approach to generate its representation; that is:

$$\underline{\mathbf{u}}(t) = \int_{-\infty}^{+\infty} e^{i\omega t} d\mathbf{Z}(\omega), \qquad (4.12)$$

<sup>&</sup>lt;sup>12</sup> The PSD is real because the quadrature spectrum has been neglected.

where  $\mathbf{Z}(\omega)$  is a zero-mean normal complex random process in which  $d\mathbf{Z}(\omega) = \mathbf{Z}(\omega + d\omega) - \mathbf{Z}(\omega)$  that satisfies:

$$\mathbf{E} \left[ d\mathbf{Z}(\omega) \right] = 0 \quad \forall \omega$$
$$\mathbf{E} \left[ d\mathbf{Z}(\omega) d\mathbf{Z}^{*T}(\omega') \right] = \begin{cases} G_{\underline{u}\underline{u}}(\omega) d\omega & \text{if } \omega = \omega' \\ 0 & \text{otherwise} \end{cases}$$
(4.13)

Moreover, since the simulated process is a real-value process the Hermitian symmetry is guaranteed; that is  $d\mathbf{Z}(\omega) = d\mathbf{Z}^*(-\omega)$ .

Introducing now the vector  $\mathbf{w}(\omega)$  containing *3N* complex-value uncorrelated white noise with unit variance and Hermetian symmetry, and an *ad hoc* deterministic matrix  $\mathbf{T}(\omega) \mid G_{\underline{u}\underline{u}}(\omega) = \mathbf{T}(\omega) \cdot \mathbf{T}^*(\omega)^T$ , it is possible to rewrite the increment  $d\mathbf{Z}(\omega)$  as follows:

$$d\mathbf{Z}(\omega) = \mathbf{T}(\omega)\mathbf{w}(\omega)\sqrt{d\omega} . \qquad (4.14)$$

Substituting (4.14) into (4.12) it is possible to rewrite the random process  $\underline{\mathbf{u}}(t)$  as:

$$\underline{\mathbf{u}}(t) = \int_{-\infty}^{+\infty} e^{i\varpi t} \mathbf{T}(\omega) \mathbf{w}(\omega) \sqrt{d\omega}, \qquad (4.15)$$

It is convenient to apply the decomposition of the 2-DPSD matrix by the Cholesky factorization<sup>13</sup>:

$$\mathbf{T}(\omega) = \begin{pmatrix} T_{11}(\omega) & 0 & \dots & 0 \\ T_{21}(\omega) & T_{22}(\omega) & \dots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ T_{3N1}(\omega) & T_{3N2}(\omega) & \dots & T_{3N3N}(\omega) \end{pmatrix},$$
(4.16)

The discrete version of the equation (4.15) can be written as:

$$\underline{\mathbf{u}}(t) \cong \sum_{-\infty}^{+\infty} e^{i\omega_k t} \mathbf{T}(\omega_k) \mathbf{w}(\omega_k) \sqrt{\Delta\omega}, \qquad (4.17)$$

<sup>&</sup>lt;sup>13</sup> This approach is not unique. Different factorization/decomposition can be applied.

where the following sequence of uniformly spaced circular frequencies  $\omega_k$  with step  $\Delta \omega$  is considered:

$$\omega_{k} = \begin{cases} (k-1)\Delta\omega \ for \ k = 1, ..., \frac{N_{\omega}}{2} + 1\\ -\omega_{N\omega+2-k} \ for \ k = \frac{N_{\omega}}{2} + 2, ..., N_{\omega} \end{cases},$$
(4.18)

in which a cut-off circular frequency  $\omega_c = \frac{N_{\omega}}{2\Delta\omega}$  is introduced.

Introducing the decomposition proposed in (4.16) into equation (4.17), the *j*-th component of the process can be rewritten as:

$$\underline{u}_{j}(t) \cong \sum_{r=1}^{j} \sum_{k=1}^{+\infty} e^{i\omega_{k}t} T_{jr}(\omega_{k}) w_{r}(\omega_{k}) \sqrt{\Delta\omega}, \ j = 1, 2, ..., 3N$$

$$(4.19)$$

and it can be used for the simulation process by truncating the summation to the finite number  $N_{\omega}$  of harmonics. The entire simulation of the process  $\underline{\mathbf{u}}(t)$  is then generated along a sequence of  $N_t$  uniformly spaced time instant defined as:

$$t_j = (j-1)\Delta t, \text{ for } j = 1, ..., N_t.$$
 (4.20)

Equation (4.19) might be interpreted as a multidimensional Fourier series in which the fundamental circular frequency is represented by  $\Delta \omega$ , and the corresponding period is defined as  $T_p = 2\pi/\Delta \omega$ . The simulation represented in (4.19) may be interpreted as a process with periodicity  $T_p$ . Therefore, the simulation of the process requires to be satisfied the following condition about its time duration:

$$t_{sample} < T_P. \tag{4.21}$$

More in general, must be satisfied the following conditions:

$$\Delta \omega \leq \frac{2\pi}{N_{t} \Delta t}$$

$$\Delta t \leq \frac{\pi}{4\omega_{c}}$$
(4.22)

An alternative representation of the equation (4.19) is given in [5]; that is:

$$\underline{u}_{j}(t) \cong \sum_{r=1}^{j} \sum_{k=1}^{N\omega} T_{jr}(\omega_{k}) \cos(\omega_{k}' t + \phi_{rk}) \sqrt{\Delta\omega}, \ j = 1, 2, ..., 3N$$

$$(4.23)$$

where  $\omega'_k = \omega_k + \delta \omega_k$  with  $\delta \omega_k$  a small random frequency introduced to avoid the periodicity of the simulated signal, and  $\phi_{rk} \in [0, 2\pi]$  are random (independent) phases uniformly distributed.

In design applications the above fields are obtained using *ad hoc* algorithms (an example is represented in Figure 4.1.3). One can refer to [7] among others. Following this approach the only parameters that one needs to set in order to get the wind field simulation are:

- 1) mean velocity of the wind;
- 2) roughness length;
- 3) time step;
- 4) grid of nodes over which the simulation is performed.

Once simulated the wind velocity field, by the Bernoulli's formula, it is possible to evaluate the force field that can be used within the FEM in order to perform a structural analysis of the structure under study; that is:

$$F_{i}^{(k)}\left(\mathbf{x}^{(k)},t\right) = q_{i}^{(k)}\Omega = \left(\frac{1}{2}\rho\left[V_{i}^{(k)}\left(\mathbf{x}^{(k)},t\right)\right]^{2}\right)\Omega, \quad i = 1, 2, 3, ..., 3N,$$
(4.24)

where  $F_i^{(k)}(\mathbf{x}^{(k)},t)$  is the *i*-th component (expressed in *N*) of the force vector  $\mathbf{F}^{(k)}(\mathbf{x}^{(k)},t)$ ,  $V_i^{(k)}(\mathbf{x}^{(k)},t)$  is the wind velocity field simulated in (4.8),  $\rho = 1,25 \text{ kg} / m^3$  the air mass density, and  $\Omega$  (expressed in  $m^2$ ) the impact surface on the structure in which the wind is blowing.

Define: - "mean" wind velocity - roughness length	
- time step	
Ū	
Define:	
<ul> <li>plane of simulation</li> </ul>	
<ul> <li>points of interesting alo</li> </ul>	ng
the structure	
I.	
run WAWS algorithms	
wind velocity field simulate	be
<u> </u>	

Figure 4.1.3. Flowchart for the wind velocity field simulation.

### 4.2 Human induced vibration source

Despite several proposals can be found in the literature (and/or guidelines) about the modelling of the load induced by a single pedestrian, no reference seems to be addressed to the common case of groups of walking people.

In this section a model for introducing the crowd loads on a footbridge is discussed. The effects of the simulated realizations are compared with the ones recorded during an experimental campaign carried out on the in-service "Tesa" footbridge (see section 2.3 and section 3.1.1).

The classical dynamic behaviour of a single pedestrian is first considered on the basis of the existing literature (more details are presented in *Appendix B*). A first distinction has to be done about the different dynamic effect of walking and running. Indeed during pedestrian walking there is always one foot in contact with the ground, while during running this disappears and usually only one foot touches the ground in a discontinuous manner. Any person walks with similar step frequencies in the range from 1.25 to 2.3 Hz, due to similar physiological human constitutions, but the step frequencies are

influenced by the purpose of the movement and the intensity of the pedestrian loading [11].

The most common time domain models for these forces scheme can be found in literature and/or in the guidelines (see [11]-[14] among others). They are based on the assumption that both human feet produce the same force. The resulting periodic force is expressed by standard Fourier series as follows:

• vertical (gravity) component:

$$F_{\nu}(t) = P\left(1 + \sum_{n=1}^{k} \alpha_{n,\nu} \sin(2\pi n f_{\nu} t - \varphi_{n})\right);$$
(4.25)

• lateral (transversal) component:

$$F_{lat}(t) = P \sum_{n=1}^{k} \alpha_{n, \text{lat}} \sin(2\pi n f_{lat} t - \varphi_n); \qquad (4.26)$$

The model of the pedestrian running is more complicated because, during the run, there is a so called "lift-off phase", during which no foot is in contact with the ground. Only the vertical component of these forces is taken into consideration because it has been observed as the horizontal components, both lateral and longitudinal, have a low "impact" on the dynamic response of the bridge [14]. The vertical component is modelled by using the standard Fourier transform as follows:

$$F_{r,vert}(t) = P + \sum_{n=1}^{k} P_n \sin(2\pi n f_r t) \quad \text{for } (j-1)T_m \le t \le \left(j-\frac{1}{2}\right)T_m$$
  

$$F_{r,vert}(t) = 0 \qquad \qquad \text{for } \left(j-\frac{1}{2}\right)T_m < t \le jT_m$$
(4.27)

#### 4.2.1 Crowd Load Model (CLM)

In order to simulate the interaction between the structure and the human induced loads in a realistic manner, a model for the load due to a group of persons is required. In this section a numerical model of the crowd load is discussed. As already observed in section 2.3, the proposed model is validated on the base of the records acquired during the experimental campaign carried out on the "Tesa" footbridge.

A numerical model for the crowd load has to be account for several aspects: psychology, motion, individual feeling and so forth... The introduction of the global effects of these aspects by an amplification factor, as suggested in some guidelines (e.g. [14]), could result quite inaccurate.

The mains ideas behind the new model for the crowd load can be summarized as follows:

- the load is regarded as the sum of a "steady/mean" and a "fluctuation" part.
- the spectrum of the fluctuation part fits the one of data recorded during the experimental campaign.

Thus the pedestrian induced force is the sum of two components. Indeed the pedestrian can be seen, from the mechanic point of view, as a moving mass; thus one can identify a steady component related to its weight, and a fluctuation part related to the inertial component of its motion. In other words the fluctuation part takes into account that, during the motion, each pedestrian foot transmits to the structure different values of the force.

The model is referred to only two main directions: transversal axis (Y), and vertical or gravity axis (Z).

Without loss of generality, the procedure is presented with reference for the case of six persons walking along the deck in couples of two.

The first step, for the lead simulation, is to identify the *grid* of k=1,2,...,N nodes (Figure 4.2.1) where the generic node has coordinates  $P_k = (X_k; Y_k)$ .

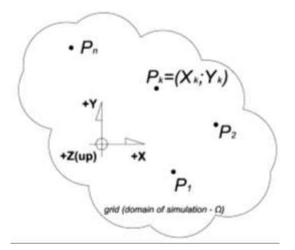


Figure 4.2.1. Grid of nodes for the simulation.

The main hypothesis on which the proposed model is based is the *ergodicity* of the process. Given this assumption, it is possible to define the Cross Power Spectral Density

Function (CPCD) in terms of auto-spectra  $S_{f_i f_j}$  and coherence function  $Coh_{f_i^{(h)}f^{(k)}_j}$  as follows:

$$\begin{bmatrix} S_{f_i f_j} \left( \mathbf{x}, \mathbf{x}'; n \right) = \sqrt{S_{f_i f_j} \left( \mathbf{x}; n \right) S_{f_i f_j} \left( \mathbf{x}'; n \right)} Coh_{f_i f_j} \left( \mathbf{x}, \mathbf{x}'; n \right) \\ Coh_{f_i^{(k)} f_j^{(k)}} \left( n \right) = \exp \left[ -\frac{n \left\| (C_j^{(h)} + C_j^{(k)}) (\mathbf{x}^{(h)} - \mathbf{x}^{(k)}) \right\|}{\left\| F^{(h)} \right\| + \left\| F^{(k)} \right\|} \right] \quad i, j = 1, 2.$$
(4.28)

where:

- $F^{(h)}$ ,  $F^{(k)}$ : are the stationary value of the forces (or mass) due to one pedestrian;
- $f_i, f_j$ : any pair forces (or mass) due to the movement of the crowd, simulated by the process, acting along the two main direction above described in any pair of position x, x';
- *n* : the frequency [Hz];
- $C_i^{(k)}$ : the second order tensor is defined as follows:

$$C_{j}^{(k)} = \sum_{s=1}^{3} C_{js} f_{s}^{(k)} \otimes f_{s}^{(k)}, \qquad (4.29)$$

where  $f_s^k$  is the *s*-th component of the local frame defined on the general node k=1,2,...,N (Figure 4.2.2), and the matrix of coefficient  $C_{js}$  is assumed equal to:

$$C = \begin{bmatrix} 0 & 0,3 & 0,05\\ 0 & 0 & 0\\ 0 & 0,65 & 0,3 \end{bmatrix}.$$
 (4.30)

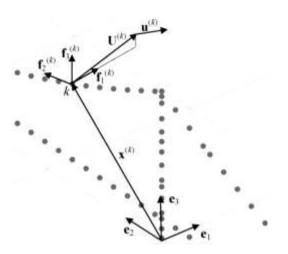


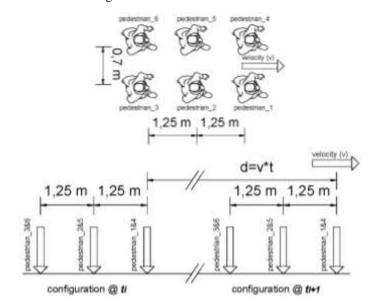
Figure 4.2.2. Local frame system for the *k*-th node.

Once a realization of the field is simulated by *ad hoc* code in MatLAB [15], a more realistic physical distribution of the above simulated forces is needed. Indeed, as suggested by the experience, the pedestrians during their motion will try to move along the centreline of the deck rather than close to the handrail of the footbridge. Moreover in a crowd situation, all the pedestrians tend to conform their footstep in accordance with the crowd velocity. Thus, a correction function  $f_{bd}(X,Y)$  is introduced, where (X,Y) are the planar coordinates of the *k*-th node of the simulation grid defined above (Figure 4.2.1). In other words the purpose of this function is to decrease the value of the simulated forces belonging to the nodes close to the external side of the pedestrian bridge, where is likely to assume that the pedestrian will avoid to walk, shifting the forces towards the centreline of the deck;

Summarizing, the field forces along Y and Z axes proposed by the above CLM model can be represented as follows:

$$F_{CROWD}(X,Y;t) = \left(F_{STEADY} + F_{FLUCTATING}(X,Y;t)\right) * f_{bd}(X,Y).$$
(4.31)

It has to be underlined as in this procedure the velocity of the crowd during the crossing of the footbridge is hypothesized constant along all the simulation; in other words the hypothesis of a *uniform linear motion* is assumed. Moreover, from a kinematic point of view it is assumed that each pedestrian will take the place of his/her predecessor (Figure 4.2.3). Furthermore the spatial configuration of the pedestrians is assumed the same along the simulation; that is the distance between two consecutive pedestrian in both the



longitudinal and transversal directions of the deck is fixed and correspond to the dimension of the simulation grid.

Figure 4.2.3. Spatial configuration of the crowd model implemented.

From another point of view the proposed model can be given in different way idealizing a group of pedestrian as a "rigid" moving body. The model of these loads leads, within the hypothesis of a deterministic approach, to a constant deterministic value at any time in the considered nodes. The loads is simply "translated" along the deck according to the classical relation "*space = velocity -by- time*" but its value ("mean" value) remain the same on time and space (Figure 4.2.4).

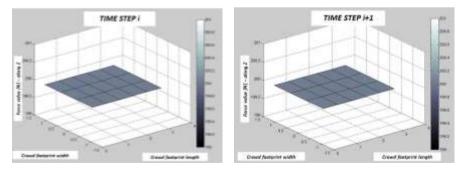


Figure 4.2.4. Classical model of moving loads. Deterministic approach. An example of two consecutive time steps.

In the model presented in this thesis, even if the *uniform linear motion* hypothesis is still adopted, a random process is assumed in order to give fluctuation around the "mean" to the components of the forces (the "fluctuating" part).

The above procedure for load modelling is be summarized by the flowchart represented in Figure 4.2.5.

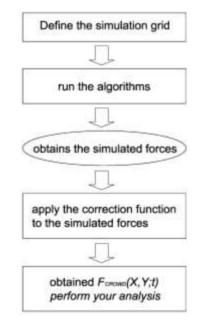


Figure 4.2.5. Flowchart for the CLM simulation.

To calibrate and validate the model the experimental campaign carried out on the "Tesa" footbridge is supported to provide the required information. Figure 4.2.6 and Table 4.2.1 summarize the main aspects of this campaign, already illustrated in section 3.1.1.

More details about the sets of data used for the calibration and validation of the proposed model are reported in *Appendix D*.

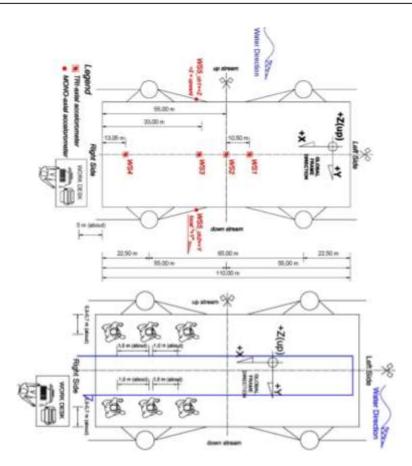


Figure 4.2.6. Above: experimental set-up; below: walking test of six person configuration.

Table 4.2.1. Walking (round trip) of six persons.

Туре	ID		
Day/Time	November 7,2013 / 3.00 pm		
Test type	walking (round trip)		
Air temperature	15°C		
Pedestrian(s) num.	6 persons		
Total Pedestrian(s) mass	480 Kg		
Total time duration	≈2:42 min		
Average velocity of the pedestrians	1.35 m/s		

# 4.3 Time-histories simulation

-

#### 4.3.1 Wind velocity field

Following the procedure for the wind velocity field simulation summarized in section 4.1 one has first to introduce the nodes of interest along the footbridge profile. With reference to the "Trasaghis" footbridge 23 nodes are selected as shown in the simulation grid in Figure 4.3.1. In particular fifteen nodes are equally spaced along the deck, while the remaining eight nodes are located along the pillars and the oblique steel stays (Figure 4.3.1 and Table 4.3.1).

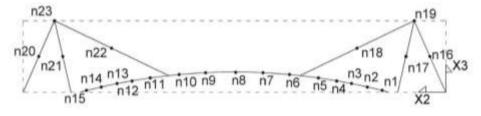


Figure 4.3.1. Grid of nodes for wind velocity field simulation.

Node ID	<i>x</i> <sub>2</sub> [m]	<i>x</i> 3 [m]	Node ID	<i>x</i> <sub>2</sub> [m]	<i>x</i> 3 [m]
1	12.47	1.79	12	77.07	2.67
2	18.33	2.08	13	82.94	2.38
3	24.20	2.38	14	88.81	2.08
4	30.07	2.67	15	94.67	1.79
5	35.95	2.96	16	4.63	9.39
6	41.82	3.25	17	10.52	9.64
7	47.70	3.45	18	24.81	9.75
8	53.57	3.52	19	9.26	16.28
9	59.44	3.45	20	102.51	9.39
10	65.32	3.25	21	96.62	9.64
11	71.19	2.96	22	82.34	9.75
			23	97.88	16.28

Table 4.3.1. Coordinates of the nodes for the simulation grid.

The parameters necessary to perform the simulation are defined below:

- shear velocity of the wind:  $u^* = 7$  m/s;
- Von Karman constant: k = 0.41;

- roughness length:  $z_0 = 0.30$  m according to [16];
- time step:  $\Delta t = 0.05$  sec as a compromise between achievable accuracy and required computational effort. The duration of the simulated time history is assigned to be 60 sec in view of the exemplification purpose of these analyses. Of course, in the actual design process, longer durations would be better recommended.

With the above data one achieves an average value of  $U(x_3)$  (see equation (4.2)) equal to about 27 m/s. An example of the simulated velocity fields for the nodes n1 and n19 belonging respectively to the deck and to the top of the pillar, are shown in Figure 4.3.2.

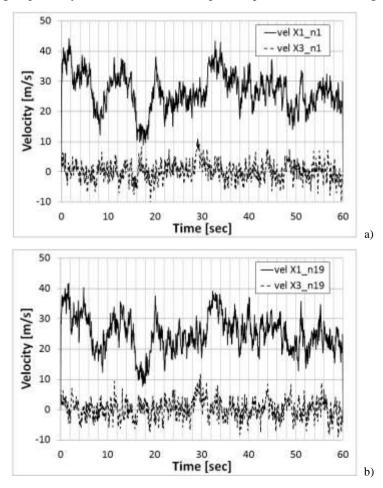


Figure 4.3.2. Wind velocity components along the  $x_1$  and  $x_3$  axes as simulated for node: a) n1, b) n19.

Once the wind velocity field is obtained, by applying the equation (4.24), the related force field for each node of the grid is calculated. When introducing the time histories as boundary conditions in the finite element analyses, one can either consider them as instantaneously applied (with consequent impact phenomena that can be create several problems during the dynamic analysis), or "smooth" them by applying an initial and a final linear ramp (to avoid the above impact phenomena). The second option is then adopted. As an example in Figure 4.3.3 the time histories of the forces for the nodes n8, n22 and n23 are plotted. For each node the two components along the  $x_1$  and  $x_3$  axes are plotted.

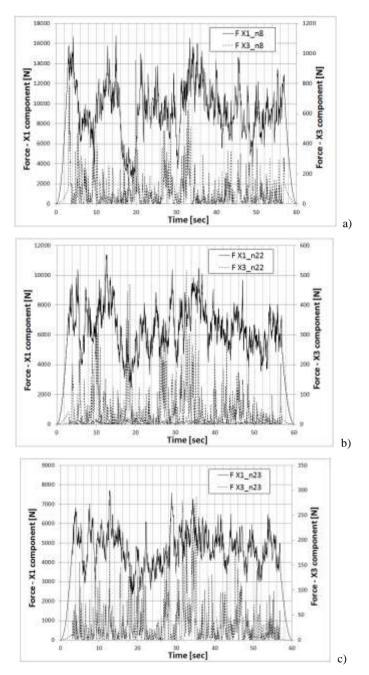


Figure 4.3.3. Force components along the  $x_1$  and  $x_3$  axes for node: a) n8, b) n22 and c) n23.

## 4.3.2 Crowd Load Model: numerical simulation and validation

Moving now to the simulation of the crowd load, following the procedure summarized in Figure 4.2.5, the first step is to define the grid over which the algorithm will be applied. For the case study (i.e. "Tesa" footbridge) the grid is made by 24 nodes subdivided in 4 rows (parallel to the longitudinal direction of the deck) and 6 columns (parallel to the transversal direction of the deck) for a global dimension of 2.5 by 3.125 m, in order to be consistent with the mesh of the "Tesa" footbridge deck (Figure 4.3.4). This means that each pedestrian insist, at any instant of the time, on four nodes to represent the possibility of the pedestrian to move "freely" during the motion. The step of the nodes is assumed equal to 0.625 m, in order to simulate an average human footstep. In other words the velocity of the crowd is assumed equal to v = 0.625 m/s; i.e. 1 footstep/sec.

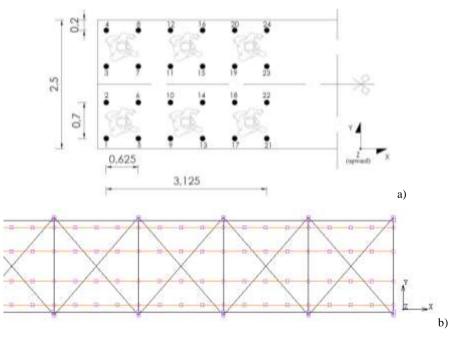


Figure 4.3.4. a) Grid of nodes for the simulation; b) example of the deck nodal grid implemented in MARC Mentat2010.

The steady value of the force (or mass) due to a singular pedestrian is assumed equal to  $F_{pedestrain}=800$  N/pedestrian (about 80 Kg/pedestrian). In other words this means that on each nodes of the grid the applied force is equal to  $F^{(k)}=200$  N/nodes with k=1,2,...,24 (Figure 4.3.5).

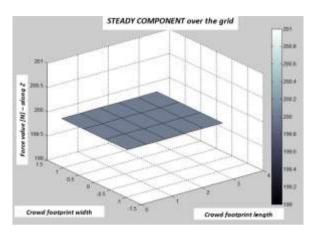


Figure 4.3.5. Steady component of the force over the simulation grid.

The correction function is defined as follows:

$$f_{bd}(X,Y) = \left(-0.82Y^2 + 1.50\right) \left( 2 \left( 0.30 \cos\left(2 \left(\frac{\pi}{l_{grid}/2,5}\right) X\right) + 0.50\right) \right), \quad (4.32)$$

where (X, Y) are the planar coordinates of the *k*-th node of the grid, and  $l_{grid} = 3.125$  m the length of the grid. In Figure 4.3.6 the 3-D plot of the above function is shown.

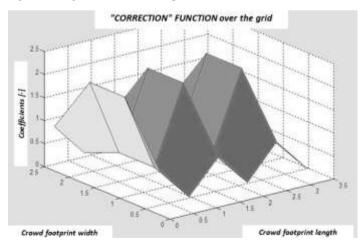


Figure 4.3.6. 3-D plot of the "barrel dented" function over the nodes of the simulation grid.

As an example, from Figure 4.3.7 to Figure 4.3.9, the plots of the simulated forces at three different time steps are represented.

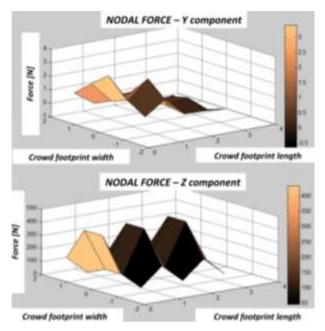


Figure 4.3.7. Step 1 – simulated force files by the proposed CLM model.

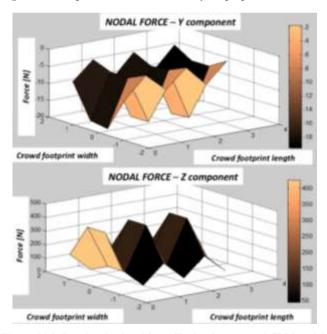


Figure 4.3.8. Step 2 – simulated force files by the proposed CLM model.

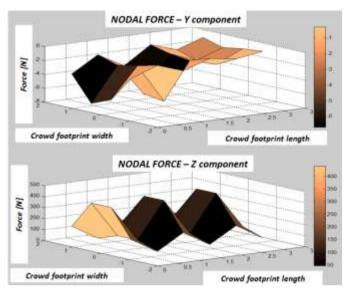


Figure 4.3.9. Step 3 – simulated force files by the proposed CLM model.

Once obtained the above field of forces, in order to evaluate the performance of the proposed CLM model, a dynamic transient analysis is performed based on the numerical model of the foorbridge developed in section 3.3. The obtained forces have been implemented as nodal forces within the FEM (Figure 4.3.10), acting along both Y and Z axes in accordance with the CLM model.

Once performed the dynamic transient analysis the nodal accelerations are extrapolated from the results and analyzed following the Short Time Fast Fourier (STFFT) algorithm (see *Appendix D*). In particular the numerical accelerations of the node of the FE model corresponding to the position along the deck of the accelerometer labelled as WSa2 (see Figure 3.1.3) are used.

In Figure 4.3.11 and Figure 4.3.12 the comparisons between the results acquired in situ and the ones obtained from the numerical simulation along Y and Z axes are presented respectively.

It is worth underlining that, for the purpose of this section, the data acquired in situ along Z axis, have been firstly filtered.

More details about the set of data used for the comparison in reported in Appendix D.

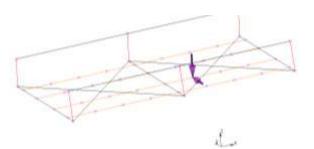


Figure 4.3.10. Example of the simulated forces implemented in MARC Mentat2010.

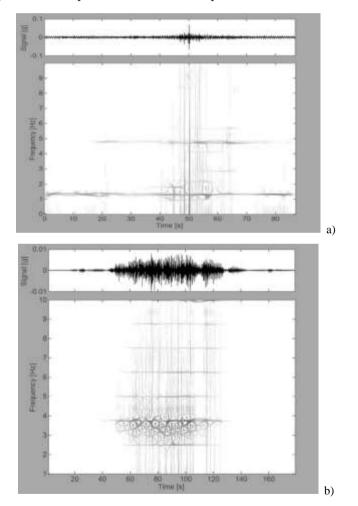


Figure 4.3.11. Time-frequency plots from the data taken by sensor WSa2 along axis Y (transversal). a) data acquired in situ; b) data from the numerical simulation.

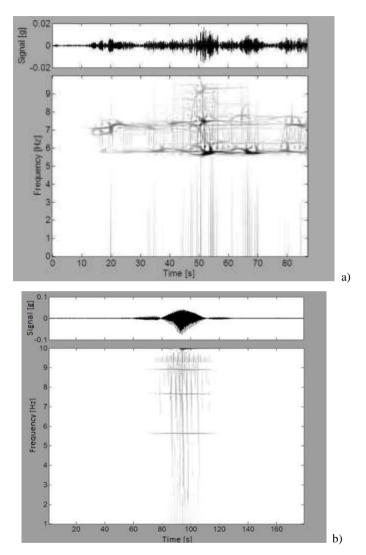


Figure 4.3.12. Time-frequency plots from the data taken by sensor WSa2 along axis Z (gravity) a) data acquired in situ and filtered at 2.50 Hz; b) data from the numerical simulation.

The analysis of the results shows as the proposed CLM model is able to capture the main features of the interaction between the pedestrians and the bridge in terms of both range of frequencies and time activation of a particular frequency. About this aspect has to be stressed as the assumption of the velocity of the pedestrian during the simulation about half of that one recorded during the test on field (in order to get equivalence between the simulated footstep and the node of the mesh), "moves" the response of the

model in terms of time activation of the frequencies. By the way this suggests to improve the model in such a way to better represent a more detailed distribution of the pedestrians velocity during the crossing.

The proposed model seems to represent the observed records in a proper manner. Anyway further studies and experimental campaigns are needed to confirm and to improve the proposed CLM model.

### 4.4 References

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# Chapter 5 Model order reduction (MOR)

Numerical simulations are nowadays in almost all fields of the sciences an important tool to predict the behaviour of the process under study by-passing the direct experimentation on large scale (i.e. laboratory test, field campaign, and so forth...) which are usually quite expensive.

Accurate simulations imply that the errors of the virtual models be small, and that different aspects of the process taken into account.

Although the calculation power is greatly improved over the last decades, in the field of structural engineering, many time one needs to work with dynamic models that may contain a large number of equations. This makes the analysis heavy in terms of calculation time. For this reason one needs to simplify these models in order to achieve lighter models of analysis, by preserving a good approximation of the final results. This means that if one wants to simplify a given problem it has to be sure to represent all the main features of the original problem in order to avoid losses of important information.

The Model Order Reduction (MOR) is the label by which such a problem is denoted. The fundamental methods in the area of Model Order Reduction were published in the eighties and nineties of the last century. In 1981 Moore [1] published the method of Truncated Balanced Realization, in 1984 Glover published his famous paper on the Hankel-norm reduction [2], that are still nowadays the "foundations" of this method. More details will be given in the followings, but in Figure 5.1 the working concept of MOR is introduced in a graphical manner. From this figure it is easy-to-understand how sometimes very little information is required to describe a model. This simple example shown as, even with only a few facets, the rabbit on the right can still be recognized as such.

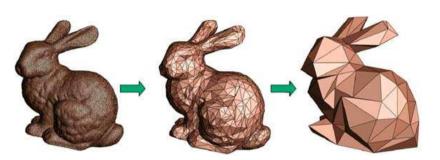


Figure 5.1. MOR: graphical representation [4].

Within the MOR techniques, current research studies are focused on the possibility to "capture" lower order models for nonlinear systems (see [3] among others). Nevertheless, also the correct application of MOR when dealing with linear systems still requires attentions and this is the topic of this chapter.

The MOR technique based on balanced transformation is first described from the mathematical point of view and then applied to the case study. It is worth noting there is no attempt to give a literature "review" in the field of MOR, but one just introduces the applied procedure.

## 5.1 MOR background

Up to 10-15 years ago, MOR techniques [4], were strictly required in order to solve, due to the low computation hardware capability available, large-size dynamic structural problems. Today the MOR strategy offers a tool that include both low computational costs, and a viable methodology toward the implementation of real time control systems useful for the solution of problems where several repeated analyses must be carried out (i.e. optimization and/or reliability problems [5]).

Within the field of structural engineering, the adopted mathematical models have the classical partial differential equations form, whose spatial derivatives are easily removed, by a finite element process. For a standard dynamic system the problem is therefore rearranged in terms of ordinary differential equations<sup>14</sup> where the second derivative with respect to time appears. In order to introduce the MOR technique for a linear dynamic system, one re-formulates the problem in the **state space**:

<sup>&</sup>lt;sup>14</sup> The number of equations represents the order of the model, say N.

$$\frac{d\boldsymbol{x}(t)}{dt} = \boldsymbol{A}\boldsymbol{x}(t) + \boldsymbol{B}\boldsymbol{u}(t)$$
(5.1)

$$\mathbf{y}(t) = C\mathbf{x}(t) + D\mathbf{u}(t) \tag{5.2}$$

where u is the input vector, related so to an external excitations (size p), x is the vector state variables<sup>15</sup> of size n and y the response vector of size q, Moreover A is a square matrix of size n by n, B a matrix of size n by p, C a matrix of size q by n and D a matrix of size q by p.

Assuming now that the response of the system depends only on the state of the system, it is possible to neglect the matrix **D**. Introducing the transformation  $z(t) = Tx(t) | T \in \mathbb{R}^{qwn}$  where **T** is an invertible matrix but not necessarily the unit matrix, and taking into mind once said above, one has<sup>16</sup>:

$$\dot{\boldsymbol{z}} = \boldsymbol{T}\boldsymbol{A}\boldsymbol{T}^{-1}\boldsymbol{z} + \boldsymbol{T}\boldsymbol{B}\boldsymbol{u} \tag{5.3}$$

$$\mathbf{y} = \boldsymbol{C}\boldsymbol{T}^{-1}\boldsymbol{z} \tag{5.4}$$

where, as usual, the superposed dot denotes time derivative.

The following steps regard the concepts of reachability and controllability.

A given system is completely reachable if it is possible to determine an *ad hoc* input signal able to lead, within a finite time interval, the system from the zero state to a predefined state. In other words the reachability represents the measure in which the state of the system can be manipulated by the input u(t). This concept regards only the state equations [6], so the matrix C and D can be neglected. From a mathematical point of view one introduces the matrix of reachability  $Q_R$  as:

$$\boldsymbol{Q}_{R} = \begin{bmatrix} \boldsymbol{B} & \boldsymbol{A}\boldsymbol{B} & \boldsymbol{A}^{2}\boldsymbol{B}...\boldsymbol{A}^{n-1}\boldsymbol{B} \end{bmatrix}^{T}.$$
(5.5)

The system is defined as fully reachable if  $rank(Q_R) = n$ .

In a similar way a given system is controllable if for each possible initial state of the system is possible to introduce an *ad hoc* input signal leads the system to the zero state in a finite time interval.

<sup>&</sup>lt;sup>15</sup> The state variables are not supposed to have any physical meaning; they have only mathematical quantities.

<sup>&</sup>lt;sup>16</sup> The dependence over the time is suppressed for sake of simplification.

The matrix of **observability** of the system can be defined as:

$$\boldsymbol{Q}_{O} = \begin{bmatrix} \boldsymbol{C} & \boldsymbol{C} \boldsymbol{A} & \boldsymbol{C} \boldsymbol{A}^{2} \dots \boldsymbol{C} \boldsymbol{A}^{n-1} \end{bmatrix}^{T}.$$
(5.6)

The system is then defined as fully observable if  $rank(Q_o) = n$ .

The next step is to introduce the concept of **Gramian** [4]. The Gramians are matrices that contain information about the proprieties of the system, let it to be W:

$$\boldsymbol{W} = \int_{t_1}^{t_2} \boldsymbol{G}(\tau) \boldsymbol{G}^T(\tau) d\tau \,. \tag{5.7}$$

In the matrix  $G(\tau)$  the columns represent each single response signal due to the generic input signal. It is possible to demonstrate that the above matrix W is positive semi-definite with eigenvalues  $\lambda_1^2 \ge \lambda_2^2 \ge \cdots \ge \lambda_n^2 \ge 0$  and the corresponding eigenvectors  $v_1, v_2, \cdots, v_n$  mutually orthogonal forming a base of orthonormal vectors. Following [1] it is possible to re-write the matrix of the generic Gramian as:

$$\boldsymbol{W} = \int_{t_1}^{t_2} \boldsymbol{G}(\tau) \boldsymbol{G}^T(\tau) d\tau = \boldsymbol{V} \boldsymbol{\Lambda}^2 \boldsymbol{V}^T, \qquad (5.8)$$

where  $V = (v_1, v_2, \dots, v_n)$  is the matrix of eigenvectors, and  $\Lambda$  the diagonal matrix.

Then introduces the following quantities [1]:

$$\boldsymbol{W}_{C} = \int_{0}^{\infty} \boldsymbol{e}^{\tau A} \boldsymbol{B} \boldsymbol{B}^{T} \boldsymbol{e}^{\tau A^{T}} d\tau, \qquad (5.9)$$

$$\boldsymbol{W}_{O} = \int_{0}^{\infty} \boldsymbol{e}^{\tau A} \boldsymbol{C} \boldsymbol{C}^{T} \boldsymbol{e}^{\tau A^{T}} d\tau , \qquad (5.10)$$

respectively the Gramian matrix of controllability and observability. These matrices satisfy the air of Lyapunov equations [6]-[10]:

$$AW_c + W_c A^T + BB^T = 0, (5.11)$$

$$\boldsymbol{A}^{T}\boldsymbol{W}_{\boldsymbol{o}} + \boldsymbol{A}\boldsymbol{W}_{\boldsymbol{o}} + \boldsymbol{C}^{T}\boldsymbol{C} = \boldsymbol{0}.$$
 (5.12)

Finally the balanced transformation to perform the reduction is achieved by:

- 1. solving the Lyapunov equations (5.11) and (5.12) find the Gramian matrices of controllability (5.9) and observability (5.10);
- 2. perform the Cholesky factorizations of the Gramian matrices:

$$\begin{bmatrix} \boldsymbol{W}_{C} = \boldsymbol{L}_{C} \boldsymbol{L}_{C}^{T} \\ \boldsymbol{W}_{O} = \boldsymbol{L}_{O} \boldsymbol{L}_{O}^{T} \end{bmatrix}$$
(5.13)

3. evaluate the Singular Values Decomposition (SVD) of the Cholesky factors:

$$\boldsymbol{L}_{O}^{T}\boldsymbol{L}_{C}=\boldsymbol{U}\boldsymbol{\Lambda}\boldsymbol{V}^{T}.$$
(5.14)

where U is the matrix of the eigenvectors of the matrix in the r.h.s., say Q, by its transpose, V is the matrix of the eigenvectors of  $Q^TQ$  and  $\Lambda$  is the diagonal matrix of the singular values.

4. introduce the balanced transformation [6]:

$$T = L_C V \Lambda^{-1/2}$$
  

$$T^{-1} = \Lambda^{-1/2} U^T L_O^T.$$
(5.15)

such that  $T^{-1}W_CT^{-T} = T^TW_OT = \Lambda$ . Evaluate the space of the balanced matrix as:

$$A_{b} = \mathbf{T}^{-1}A\mathbf{T} = \Lambda^{-1/2} \boldsymbol{U}^{T} \boldsymbol{L}_{O}^{T} \boldsymbol{A} \boldsymbol{L}_{C} \boldsymbol{V} \Lambda^{-1/2}$$
$$\boldsymbol{B}_{b} = \mathbf{T}^{-1} \boldsymbol{B} = \Lambda^{-1/2} \boldsymbol{U}^{T} \boldsymbol{L}_{O}^{T} \boldsymbol{B} \qquad . \tag{5.16}$$
$$\boldsymbol{C}_{b} = \boldsymbol{C} \mathbf{T} = \boldsymbol{C} \boldsymbol{L}_{C} \boldsymbol{V} \Lambda^{-1/2}$$

To operate a model reduction the Hankel singular values are then introduced.

A given system is considered as fully balanced within the time interval [0,T] if:

$$W_c = W_o = \Lambda^2 \,, \tag{5.17}$$

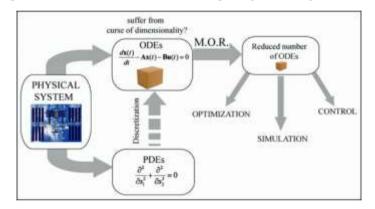
where:

$$W_{c} = W_{o} = \Lambda^{2} = \begin{bmatrix} h_{1} & 0 & \cdots & 0 \\ 0 & h_{2} & \cdots & \vdots \\ \vdots & \vdots & \ddots & 0 \\ 0 & \cdots & 0 & h_{n} \end{bmatrix},$$
(5.18)

is a diagonal matrix in which  $h_1 \ge h_2 \ge \cdots \ge h_n \ge 0$  are the Hankel's singular values. Following [7]-[9] these values can be evaluated by:

$$h_1 = \sqrt{\lambda_i \left( \mathbf{W}_C \mathbf{W}_O \right)} \,. \tag{5.19}$$

Concluding and generalizing the problem, let's consider a generic balanced model described by the matrices  $A_b$ ,  $B_b$ ,  $C_b$  and  $D_b$  with the corresponding Hankel's singular values sorted in descending order. Choose an appropriate truncation r such that  $h_r$  is strictly greater than  $h_{r+1}$  and define the state vector  $x = \{x_1, x_2\}$  where  $x_1$  corresponds to the first r component, while  $x_2$  the remaining n-r components.



A visual representation of the above MOR technique is given in Figure 5.1.1 [11].

Figure 5.1.1. MOR: visual representation [11].

The concept behind this reduction technique is that the singular values of the corresponding Hankel matrix tend to the second order modes  $\lambda_i$ . Because the number of non-zero singular values of the Hankel matrix determines the order of the system, the reduction consists in neglecting the  $\lambda_{k+1}$  smaller than a fixed value  $\lambda_k$ ; This operation is called truncation.

The balance of the system, instead, is the core of the optimization for the order reduction because the reduced model must retain the properties of controllability and observability. Thanks to the balancing the values are arranged in descending order that give a measure of the controllability and observability of the model. Once balanced, one can proceed with the truncation of the states size in order to exclude the smallest Hankel's singular values to achieve the approximate lower model order.

This truncation technique is called *balanced truncation* and consists in the reduction of the model for elimination of the system's vibration modes less significant. By neglecting the smallest Hankel's values, in other words, one deletes the states that, with difficulty, will be reached and observed by the system.

If are pushed too much the reductions, or if one decides to ignore a large number of  $\lambda$ , there is the risk of losing important information about the model (i.e. the structure under study) analyzed. The result is that the response of the system does not correspond to the results calculated by an exact solution of the model. For this reason it is important make a right decision about the appropriate value of  $\lambda$  from where to truncate the model without compromising the goodness of the final results.

### 5.2 Structural response

In this section the procedure described in the previous section is applied to the "Trasaghis" footbridge. In order to apply the MOR to the case study a "lighter" numerical model compared with the "full" one described in section 3.2.2, is developed within the MARC Mentat2010 environment [12]. Only *beam* elements are employed as reported in Figure 5.2.1. Also for this model great care is taken for the implementation of the so called non-structural mass as described in section 3.2.2 (e.g. larch cladding, handrail and so forth...) and for the kinematic boundary condition in order to achieve the best representation.

The new model is characterized by 301 beam elements and 208 nodes, and it is able to represent, for the dynamic point of view, the full FEM. A dynamic modal comparison between the "full" and the "light" FEM of the "Trasaghis" footbridge is reported from Figure 5.2.2 to Figure 5.2.6. Some discrepancies appear in terms of frequencies as reported in Table 5.2.1. They could have been reduced by a suitable refinement but this is out from the exemplification nature of the computational reported.

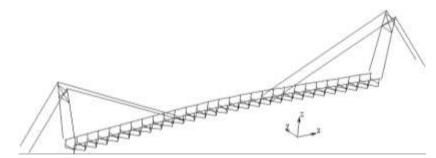


Figure 5.2.1. "Light" FEM implemented in MARC Mentat2010 environment.

ID	$\varepsilon = \left  \frac{f_{full} - f_{light}}{f_{light}} \right  x100$	
mode I	13.1 %	
mode II	5.7 %	
mode III	22.4 %	
mode IV	33.9 %	
mode V	13.0 %	



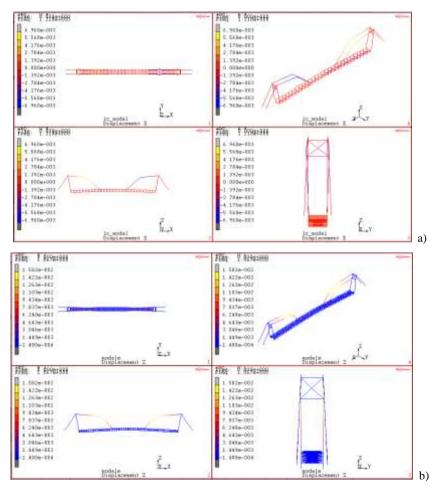


Figure 5.2.2. Dynamic modal comparison of the "Trasaghis" footbridge FEM: mode I. a) "Light" FEM f = 1.22 Hz; b) "Full" FEM f = 1.06 Hz.

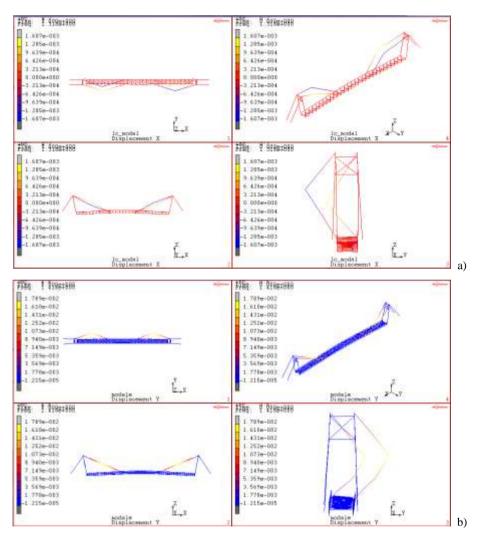


Figure 5.2.3. Dynamic modal comparison of the "Trasaghis" footbridge FEM: mode II. a) "Light" FEM f = 1.32 Hz; b) "Full" FEM f = 1.41 Hz.

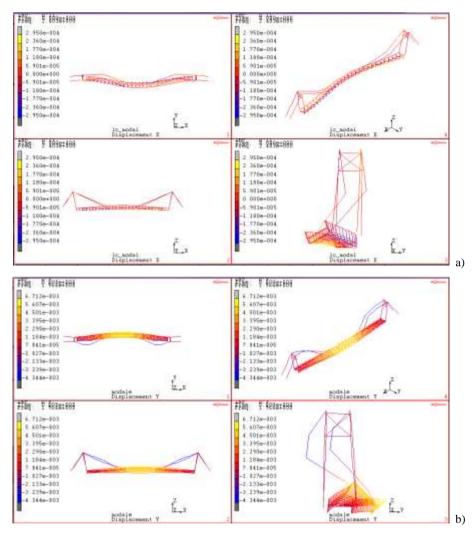


Figure 5.2.4. Dynamic modal comparison of the "Trasaghis" footbridge FEM: mode III. a) "Light" FEM f = 2.40 Hz; b) "Full" FEM f = 1.96 Hz.

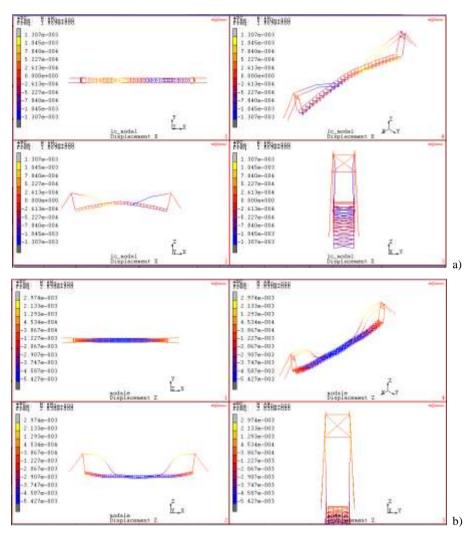


Figure 5.2.5. Dynamic modal comparison of the "Trasaghis" footbridge FEM: mode IV. a) "Light" FEM f = 1.87 Hz; b) "Full" FEM f = 2.83 Hz.

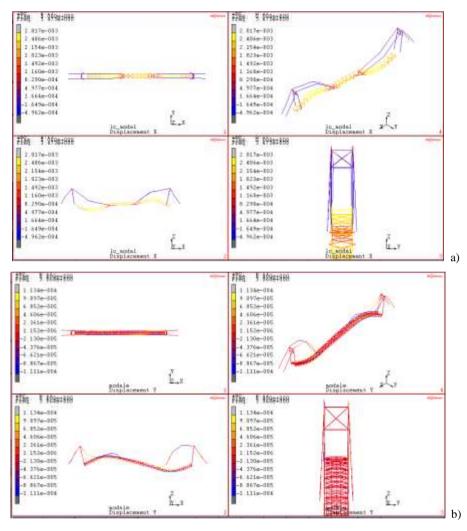


Figure 5.2.6. Dynamic modal comparison of the "Trasaghis" footbridge FEM: mode V. a) "Light" FEM f = 3.47 Hz; b) "Full" FEM f = 3.07 Hz.

Once defined the "light" FEM, a *Model Order Reduction* (MOR) methodology is applied and developed within MatLAB environment [13]. In particular three stable sets of modes are identified of size *145 states*, *114 states* and *84 states*, respectively, as shown in Figure 5.2.7 [14].

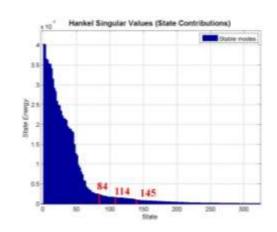


Figure 5.2.7. Hankel Singular Values - State Contributions.

To test the MOR method a dynamic transient analysis under wind loads is performed [15]. Following the simulation procedure proposed in section 4.1 one has to introduce the nodes of interest along the footbridge profile. With reference to the "light" FE Model of the "Trasaghis" footbridge 11 nodes are selected as shown in the simulation grid here below. In particular five nodes are equally spaced along the deck, while the remaining six nodes are located two at the top of the pillars and four along the oblique steel stays (Figure 5.2.8). The others parameters necessary to perform the simulation are:

- shear velocity of the wind:  $u^* = 7$  m/s;
- Von Karman constant: k = 0.41;
- roughness length:  $z_0 = 0.30$  m according to [16];
- time step:  $\Delta t = 0.01$  sec to achieve a good compromise between accuracy and required computational effort. The total duration of the simulated time history is assigned to be 60 sec, of which 30 sec are related to the "wind action" while the remaining 30 sec are related to the "structural rest" to capture the bridge response once the wind stops.



Figure 5.2.8. Gird of nodes for the "light" numerical model of the "Trasaghis" footbridge.

It is worth underlining that in this particular case only the fluctuating part of the wind velocity is considered. Once simulated the wind velocity field, the related force field is evaluated by the Bernoulli's formula. After that these forces are then implemented within the numerical model. As an example in Figure 5.2.9 and Figure 5.2.10 are represented respectively the time histories of the wind velocity and the related wind forces for the node of the grid located in the middle of the span (*node 5*).

Then a dynamic transient analysis is performed: total analysis time 60 sec - time step = 0.01 sec - total steps = 6000; i.e. twice the duration of the wind velocity time history as observed above.

The dynamic response of the numerical model in terms of displacements and accelerations of the node located in the middle of the steel stay (*node 3*) is plotted in Figure 5.2.11 and Figure 5.2.12.

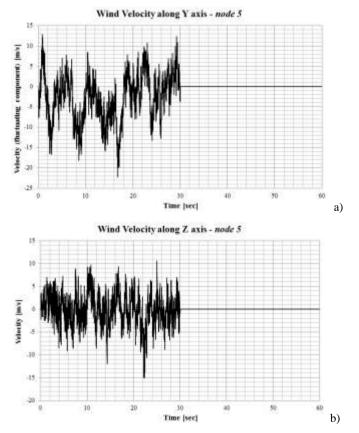


Figure 5.2.9. Fluctuating part of the simulated wind velocity for the *node* 5. a) Y component; b) Z component.

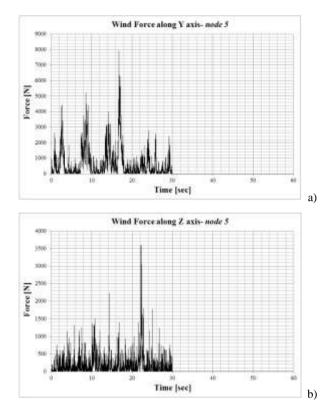


Figure 5.2.10. Wind forces for the *node* 5. a) Y component; b) Z component.

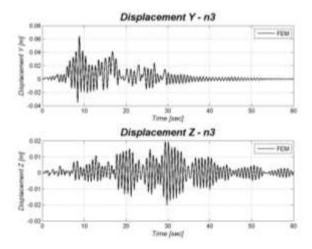


Figure 5.2.11. Dynamic response of the "light" FEM under the simulated wind loads – node 3. Displacement along Y and Z axes.

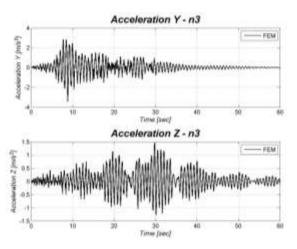


Figure 5.2.12. Dynamic response of the "light" FEM under the simulated wind loads – node 3. Accelerations along Y and Z axes.

## 5.3 MOR accuracy

The MOR method allows one to obtain a lighter model in terms of computational costs, compared with FEM, but able to catch all the main features in the dynamic response of the pedestrian bridge.

The advantage in terms of "time consuming" of the MOR technique for the analysis carried out as described in the previous section, are summarized in Table 5.3.1. It is worth underlining that all the calculations are performed with a *AMD* A4-4355m *Processor* – *RAM* 4.00 *Gb*.

Model	Time Consuming	Code
FE Model	660.00 sec	MARC Mentat2010
LIGHTER model	about 4 days	MATLab
MOR 145 states	672.65 sec	MATLab
MOR 114 states	287.15 sec	MATLab
MOR 84 states	274.14 sec	MATLab

Table 5.3.1. Time consuming

To verify the accuracy of the above procedure, a comparison in terms of the time histories of the displacements obtained by the "light" FEM and the ones from the three

obtained MOR models are presented here below. In particular, without loos of generality, the results are presented in terms of *Y* (transversal) and *Z* (vertical-gravity) axe displacement of the *node* 5 (Figure 5.3.2 and Figure 5.3.2).

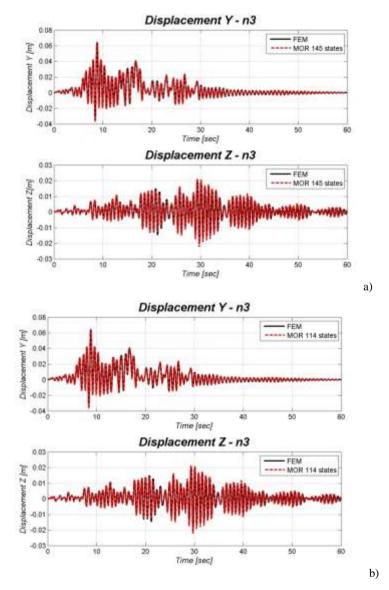


Figure 5.3.1. Comparison among "light" FEM and the implemented MOR models in terms of displacements along Y and Z axes – *node 3*. a) MOR 145 states; b) MOR 114 states.

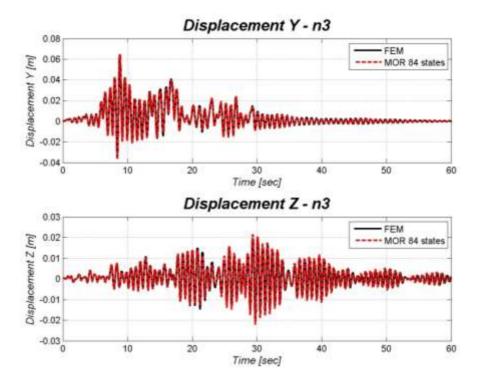


Figure 5.3.2. Comparison among "light" FEM and the implemented MOR models in terms of displacements along Y and Z axes – *node 3* - MOR 84 states.

As one can observe from the above plot, the response labelled as "MOR 145 states" well fits with the response achieved by the full FE Model of the footbridge. Moving to schemes including a lower value of states (i.e. MOR 114 and 84 states) some small discrepancies appear, even if compensated by a significant decrease of the calculation time.

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## **Chapter 6** Control solutions

Vibration control is a branch of the structural mechanic that study the possibility of mitigate the effects of an "external" input acting on a generic structure (i.e. bridges, buildings and so forth...) by using an *ad hoc* set of technical devices [1].

The vibration control devices are classified into three different sets:

- 1) *passive control* devices have no feedback capability between them, structural elements and the ground;
- 2) *active control* devices incorporate real-time recording instrumentation, input processing equipment and actuators within the structure;
- 3) *hybrid control* devices have combined features of active and passive control systems.

The distinction between passive and active control systems [2] is represented in Figure 6.1 and Figure 6.2.

A passive control system (e.g. energy dissipation system or a dynamic vibration absorber) develops motion control forces at the points of attachment of the system (see [3]-[5] among others). The motion of the points at which these devices are hanged produces the power needed to generate the proper control forces; while the relative motion of these points determine the amplitude and direction of the control forces.

An active control system also develops motion control forces; however the magnitude and direction of these forces are determined by a controller based on the information recorded by sensors throughout a control strategy, i.e. a proper algorithm. These forces are supplied by the active control system. In this solution the forces have to be fully powered by an *ad hoc* "external" power source. The related power, depending on the application and on the structure under study, may be large and require a lot of energy supply. In general an active control system should provide for better or more versatile response control.

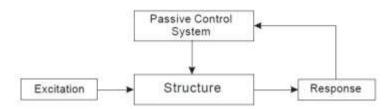


Figure 6.1. Scheme of a Passive Control System [2].

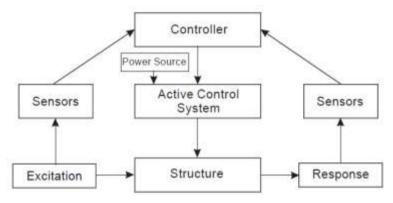


Figure 6.2. Scheme of an Active Control System [2].

Semi-active control systems (see [6] among others) may be deducted starting from a passive control systems which have been modified to allow for adjustment of their mechanical properties [2]. The mechanical properties of semi-active control systems may be adjusted by a controller as for the active control system in Figure 6.2. Nevertheless has to be stressed as in this case the control forces are developed as a result of motion of the structure points at which the semi-active devices (per passive control systems) are attached. Semi-active control systems usually require a low demand of power for the adjustment of the mechanical properties of the system that can be related to the power output of the system.

Different approaches can be used to control a structure in order to smooth the effects of the external actions and to improve the structures' performance. In particular:

- to dissipate the wave energy with properly engineered dampers mounted inside the structure;
- to disperse the wave energy between a wider range of frequencies;
- to absorb the resonant portions of the whole wave frequencies band with the help of so-called mass dampers.

Within the last kind of the above devices one has: the so called TMD (tuned mass dampers – see [4] among others) for the passive control approach, the AMD (active mass dampers) for the active one, and the HMD (hybrid mass dampers) for the hybrid control.

The following control solutions are introduced with reference of the "Trasaghis" footbridge (see section 3.3) for the mitigation of the wind induced vibrations.

#### 6.1 Passive control solutions

The finite element discretization leads one to write the standard dynamics equations in a matrix form (see section 2.1):

$$\mathbf{M}\ddot{\mathbf{x}}(t) + \mathbf{C}\dot{\mathbf{x}}(t) + \mathbf{K}\mathbf{x}(t) = \mathbf{f}(t), \qquad (6.1)$$

where, **M**, **C** and **K** are the mass, damping and stiffness matrices respectively, **x** are the node generalized displacements, **f** are the nodal forces, *t* denotes the time and a "dot" the time derivative as usual.

The governing relations with tuned mass dampers add forces at given locations [2] as follows:

$$\mathbf{M}\ddot{\mathbf{x}}(t) + \mathbf{C}\dot{\mathbf{x}}(t) + \mathbf{K}\mathbf{x}(t) = \mathbf{f}(t) + \mathbf{T}[\mathbf{c}\dot{\mathbf{z}}(t) + \mathbf{k}\mathbf{z}(t)], \qquad (6.2)$$

where **T** is a topological matrix while the matrices **c** and **k** contain, for each TMD, the damping coefficient *c* and the stiffness *k*. The variable z is the relative displacement of the added mass of the single TMD at the anchorage point, so that:

$$m\ddot{z}(t) + c\dot{z}(t) + kz(t) = -m\ddot{x}(t), \qquad (6.3)$$

where m is the added mass and is the displacement of the hanging point in the direction of the TMD degree of freedom z.

The standard design of a single TMD device requires the tuning of the frequency and the added mass, or better the ratio  $\mu$  between this secondary mass and the primary one, for a *1*-DOF system. The damping and stiffness coefficients, *c* and *k* in equation (6.3), depend on the ratio  $\mu$  as reported in [2], [7] and [8]-[11] among others. A first extension to MDOF's primary systems, see [2], was developed for the design of TMD devices in frames with lumped masses at each stories.

Let's consider the case of a *N*-DOF damped structure. If a single TMD is installed in correspondence to the *j*-th DOF, the equations of motion can be written as:

$$\mathbf{M}\ddot{\mathbf{x}}(t) + \mathbf{C}\dot{\mathbf{x}}(t) + \mathbf{K}\mathbf{x}(t) = \mathbf{f}(t) + d_j \left[ c_j \dot{z}(t) + k_j z(t) \right], \qquad (6.4)$$

where  $\delta_j$  is the standard diagonal matrix of Kronecker indexes. To complete the formulation the above *j*-th equation have to be coupled with equation (6.3) where all the coefficients come with the index *j*; that is:

If the TMD is to be designed for the *i*-th structural mode with modal properties  $M_i$ ,  $K_i$  and  $C_i$ , the design problem is similar to the design of a TMD device for a *l*-DOF structure. Indeed, introducing the modal coordinate y(t), the resulting modal equation can be written as follows:

$$M_{i}\ddot{y}(t) + C_{i}\dot{y}(t) + K_{i}y(t) = \Phi_{i}^{T} \{\mathbf{f}(t) + d_{j}[c_{j}\dot{z}(t) + k_{j}z(t)].$$
(6.6)

Two critical problems, not usually considered in the standard design procedure of a TMD(s) device(s), are here discussed:

- 1) when using a commercial software for the finite element analyses (FEA), the modal mass normalization is pursued toward having that matrix built as an identity matrix;
- 2) when the primary system is a bridge, there are longitudinal and transversal symmetries which suggest to hang several identical devices at different points. This is especially true when the vibration to be mitigated give rise to local vibration problems, instead than the case when the vibration problems originates from an oscillation of the whole structural system.

The first aspect requires<sup>17</sup> to obtain from the numerical model the mass, damping and stiffness matrices, and to work directly on them toward the suitable normalization procedure for instance working in the MatLAB environment [12].

Due to the structural, material and geometry symmetry of the "Trasaghis" footbridge, it is reasonable to assume that the vibration of the structural elements occurs in phase or in opposition of phase, as well as in combined modes. Thus, the TMD system becomes a multiple tuned mass damper, with devices hung in several companion points.

Two situations may be identified:

<sup>&</sup>lt;sup>17</sup> If no specific options are provided in the FEA software options.

- among the modes of interest, there is one of them which sees the mass normalization associated with a selected hanging point to produce unit entries in all the companion points;
- among the modes of interest, there are two modes at the same frequency or at frequencies close to each other which see unit entries of the first eigenvector in one half of the companion points, while the remaining points see the unit entries in the second eigenvector.

For both these cases, the normalized modal mass is easily computed. A suitable per cent of it is then distributed across all the companion positions, and the TMD parameters are accordingly computed, within the hypothesis that these parameters are assumed to be equal for all the devices because of the symmetry hypothesis for the structure given previously.

Usually commercial FEA implement a normalization of the modal masses values, which is useful for the calculation, but that have no scientific meaning. This suggested implementing the following alternative procedure:

- 1) select the mode and the companion TMD hanging position;
- 2) identify and isolate a group of nodes whose masses could influence the single hanging point is. For instance, in the procedure proposed in this thesis, three nodes belonging to the structure are considered for this application; namely, one is the node at which the TMD will be physically hanging, and the other ones are its two adjacent nodes;
- perform a dynamic modal analysis and for each TMD location, collect the modal masses of any single cluster of nodes above selected. In other words, for the procedure proposed, one needs to collect the modal mass of the three nodes;
- 4) the dynamic modal analysis is then repeated with a rigid link connecting the hanging node to an additional node where the given added mass of the TMD device is assigned. This produces a further modal mass, whose ratio-to the sum of the cluster modal masses is computed toward the standard TMD design.

Summarizing in the above proposed procedure, instead of computing as usual the modal mass at the hanging node, and introducing the mass damper as  $\mu$  times the modal mass, it is considered the ratio  $\mu$  between the nodal masses as normalized by the FEA code. It is important to underline that, following this procedure, the influence of the mesh discretization on the result needs to be checked by, for example, comparing them with

the ones obtained from a coarser mesh. This because the discretization influences the choice of the cluster of nodes whose masses is influenced by the presence of the TMD.

A real-case exemplification of the proposed method is performed with reference of the "Trasaghis" footbridge (see *Appendix A*, section A.1 for more details about the bridge). A summary of the mains elements masses is given in Table 6.1.1. The performance of the designed TMD passive solution is checked with the reference of the wind loads as expressed in *Chapter 3*. The resulting forces coming from the wind velocity file act on both sides of the footbridge, as prescribed in [13]. Two different numerical analyses are performed:

- structural "static" analysis, from which the baseline for the dynamic vibration is obtained;
- structural "dynamic transient" analysis.

Subtracting the "static" response from the "dynamic" one in terms of displacements, it is possible to isolate the pure dynamic response of the pedestrian bridge. The time duration of the analysis is assigned to be 120 sec; i.e., twice the duration of the wind velocity time history. In this manner, the free vibrations of the system, once the external excitation stops, are also simulated.

As described in section 3.2.2 the role of the steel skeleton of the deck is dominant on the dynamic response of the bridge. The recorded signals from the in situ campaigns confirmed the role of the long tubular stays in determining the vibration of the system at frequencies in the range from 1 to 2 Hz. Actually, as a result of the modal analysis carried out on the numerical model (Figure 6.1.1) developed within Marc MENTAT2010 environment [14] (see section 3.2.2 for further information), the possible combinations of their vertical (in plane) movements correspond to the lowest frequency value, and those of their horizontal (out-of-plane) movements correspond to an intermediate frequency range, both resulting in modest movements of the deck. This different orthogonal behavior is due to the connections of the stays, which are represented by hinges in the vertical plane, but can transfer the moments in the orthogonal plane. A further slightly higher frequency show the movement of all the oblique elements in the same (transversal/out-of-plane) direction resulting in torsion of the central third of the deck. A summary of the frequencies range is given in Table 6.1.2.

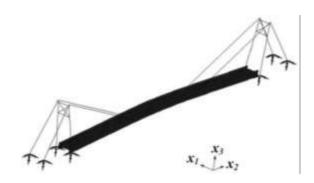


Figure 6.1.1. 3-D view of the numerical model of the "Trasaghis" footbridge implemented in Marc MENTAT2010.

Table 6.1.1. Masses of the structura	and non-structural elements for	or the	"Trasaghis"	footbridge.

System component	Single element mass[kg]	Number of elements	Mass [kg]
Larch revetment	-	-	11051
GLT main longitudinal beams	16140	2	32880
Wooden walking surface	-	-	7730
Timber beams under the walking surface	828	5	4140
"Internal" tubular steel stays	1760	4	7040
"External" tubular steel stays	845	4	3380
Steel pillars	6270	2	12540
Steel railing	-	-	1200
Steel bracings	-	-	3365
Steel transversal beams	169.7	30	5090
TOTAL			88416

Table 6.1.2. Frequency ranges for the lowest modes of the "Trasaghis" footbridge.

Set ID	Frequency	Involved	Oscillation plane
Set ID	[Hz]	Elements	Oscillation plane
1	1.06	Tubular stay	$x_2 - x_3$
2	1.41	Tubular stay	<i>x</i> <sub>2</sub> - <i>x</i> <sub>1</sub>
3	1.96	Tubular stay	$x_2 - x_1 \& x_2 - x_3$

In the case of "Trasaghis" footbridge the deck results quite stiff while the steel supporting skeleton is rather flexible. In particular the values around  $1\pm1.50$  Hz of the frequencies of the modes along  $x_3$  and  $x_1$  are associated to all the central nodes of the

stays, which move in a synchronous manner as reported in Table 6.1.2. This fact suggests to append the TMD directly to the tubular steel stays as reported in Figure 6.1.2.

The selected tuning frequencies for the design are:

- 1.05 Hz for the motion along the  $x_3$  (gravity) axis;
- 1.40 Hz for the motion along the  $x_1$  (transversal to the deck) axis.

The small discrepancies of the frequencies values with the values reported in Table 6.1.2 are due to the presence of the mass of the TMDs within the numerical model.

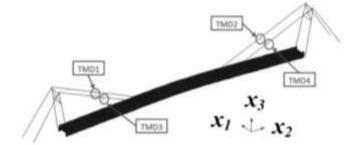


Figure 6.1.2. View of the implemented TMDs solution on "Trasaghis" footbridge. In particular TMD1 and TMD3 hung on the tubular elements in the l.h.s. of the footbridge, while TMD2 and TMD4 are hanged on the tubular elements in the r.h.s.

After having created a model of only 200 nodes, instead of the 500 nodes charactering the full scheme of Figure 6.1.1, the corresponding structural matrices are derived and further elaborated using MatLAB. The modal mass associated to the vertical mode is found to be 4000 kg, with 1000 kg in each of the four stay central nodes. For  $\mu$ =0.05, the added mass would then be 50 kg per stay. For the mode in the horizontal direction, the reduced model yields to a decoupling of two very close frequency values, so that only two nodes move synchronously (with unity displacement). In this case, a modal mass of 1800 kg is found and must be divided by the number of synchronous nodes (i.e., by 2<sup>18</sup>), leading to 900 kg each. For  $\mu$ =0.05, the added mass would then be 45 kg. The final TMD mass is then assumed to be 50 kg and is applied at each stay central node, having set  $\mu$ =0.0556 for the mode in the horizontal direction.

Adopting the table for the design of the TMD devices given by [15] and summarized in Table 6.1.3, one enters with the selected  $\mu$  value and a very low damping of the primary

<sup>&</sup>lt;sup>18</sup> Because two are the nodes that are moving synchronously.

system (of about 1%, which represents a conservative value of damping for wood structures) and obtains the optimum absorber parameters that are:

$$\alpha_{opt} \simeq 0.95$$

$$\xi_{opt} \simeq 10\%$$
(6.7)

Thus optimum stiffness and damping of the TMDs may be evaluated as:

$$\begin{aligned} k_{opt} &= 4\pi^2 \left( f \,\alpha_{opt} \right)^2 M_{TMD} \\ c_{opt} &= 2\xi_{opt} \sqrt{M_{TMD} k_{opt}} \end{aligned}$$
(6.8)

Table 6.1.3. Optimum absorber	parameters for	TMD	design	[15].
-------------------------------	----------------	-----	--------	-------

μ	ζs	<b>R</b> opt	$\alpha_{s}$	ζopt
	0	10.138	0.9876	0.04981
	0.01	7.743	0.9859	0.04981
0.01	0.02	6.205	0.9819	0.04981
	0.05	3.798	0.9704	0.04982
	0.1	2.249	0.9436	0.04982
	0	6.058	0.9636	0.08566
	0.01	5.110	0.9592	0.08566
0.03	0.02	4.424	0.9545	0.08566
	0.05	3.109	0.9380	0.08567
	0.1	2.036	0.9032	0.08569
	0	3.602	0.8861	0.1527
	0.01	3.285	0.8789	0.1527
0.1	0.02	3.014	0.8714	0.1528
	0.05	2.399	0.8468	0.1529
	0.1	1.765	0.7991	0.1531
	0	2.865	0.7906	0.2097
	0.01	2.680	0.7815	0.2098
0.2	0.02	2.516	0.7721	0.2099
	0.05	2.113	0.7421	0.2103
	0.1	1.649	0.6862	0.2112

Then assuming  $M_{TMD} = 50$  Kg as evaluated above one achieves:

- for the motions along  $x_3$  characterized by a frequency  $f = 1.05 \text{ Hz} \Rightarrow k_{opt} = 4\pi^2 (1.05 \times 0.95)^2 50 = 1950 \text{ N / } m$ ;  $c_{opt} = 2 \times 0.1 \sqrt{50 \times 1950} = 60 \text{ Ns / } m$ ;
- for the motion along  $x_1$  characterized by a frequency  $f = 1.40 \text{ Hz} \rightarrow$

$$\begin{split} k_{opt} &= 4\pi^2 \left(1.40*0.95\right)^2 50 = 3400 N \,/\, m \\ c_{opt} &= 2*0.1 \sqrt{50*3400} = 80 N s \,/\, m \end{split} .$$

Summarizing, for this proposed TMD system, called TMD\_A, the main features are collected in Table 6.1.4

ID	Value
mass M <sub>TMD</sub>	50 Kg each TMD
stiffness kx1	3400 N/m
damping <i>cx1</i>	80 Ns/m
stiffness kx3	1950 N/m
damping $c_{x3}$	60 Ns/m

Table 6.1.4. Main features of the TMD\_A solution.

The alternative design approach consists of the following steps:

- identify the sub-system whose vibrations have to be mitigated. In the considered case study, it is represented by the four tubular stays of total mass 1760 Kg (Table 6.1.1). Each stay is discretized into four hollow elements leading to a nodal mass of 440 kg per node. For further calculations, the attention is focused on the three internal nodes.
- 2) when considering the mode at 1.05 Hz and a rigid link to the added mass, the modal analysis provides the following (dimensionless) modal masses as provided by the FEA coded employed [14]: 3.22, 6.55 and 4.52 for the three internal nodes, and 0.77 for the added mass. These values correspond to  $\mu$ = 5.39%; the error is 7% with respect to the "lightest" FE mesh, and it will be seen to be dependent on the discretization.
- 3) for the mode at 1.40 Hz and a rigid link to the added mass, the modal analysis provides the following (dimensionless) modal masses: 4.09, 7.31 and 4.12 for the three internal nodes, and 0.87 for the added mass. The resulting value of  $\mu$  is 5.61%, with an even lower error with respect to the "lightest" FE mesh.

Thus, it is seen that the alternative approach leads to consistent results within a suitable degree of approximation. If the reasoning is repeated with a model in which the stays are just discretized into two elements, one obtains higher over-estimations of the values of  $\mu$ ; namely, 0.75 and 0.76 in the two directions  $x_3$  and  $x_1$ , respectively.

The design of the implemented TMD's solution is summarized in Figure 6.1.3. Each TMD are realized by a steel hollow cylinder of radius R = 35 cm, thickness t = 4 mm and length l = 64 cm. The design choices allow to fit well with both aesthetics and structural aspects; in fact it is hanged in the hinge located in the middle of the stay.

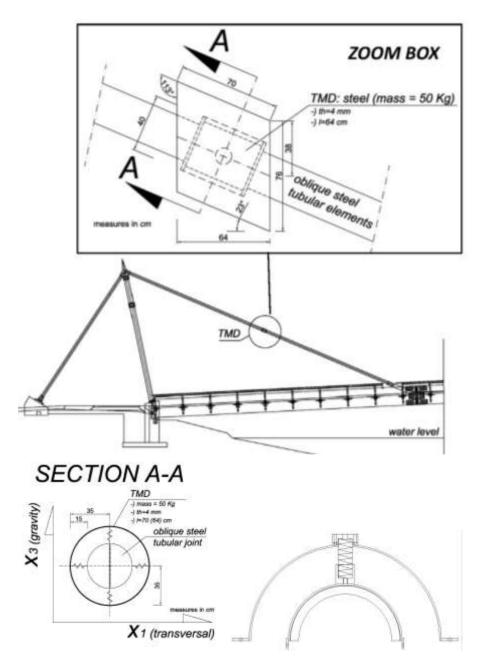


Figure 6.1.3. Above: Lateral view of the location of the i-th TMD. Below: cross-section of the i-th TMD.

The performance of the proposed TMD system (TMD\_A), is evaluated by computing the root-mean-square values of the acceleration and displacement time histories obtained at nodes n8 and n22 in the two directions  $x_1$ ,  $x_3$  respectively located at the mid span of the deck, and the mid length of the oblique steel stay (Figure 6.1.4). The calculations are carried out by considering both the whole duration of these signals and only their tails (after the 60 sec. of wind excitation); the results are reported in Table 6.1.5 and Table 6.1.6, respectively. An example of the analyzed time histories results is plotted from Figure 6.1.5 to Figure 6.1.7 for the accelerations and displacements respectively for the selected nodes.

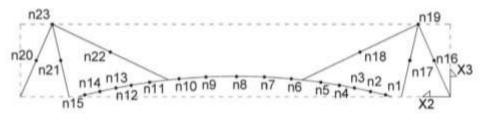


Figure 6.1.4. Grid of nodes.

Table 6.1.5. Root-mean-square values computed on the whole duration (120 sec) of the acceleration and displacement time histories obtained from the numerical analyses. The peak values are also reported in italic, and they are expressed in  $m/s^2$  for the accelerations, and in mm for the displacements.

		nod	e n8	
	acceleration [m/s <sup>2</sup> ]		displacement [m	
	$x_{I}$	<i>x</i> <sub>3</sub>	$x_{I}$	<i>X</i> 3
N- TMD	0.1348	0.0665	1.2015	0.5593
No TMD	0.6079	0.3503	5.8083	2.7658
	0.1335	0.0659	1.1754	0.5467
TMD_A	0.6217	0.3571	5.9399	2.8283
		node	e n22	
	accolorat	ion Im/s21	displacen	ant [mm]

	accelerati	acceleration [m/s <sup>2</sup> ]		nent [mm]
	<i>x</i> 1	Х3	<i>x</i> 1	Х3
No TMD	1.089	0.1605	15.8961	3.8396
	4.8971	0.7539	73.4899	19.8873
TMD A	0.8296	0.0822	12.7703	1.9309
TND_A	3.6528	0.5396	58.4994	14.0121

		nod	e n8	
	acceleration [m/s <sup>2</sup> ]		displacen	ient [mm]
	<i>x</i> 1	Х3	<i>x</i> 1	x3
No TMD	0.0108	0.1174	0.1405	0.0669
NO IMD	-0.0121	0.0896	0.1686	0.0580
	0.0058	0.0597	0.0513	0.0239
TMD_A	0.0011	-0.2571	0.1547	0.0735

Table 6.1.6. Root- mean-square values computed on the tails (the first 15 sec after the event) of the of
the response time histories obtained from the numerical analyses. The peak values are also reported in
italic, and they are expressed in $m/s^2$ for the accelerations, and in $mm$ for the displacements.

	node n22			
	acceleration [m/s <sup>2</sup> ]		displacement [mm]	
	<i>x</i> 1	Х3	<i>x</i> 1	x3
No TMD	0.0051	0.1014	1.5997	2.4365
	-0.0059	0.0883	0.6423	0.2851
TMD_A	0.0027	0.0113	0.9543	0.2498
	0.0012	0.0275	0.2421	0.1143

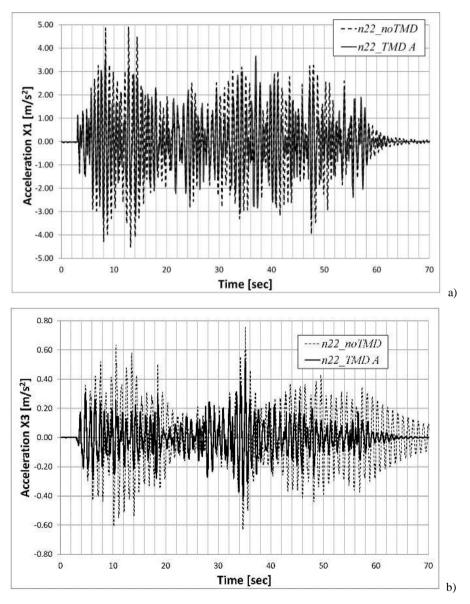


Figure 6.1.5. Time histories response before and after mounting the passive control system ("TMD\_A") designed. Acceleration responses for the node n22 along x1 (a) and x3 (b), respectively.

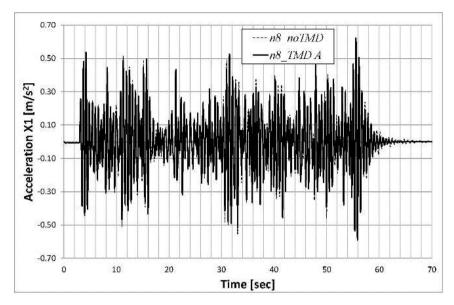


Figure 6.1.6. Time histories response before and after mounting the passive control system ("TMD\_A") designed. Acceleration responses along x1 at the node n8.

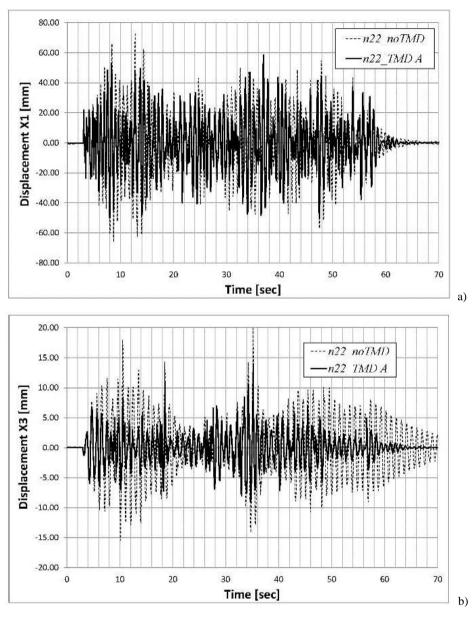


Figure 6.1.7. Time histories response before and after mounting the passive control system ("TMD\_A") designed. Displacement responses for the node n22 along x1 (a) and x3 (b), respectively. The response of the node n8 is meaningful due to the stiffness of the deck.

A significant mitigation of the vibrations is achieved at the central node, n22, of the l.h.s. stay, whereas at the deck node, n8, only the free response seems to be reduced because of the deck stiffness. Indeed, when the whole duration of the signals at node n8 is considered in the calculations of the root-means-square, the devices seem to be only able to reduce the peaks in terms of both acceleration and displacement, not the root-mean-square values.

A further understanding of the implemented passive solution is achieved analyzing the results within the frequency domain as reported in Figure 6.1.8 and Figure 6.1.9. These results confirm the observations about the TMD's effect in terms of vibrations mitigation. In fact, by looking at the frequencies considered in the design of the TMD(s), i.e. 1.05 and 1.40 Hz, it is clear that the designed architecture is able to smooth the peaks of the spectra, obtained from the numerical simulations under wind loads for the "Trasaghis" footbridge, for both the accelerations and displacements. As already observed for the response in terms of standard accelerations and displacements, the effects of the implemented solution are clearly evident for the central node, n22, of the l.h.s. stay, whereas the TMD's effects are not so significant as expected.

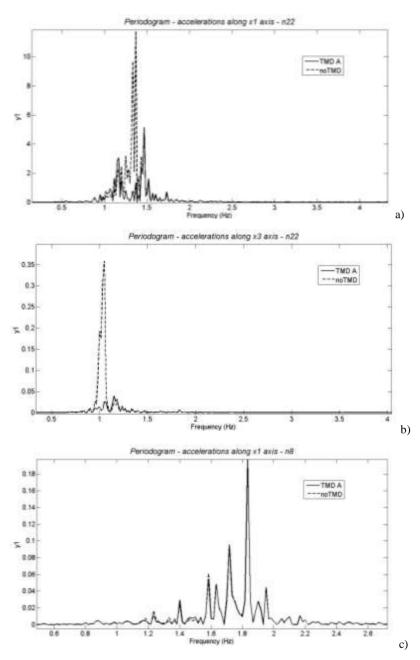


Figure 6.1.8. Response spectra as obtained from the acceleration responses along  $x_1$  (a) and  $x_3$  (b) at the node n22; and (c) from the acceleration response along  $x_1$  at the node n8, before and after the installation of the passive control system (TMD\_A).

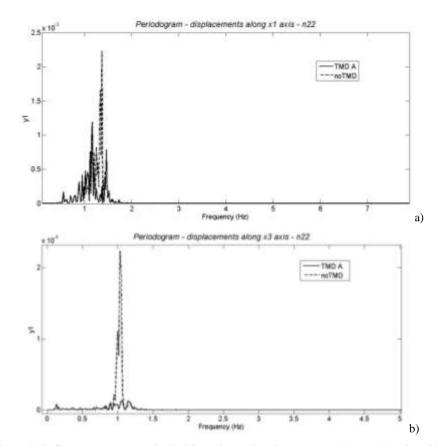


Figure 6.1.9. Response spectra as obtained from the acceleration responses along  $x_1$  (a) and  $x_3$  (b) at the node n22, before and after the installation of the passive control system (TMD\_A).

## 6.2 Semiactive control solutions

In this section the possibility of introducing a control law to improve the vibration mitigation, induced by the wind action on the "Trasaghis footbridge", achieved by passive devices is discussed. In particular a semiactive strategy is adopted.

The basic idea behind the semiactive control solution is to change some parameters of the structural system to optimize the behaviour of the system with respect to a control criterion. In this approach the only energy necessary here is the energy for regulating the devices which modify the parameters of the system [16]. A semiactive control system varies the values of the stiffness and/or the damping. Recalling for simplicity and without loss of generality the equation of motion for a *1*-DOF system (see equation (2.3)), this can be represented simply as:

$$\ddot{x}(t) + 2\xi \omega_0 \dot{x}(t) + \omega_0^2 x(t) = \frac{F(t)}{m} + f_C, \qquad (6.9)$$

where the control force in defined as:

$$f_{C} = -\left[2\xi\omega_{0}\alpha_{C}(t)\right]\dot{x}(t) - \left[\omega_{0}^{2}\alpha_{K}(t)\right]x(t), \qquad (6.10)$$

in which  $\alpha_{c}(t)$  and  $\alpha_{K}(t)$  are functions of time. Control systems where parameters of the system can be varied are called variable structure systems.

This implementation is mainly motivated by the fact that the good performance of the passive solutions, as observed in section 6.1 is restricted to a narrow window around the tuning frequency chosen during the device design [7].

Nevertheless, the global cost of the footbridge suggests to restrict the investigation within very simple technological solutions. Thus, first of all a limitation is set on the available feedback; the relative displacement z and velocity  $\dot{z}$  between the anchorage point and the secondary mass seems to be a suitable solution. A second limitation concerns the simplicity of the control law, which is generally made of a premise (IF) and a consequence (THEN). In this way a good compromise between the cost and efficacy of the solution can be achieved.

One assumes that, at each time step, the premise relies on the instantaneous values of the relative displacement and velocity introduced above, while the consequence is on the value of the device output force. In particular the simple "bang-bang control" is adopted [16].

Bang-bang control provides a simple and yet often effective approach. It is usually applied for dissipative devices in order to increase their capabilities in dissipating energy with respect to a classical passive device. The working principle of this control, approach can so summarized:

- when the relative displacement and the relative velocity of the two ends of the damping device are in the same direction, bang-bang control acts in the direction of increasing the forces in the device to a maximum value;
- vice-versa when the relative displacement and the relative velocity of the ends of the device are in opposite directions, the control law decreases the value of the forces to a minimum in order to make the device movement as easy as possible.

Thus, the maximum value of the forces is obtained when the control signal reach its maximum, and the minimum when the control signal reach its minimum. In other words the general representation of the control law can be written as:

$$\begin{cases} F_C(t) = F_{MAX} & \text{if } sign(\dot{x}) = -sign(\dot{x}) \\ F_C(t) = F_{MIN} & \text{if } sign(\dot{x}) = sign(\dot{x}) \end{cases}.$$
(6.11)

Then it is clear as the minimum value of the control parameter should be set at as small a level as possible, to reduce the dissipative forces to a minimum, while the maximum value of the control parameter should be at the optimal value, that is a value which provides the maximum energy dissipation.

The advantage of this control approach, is that it is easy to implement the "bang-bang" strategy in a real-time algorithm to achieve the structural control purpose. In fact the control signal switches between two values and only the relative displacement needs to be measured by a sensor and fed back to the control signal. Thus, the function sign(x) can be obtained simply by the relative displacement evaluated by the recorded signal and there is no need to measure the velocity.

More in details for the case study of the "Trasaghis" footbridge the control force is given different values according to the sign of the product zz. In particular one writes:

$$\begin{cases} F(t) = F_1 & \text{if } \dot{z}z > 0 \\ F(t) = F_2 & \text{if } \dot{z}z < 0 \end{cases}$$
(6.12)

where  $F_1$  and  $F_2$  are the maximum and minimum levels, respectively, and they are assigned as functions of the feedback. Note that without an accelerometer one cannot estimate the inertial force of the TMD, whereas the corresponding damping force and the elastic one can be directly calculated based on the available feedback,  $\dot{z}$  and z, respectively. An example of the relative displacement along  $x_1$  and  $x_3$  direction is reported in Figure 6.2.1.

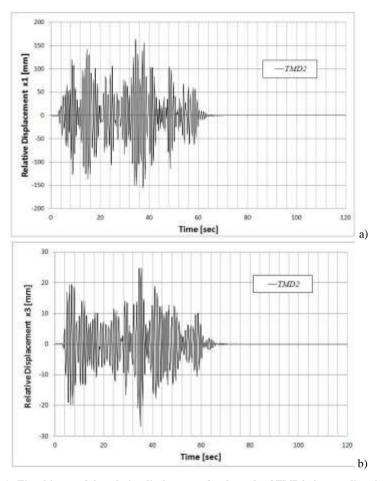


Figure 6.2.1. Time history of the relative displacement for the node of TMD2 along  $x_1$  direction a), and  $x_3$  direction b).

The inertial term is accounted by artificially augmenting the sum of the last two terms in the l.h.s. of equation (6.3) when the maximum control force is calculated at each step of the analysis. The simplest option consists of introducing an amplification factor of 1.8 when assigning the  $F_2$  value in equation (6.12), whereas a de-amplification factor of 0.6 is set to effectively reduce the required minimum control force  $F_1$ . Such a control scheme is denoted as "s1". Two more control policies are investigated:

• scheme denoted as "s2": the factors are progressively varied based on the feedback;

• scheme denoted as "s3": a contribution which replaces the inertia term by an estimate is added.

To check the performance of the semiactive solution, three different configurations for the TMDs are considered:

- 1) the first one is the TMD\_A solution as evaluated in section 6.1 (see Table 6.1.4 for the main features of this solution);
- 2) the second one, called TMD\_B solution, is obtained tuning the TMD at the frequency f = 1.96 Hz (that regards both the motion along x₁ and x₃ as from Table 6.1.2). Following the standard design of the TMD described in section 6.1 and given by [15], assuming a mass of the singular TMD equal to M<sub>TMD</sub> = 50 Kg the stiffness of the TMD may be evaluated as: f = 1.96 Hz → k<sub>opt</sub> = 4π<sup>2</sup> (1.96\*0.95)<sup>2</sup> 50 = 6727N/m for both x₁ and x₃ directions. The damping is assumed conservatively equal to case TMD\_A along x₁, that is c<sub>opt</sub> = 480Ns/m, for both the direction x₁ and x₃;
- 3) the third architecture, called TMD\_C solution, is obtained mixing the two previous solutions. It mounts the devices of TMD\_A in the positions denoted as TMD1 and TMD4 in Figure 6.1.2 and the devices of TMD B in the other two locations (Figure 6.2.2).

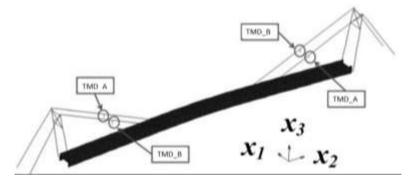


Figure 6.2.2. View of the TMD\_C of the "Trasaghis" footbridge.

The main results achieved by the semiactive architecture above described are summarized here below. In particular the results are represented in terms of the root-mean-square for the whole signal (Table 6.2.1) of accelerations and displacements, and in terms of the root-mean-square of the first 15 sec after the event (Table 6.2.2) for both the signal of accelerations and displacements. The control solutions are denoted by the same acronym as the corresponding strategies adopted; i.e. " $s_i$ , i=1,2,3".

The results shown as the semiactive implementation based on the stiffness and damping coefficients of TMD\_A is unable to improve the passive performance, while this occurs for TMD B. In other words a semiactive approach allows the designer to tune the device on a frequency higher than the first one so that a similar effectiveness can be achieved in a more robust manner. It is worth noticing that, from Tables 5 and 6, the least sophisticated control scheme denoted as "TMS\_Bs1" results to be the most effective one, with a performance which is very similar (sometimes better) to that of the TMD\_A passive configuration, with the difference that the latter one is tuned (in the two directions  $x_1$  and  $x_3$ ) on a single frequency f = 1.96 Hz. Examples of the controlled acceleration are given from Figure 6.2.3 to Figure 6.2.10.

Table 6.2.1. Root-mean-squares on the whole duration (120 sec) of the response time histories computed from the numerical analyses carried out in a semi-active environment. The value in italic in each cell gives the peak value (with its sign) expressed as  $m/s^2$  for the accelerations, and as mm for the

	node n22				
	acceleration [m/s <sup>2</sup> ]		displacement [mm]		
	<i>x</i> 1	Х3	<i>x</i> 1	<i>X3</i>	
No TMD	1.089	0.1605	15.8961	3.8396	
	4.8971	0.7539	73.4899	19.8873	
TMD_As1	0.8365	0.0866	12.3125	2.1739	
	3.9462	0.4949	54.4649	14.2123	
TMD_As2	0.9215	0.0833	13.1832	2.0563	
	4.4611	0.5003	59.4422	13.4807	
TMD_As3	0.8923	0.0908	12.4023	2.2441	
	4.2431	0.5171	56.3241	14.2929	
TMD_Bs1	0.9861	0.1415	15.5892	3.5921	
	-4.6187	-0.6132	-74.6187	17.4190	
TMD_Bs2	1.0143	0.1605	16.3107	3.5868	
	5.0301	0.7539	-81.9679	17.7537	
TMD_Bs3	0.9715	0.1391	15.3339	3.5241	
	-4.4669	0.6060	-70.3799	17.2960	
TMD_Cs1	0.9858	0.1414	15.5890	3.5919	
	-4.6184	-0.6130	-74.6185	17.4188	
TMD_Cs2	0.9937	0.1401	15.7661	3.5537	
	4.6502	-0.6104	-75.6504	17.4577	
TMD_Cs3	0.9714	0.1390	15.3338	3.5240	
	-4.4668	0.6059	-70.3798	17.2959	

displacements.

	node n22				
	acceleration [m/s <sup>2</sup> ]		displacement [mm]		
	<i>x</i> 1	Х3	<i>x</i> 1	<i>X3</i>	
No TMD	0.0051	0.1014	1.5997	2.4365	
	-0.0059	0.0883	0.6423	0.2851	
TMD_As1	0.0434	0.0117	0.5778	0.3039	
	0.0412	-0.0396	0.3939	0.1873	
TMD_As2	0.0555	0.0103	0.7216	0.3955	
	0.0989	-0.0312	0.5412	0.1930	
TMD_As3	0.0393	0.0150	0.5420	0.4022	
	-0.1015	-0.0475	0.3979	0.1903	
TMD_Bs1	0.1659	0.0917	2.5431	2.3284	
	0.6093	-0.2043	0.2570	0.1410	
TMD_Bs2	0.1970	0.0914	3.0474	2.3191	
	0.7351	-0.2033	0.1992	0.1126	
TMD_Bs3	0. 1450	0.0897	2.2009	2.2747	
	0.5368	-0.1997	0.2547	0.1394	
TMD_Cs1	0.1659	0.0917	2.5430	2.3282	
	0.6093	-0.2043	0.2568	0.1408	
TMD_Cs2	0.1656	0.0913	2.5307	2.3143	
	0.6084	-0.1988	0.2610	0.1426	
TMD_Cs3	0.1449	0.0896	2.2008	2.2746	
	0.5367	-0.1996	0.2546	0.1393	

Table 6.2.2. Root-mean-squares on the tail (the first 15 sec after the event) of the time histories computed as response from the numerical analyses carried out. The value in italic in each cell gives the starting point (with its sign) of the decay, expressed as  $m/s^2$  for the accelerations, and as mm for the displacements.

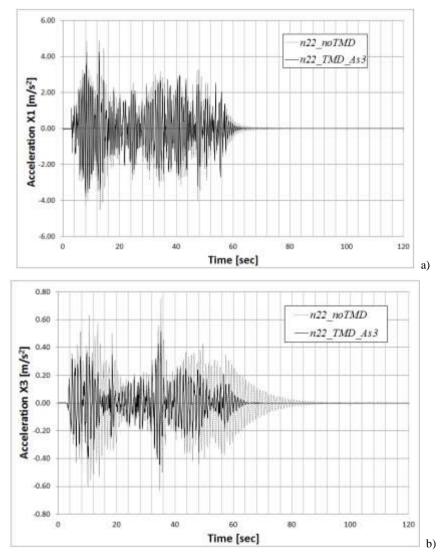


Figure 6.2.3. Time histories of the accelerations (a) along direction  $x_1$  and (b) along direction  $x_3$  obtained for node n22 under the semiactive control scheme TMD\_As3.

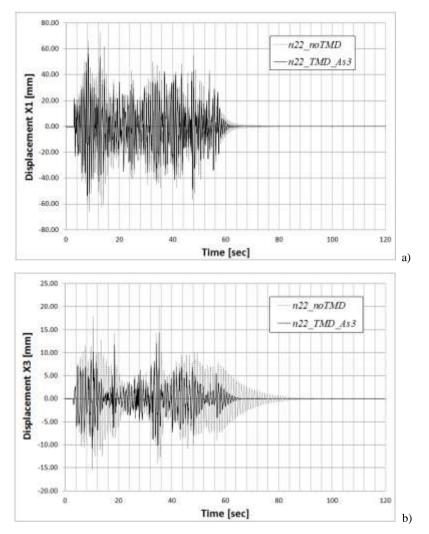


Figure 6.2.4. Time histories of the displacements (a) along direction  $x_1$  and (b) along direction  $x_3$  obtained for node n22 under the semiactive control scheme TMD\_As3.

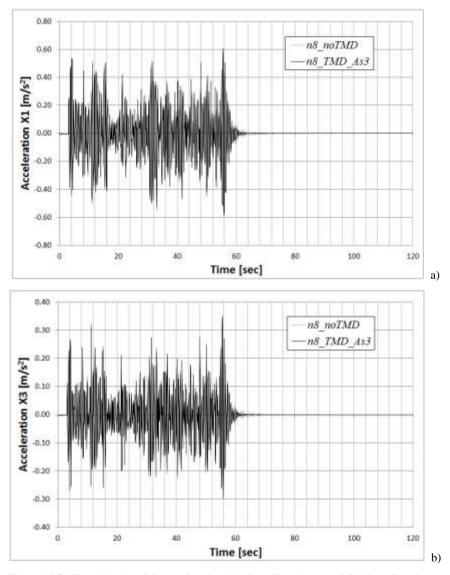


Figure 6.2.5. Time histories of the accelerations (a) along direction  $x_1$  and (b) along direction  $x_3$  obtained for node n8 under the semiactive control scheme TMD\_As3.

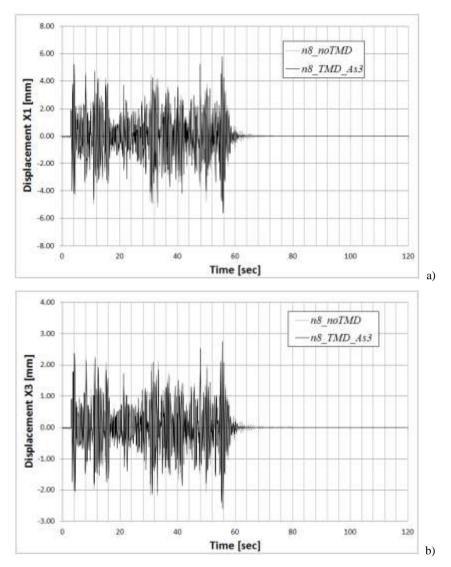


Figure 6.2.6. Time histories of the displacements (a) along direction  $x_1$  and (b) along direction  $x_3$  obtained for node n8 under the semiactive control scheme TMD\_As3.

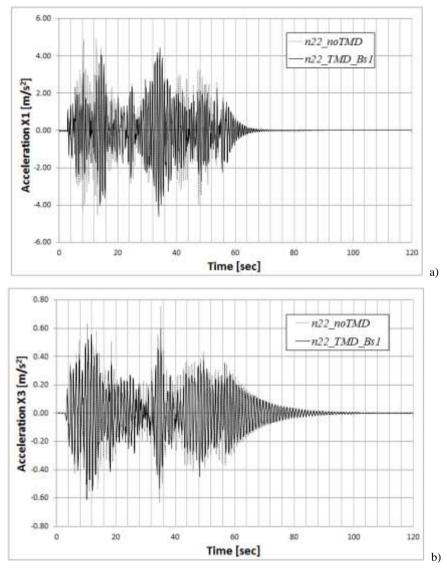


Figure 6.2.7. Time histories of the accelerations (a) along direction  $x_1$  and (b) along direction  $x_3$  obtained for node n22 under the semiactive control scheme TMD\_Bs1.

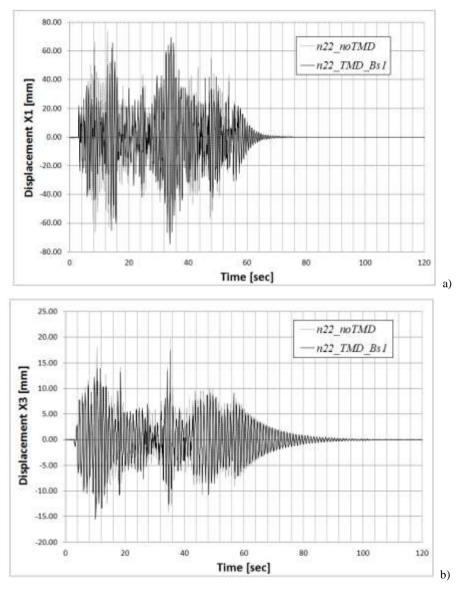


Figure 6.2.8. Time histories of the displacements (a) along direction  $x_1$  and (b) along direction  $x_3$  obtained for node n22 under the semiactive control scheme TMD\_Bs1.

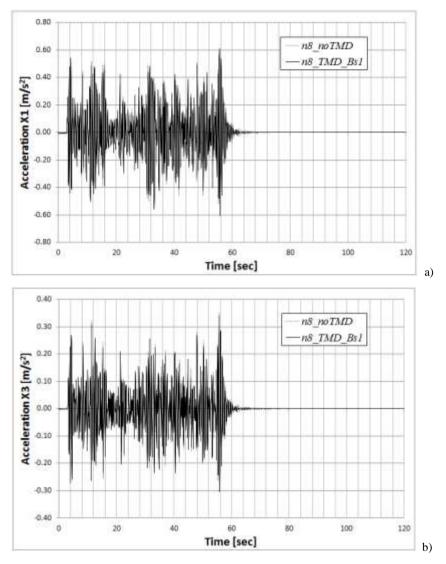


Figure 6.2.9. Time histories of the accelerations (a) along direction  $x_1$  and (b) along direction  $x_3$  obtained for node n8 under the semiactive control scheme TMD\_ Bs1.

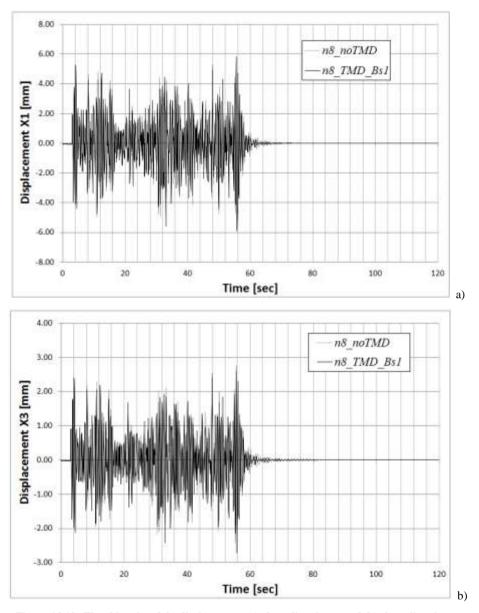


Figure 6.2.10. Time histories of the displacements (a) along direction  $x_1$  and (b) along direction  $x_3$  obtained for node n8 under the semiactive control scheme TMD\_Bs1.

### 6.3 Active control solutions

Once achieved the reduction of the FEM for the "Trasaghis" footbridge (see section 5.2), a structural control is implemented within the MatLAB environment in order to mitigate the vibration induced by wind loads (see section 5.2 and section 5.3).

The implemented active control strategy, or in other words the control forced introduced within the model in order to achieve the purpose of the control, is based on the Linear Quadratic Regulator (LQR) theory. This theory belong to the more general field of the theory of optimal control that concern with the possibility to manage a given dynamic system at its minimum "cost". In all those cases where the dynamic of the system is described by a set of linear differential equations and the cost may be described by a quadratic function, a feedback control can be achieved by applying the LQR strategy as follows described ([1] and [17]).

Let's consider a linear dynamic system expressed in state variable form as:

$$\dot{x}(t) = \mathbf{A}x(t) + \mathbf{B}u(t), \qquad (6.13)$$

where  $x(t) \in \mathbb{R}^n$ ,  $u(t) \in \mathbb{R}^m$  are respectively the states and input vectors, and the initial conditions are assumed to be  $x(0) = x_0$ .

The LQR strategy problem can be presented as follows: "*find a state-variable feedback* (*SVFB*) *control that gives desirable closed-loop properties, such that:*"

$$u(t) = -Kx(t) + v(t).$$
(6.14)

where K is unknown matrix to be found and v(t) the new command input.

Substituting equation (6.14) in (6.13) the problem becomes:

$$\dot{x}(t) = (\boldsymbol{A} - \boldsymbol{B}\boldsymbol{K})x(t) + \boldsymbol{B}v(t) = \boldsymbol{A}_{c}x(t) + \boldsymbol{B}v(t), \qquad (6.15)$$

where  $A_{c} = A - BK$  represents the closed-loop plant matrix.

It is reasonable now to design an optimal SVFB control. Thus, let's introduce the *performance index (PI)* as:

$$J \triangleq \frac{1}{2} \int_0^\infty \left( x^T \boldsymbol{Q} x + u^T \boldsymbol{R} u \right) dt , \qquad (6.16)$$

where  $\boldsymbol{Q} = \begin{bmatrix} \boldsymbol{Q}_{1n} & \cdots & 0 \\ \vdots & \boldsymbol{Q}_{ii} & \vdots \\ 0 & \cdots & \boldsymbol{Q}_{nn} \end{bmatrix}$  and  $\boldsymbol{R} = \begin{bmatrix} r_i \delta_{ij} \end{bmatrix}$ , i, j = 1, 2, ..., m are the so called weighting

square diagonal matrices. The performance index J can be interpreted as an energy function.

Since the only concerns are the internal stability properties of the closed-loop system, it is reasonable to assume v(t) = 0. Thus, substituting equation (6.14) into (6.16) we obtain:

$$J \triangleq \frac{1}{2} \int_0^\infty \left[ x^T \left( \boldsymbol{Q} + \boldsymbol{K}^T \boldsymbol{R} \boldsymbol{K} \right) x \right] dt .$$
 (6.17)

Then the optimal SVFB design can be written as:

find **K** such that J=min(J); (6.18)

that is in other words, the optimal design concerns the search for the minimum of the energy function J. Since the plant matrix is linear and the PI is quadratic, the problem to minimize J is called the *Linear Quadratic Regulator* (LQR) where the word 'regulator' refers to the fact that the function of this feedback is to regulate the states to zero (see section 5.1). Depending on the parameters selected for the weighted matrices Q and R during the design of the control law, the closed-loop system will exhibit a different response.

It is important to observe that since x(t) and u(t) are weighted in *J*, in order to minimize the function *J* neither x(t) nor u(t) can be too large. Note that if *J* is minimized, then it is for sure finite quantity, and since by definition it is an infinite integral of x(t), then one can said that x(t) goes to zero as time *t* goes to infinity. This in other words guarantees that the closed-loop system will be stable.

Assuming that the closed-loop system is stable so that x(t) goes to zero as time t goes to infinity, it can be shown as the function J is a constant that depends only on a new auxiliary matrix **P** and the initial conditions, and it is independent from **K**. Then the optimal feedback **K** can be evaluated as follows:

$$\boldsymbol{K} = \boldsymbol{R}^{-1} \boldsymbol{B}^T \boldsymbol{P} , \qquad (6.19)$$

where P is a constant auxiliary matrix that can be evaluated solving the following continuous time *algebraic Riccati equation* (ARE):

$$\boldsymbol{A}^{T}\boldsymbol{P} + \boldsymbol{P}\boldsymbol{A} - \boldsymbol{P}\boldsymbol{B}\boldsymbol{R}^{-1}\boldsymbol{B}^{T}\boldsymbol{P} + \boldsymbol{Q} = 0.$$
(6.20)

This means in other words that the minimum of J can be evaluated solving equation (6.19) which only depends on the initial condition. Thus, the "cost" of the SVFB strategy can be computed knowing only the initial conditions, and so before the control law is applied to the system.

It is worth underlining that the LQR design procedure is guaranteed to produce a feedback that stabilizes the system if two basic properties hold:

- Theorem 1. Let the system (*A*,*B*) be reachable (see section 5.1). Let *R* be positive definite and *Q* be positive definite. Then the closed loop system (*A*-*BK*) is asymptotically stable;
- **Theorem 2.** Let the system (A,B) be stabilizable. Let R be positive definite, Q be positive semi definite, and (A, Q) be observable (see section 5.1). Then the closed loop *system* (A-BK) is asymptotically stable [18].

Summarizing, the above strategy to design the LQR feedback K can be represented by the following flowchart:

- 1) select design parameter matrices Q and R;
- 2) find  $\boldsymbol{P}$  by solving equation (6.20);
- 3) find the SVFB K by solving equation (6.19).

Concluding this introduction about LQR approach, this control strategy to feedback design it is characterized by:

- selecting some design matrices *Q* and *R* that are tied to the desired closed-loop performance;
- introducing an intermediate quantity *P*;
- solving a matrix design equation;
- obtaining a guaranteed solution that stabilizes the system;
- obtaining very little insight into the robustness or structure of the closed-loop system.

Following the LQR control law strategy above introduced, the proposed active structural control is then achieved by introducing four forces  $F_Y$  acting along the *Y* direction (i.e. transversal to the deck), and applied in the middle of the four oblique steel stays (Figure 6.3.1). In Figure 6.3.2 the time history of the obtained control forces implemented for the three MOR models are shown. The choice to introduce the control forces only along *Y* direction is due to the fact that, as shown in section 5.2 and section

5.3, the displacements, and also the related accelerations, under wind load are about ten times bigger than the associated ones along the vertical (gravity) Z direction.

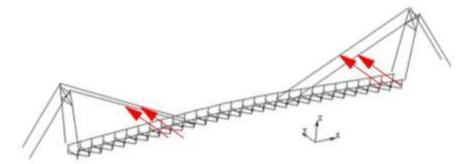


Figure 6.3.1. Scheme of the control forces implemented for the "Trasaghis" footbridge.

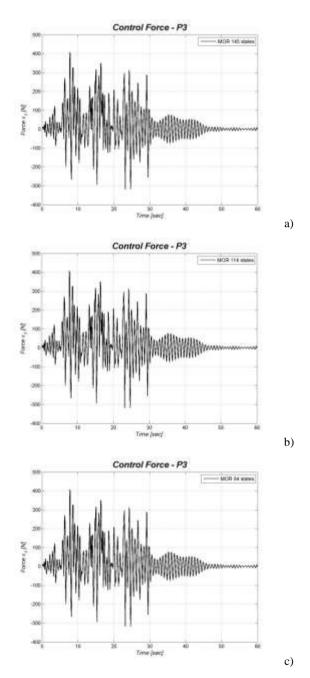


Figure 6.3.2. Time history of the control forces – *node* 3. a) MOR 145 states; b) MOR 114 states; c) MOR 84 states.

In the following figures (from Figure 6.3.3 to Figure 6.3.8) the results in terms of displacements and accelerations along Y and Z axes for the case identified as respectively "MOR 145 states", "MOR 114 states" and "MOR 84 states", obtained from the above active control strategy, are compared with the related uncontrolled cases.

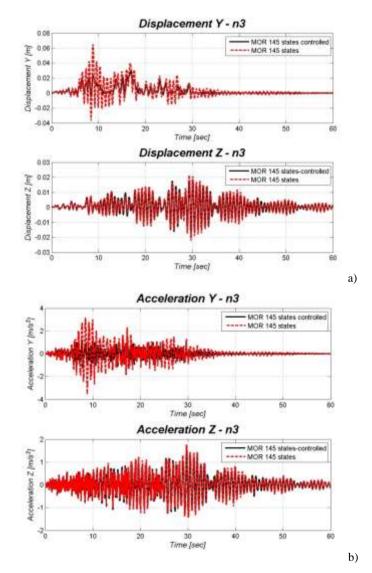


Figure 6.3.3. Active control solution for the "Trasaghis" footbridge - model MOR 145 states: comparison of uncontrolled vs. controlled. Response of: a) Displacements along Y and Z axes – *node* 3; b) Accelerations along Y and Z axes (zoom between 10 to 20 sec) – *node* 3.

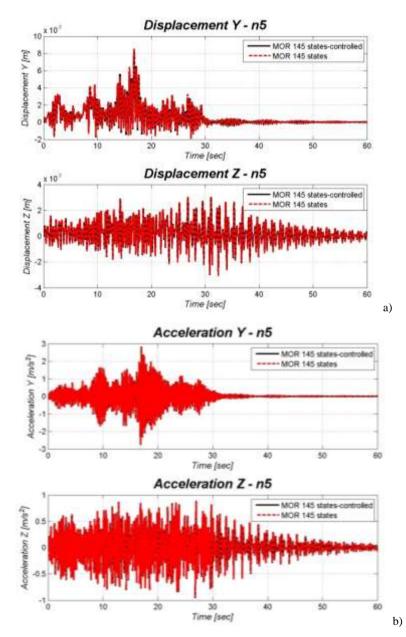


Figure 6.3.4. Active control solution for the "Trasaghis" footbridge - model MOR 145 states: comparison of uncontrolled vs. controlled. Response of: a) Displacements along Y and Z axes – *node* 5; b) Accelerations along Y and Z axes (zoom between 10 to 20 sec) – *node* 5.

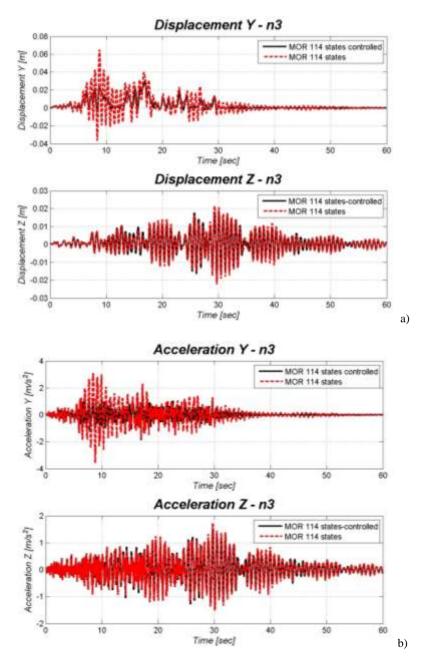


Figure 6.3.5. Active control solution for the "Trasaghis" footbridge - model MOR 114 states: comparison of uncontrolled vs. controlled. Response of: a) Displacements along Y and Z axes – *node* 3; b) Accelerations along Y and Z axes (zoom between 10 to 20 sec) – *node* 3.

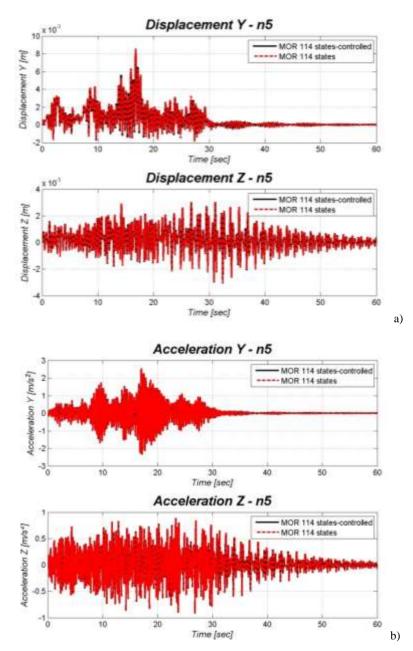


Figure 6.3.6. Active control solution for the "Trasaghis" footbridge - model MOR 114 states: comparison of uncontrolled vs. controlled. Response of: a) Displacements along Y and Z axes – *node* 5; b) Accelerations along Y and Z axes (zoom between 10 to 20 sec) – *node* 5.

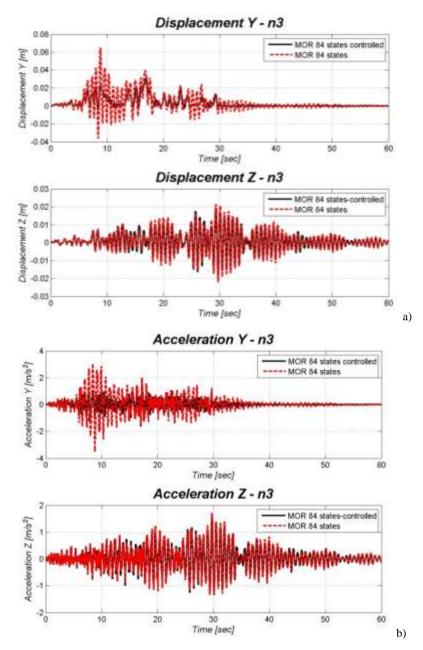


Figure 6.3.7. Active control solution for the "Trasaghis" footbridge - model MOR 84 states: comparison of uncontrolled vs. controlled. Response of: a) Displacements along Y and Z axes – *node* 3; b) Accelerations along Y and Z axes (zoom between 10 to 20 sec) – *node* 3.

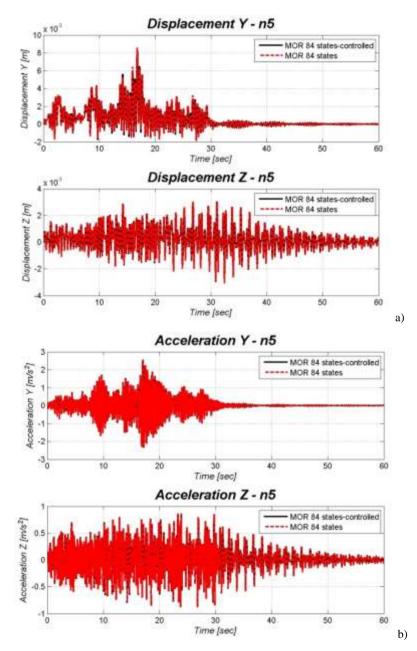


Figure 6.3.8. Active control solution for the "Trasaghis" footbridge - model MOR 84 states: comparison of uncontrolled vs. controlled. Response of: a) Displacements along Y and Z axes – *node 5*; b) Accelerations along Y and Z axes (zoom between 10 to 20 sec) – *node 5*.

Concluding, the active proposed structural control implemented for the wind vibration control of the "Trasaghis" footbridge seems to be consistent in terms both of quality and quantity of the vibrations reduction. The better results, due to the design choices about the control forces, are achieved for the reduction of the vibration along the transversal Y axis; while small advantage seems to appear along Z direction. Moreover the results shown, as expected, that the implementation of an active control solution allows to achieve better results in terms of vibrations mitigation compared with the other solutions previously analyzed [19].

#### 6.4 References

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# **Chapter 7** Conclusions

In this thesis some control solutions for the vibrations mitigation of pedestrian timber bridges are studied and analyzed.

The main motivations of this thesis are:

- the research in this field is still in progress and several researches all around the world are addressing the study of comfort problems, vibrations control, and so forth... in the area of pedestrian bridges;
- the design process of the footbridge should include the effects of the so called Human Induced Loads (HIL) which are presently under investigation in view of an accurate modelling.

After a state of the art focused on contributions to international conferences held in 2014 on this theme and on existing codes and recommendations, two chapters are devoted to the numerical modelling of the structural system and of the actions, respectively.

Model Order Reduction is discussed in the fifth chapter, while the last chapter before these conclusions summarises the proposed passive and active control solutions conceived and designed within this work of thesis.

Two case studies have supported the numerical investigations. Their importance relies on the fact that those structural systems were fully identified by "*in situ*" experimental campaigns, designed and carried out by the author. Those structural systems were designed respectful of a structural code which only prescribes a static design for them. As a result control schemes are quite desirable in view of ensuring (enhancing) the comfort of the crossing persons (serviceability limit state).

In this thesis, however, there is also an effort to build a design procedure accounting for the induced vibrations which should be able to avoid control correction once the footbridge is put in service.

# Appendix A: Technical details for the two footbridges used as case studies

In this chapter some technical details concerning the two footbridges used as case studies are discussed. In particular information on dimensions, materials, constructions techniques and stages, and so forth...are summarized.

In the first section the pedestrian bridge located, in the Trasaghis Municipality and labelled as "Trasaghis footbridge", is reported. In the second session the pedestrian bridge, located in Farra d'Alpago Municipality and labelled as "Tesa footbridge", is discussed.

## A.1 Trasaghis footbridge

In the North-Eastern Italian region of Friuli, there is a mountain area (Gemona Municipality) which became world-wide famous in 1976 when shaken by a strong earthquake in the spring. The event was followed by a further significant shake in the late summer of the same year. Nevertheless, that area recovered from those catastrophic events and its economy is now strong with an important source coming from the tourism.

The local largest natural lake is called the "Lake of the three municipalities" (or the "Cavazzo Lake" – see the red circle in Figure A.1.1) with an emissary channel which was a barrier for the jogging activity across the surrounding park.

Thus, the designer in accordance with the owner authority, decided to link the two side of the outlet channel with a pedestrian bridge (Figure A.1.2). In particular, for both economical and environmental reasons, only eco-friendly materials (i.e. steel and timber) were adopted to harmonize the structure with the surrounding environment. Indeed, as explained later on, the use of reinforced concrete was limited only to the foundation components. From the geometric point of view the span is about 83 m and the double- beam deck width is 4 m, of which 3.22 m represents the free crossing width (Figure A.1.3).

Two main materials were adopted for this structure:

- 1) glued laminated timber (GLT) element of high strength: according to DIN1052 code [1];
- 2) steel elements of high strength: according to UNI EN10025 code [2].



Figure A.1.1. Chorography at scale 1:25000: the map shows the lake and the emissary channel. The red circle indicates the bridge location.



Figure A.1.2. A general view of the "Trasaghis footbridge".

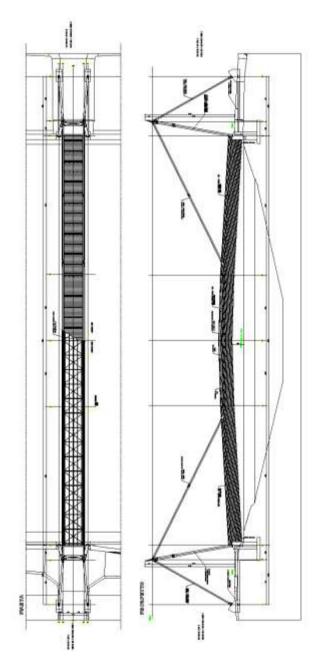


Figure A.1.3. Technical drawings of the footbridge.

Despite the bridge looks of the cable-stayed type (Figure A.1.2), the static scheme sees a vertical antenna on both sides but the usual cables are replaced by tubular steel elements.

The resistant scheme of the deck is made by two lateral main curved GLT beams, of cross-section 20 by 19.41 cm (Figure A.1.4), linked by a sequence of H shape<sup>19</sup> transversal tubular steel elements of type S355JR. The transversal elements that realize the above H shape steel elements are 3.64 m long; each pair of them is braced with *ad hoc* steel elements to ensure the lateral stiffness of the footbridge. Moreover they support five longitudinal timber beams, of cross-section 12 by 16.30 cm, on which the walking tables made by GLT elements, 6.20 cm thick, are mounted and fixed by high-strength screws. More details are represented from Figure A.1.5 to Figure A.1.7.

About the timber materials, glued laminated timber of high strength GL28c is adopted for all structural elements, except for the walking deck made by glued laminated timber of type GL24c (see Figure A.1.6).



Figure A.1.4. Details of the main GLT beams: a) view during the construction stage; b) details of the joint connection.

<sup>&</sup>lt;sup>19</sup> Steel reinforcement: cross-section 22x12x0.63 cm gives the section of the transversal element.

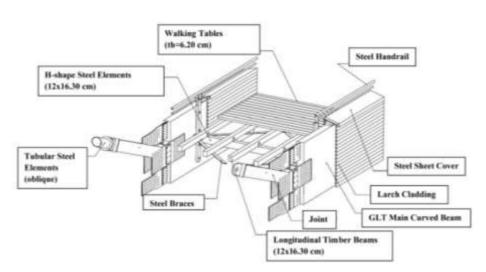


Figure A.1.5. Details of the deck cross-section resistant scheme.



Figure A.1.6. View from above of the structural scheme of the deck.



Figure A.1.7. Detail of the walking table during the construction stage.

For transportation and construction needs, the entire length of the footbridge is subdivided in three sections, each of about 25 m long. *Ad hoc* structural joint connection (Figure A.1.9) were designed and realized at each end of these three parts to allows the final "structural connection" once mounted. They also permit the connection with the steel oblique elements as described later on. It is worth underlining that all these structural elements were realized in factory and mounted *in situ*, limiting in this way the activity on the field , thus making the construction process more automatized (Figure A.1.10).

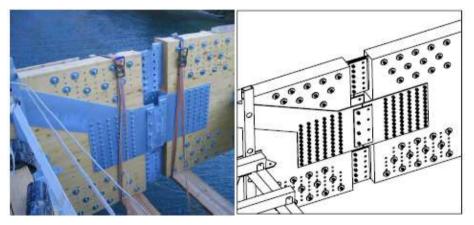


Figure A.1.8. Example of the structural joint connection: detail of the joint.



Figure A.1.9. Example of the structural joint connection: connection of the lateral section in correspondence of the steel oblique tubular element.



Figure A.1.10. Construction stage: assembling the central section of the footbridge. a) before the assembly; b) during the stage.

The two main GLT curved beams are anchored at the thirds (i.e. in correspondence of one of the structural joints described above) to the steel antennas on the two sides. The height of the antennas is about 15 m, and they are made by elements of tubular steel section of external diameter 45.75 cm and thickness 1.42 cm. The link

antenna-beam is made by elements of tubular steel section of external diameter 27.30 cm and thickness 8 mm (Figure A.1.11).

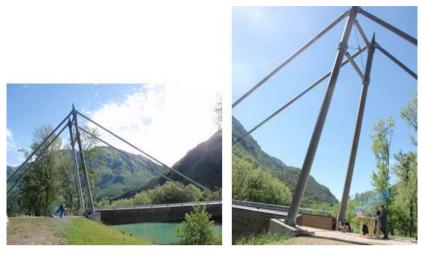


Figure A.1.11. A detail of the tubular steel oblique elements (left). A lateral view of the antenna (right).

Great importance has been given to the durability of the timber elements of this footbridge, especially against environmental, mold and so forth... attach. In fact, as one can see in Figure A.1.12, the external surface of the two main timber beams, are covered by a cladding made by larch planks. Moreover, in its upper side there is a layer of steel material used as water proof system against the rain.

The structural scheme is completed by the *neoprene* supports posed below each ends of the two main curved GLT beam. They allow the thermal deformation of the footbridge. An example of these supports is presented in Figure A.1.13.



Figure A.1.12. View of the upper protection system of the beam made by copper (left). A detail of the external larch cladding (right).



Figure A.1.13. View of the neoprene support.

From the prescriptive point of view, the pedestrian bridge is regarded as category 3 according with the Italian code [3]-[4]. This means that during the design stage beyond the usual "dead" loads, other two "live" loads have to be considered in a static configuration; in particular:

- 1)  $q_{1.d}$  = isolated load of 10 kN over an area of 70 by 70 cm;
- 2)  $q_{1.e}$  = distributed load of 4.0 kN/m<sup>2</sup> resulting from a dense people gathering.

In addition<sup>20</sup> to the above loads, the effects of snow, wind and earthquake were taken into account by the designer.

Once finished the construction stage, the static *proof test* required by the Italian regulation [3]-[4] was carried out. In addition to the dead loads due to the structural and non-structural elements of the footbridge, an additional loads provided by *ad hoc* devices was added (Figure A.1.14) up to the value suggested by the regulation:  $q_{1.e} = 4.0 \text{ kN/m}^2$ . Once loaded the deck, the measurement of the vertical displacement of sensitive points along the footbridge due to the above loads (Figure A.1.15) was carried out. The maximum vertical displacement measured in the middle of the span was 14.5 cm.

<sup>&</sup>lt;sup>20</sup> It is worth noting that for this structure no particular analysis/assessment of comfort was required during the design stage.

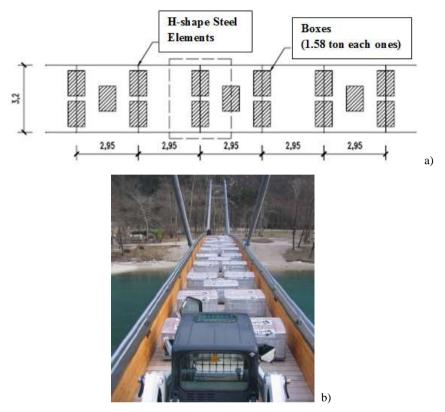


Figure A.1.14. Load test stage. a) Load configuration (measure in *m*); b) during the test (below).



Figure A.1.15. Measurement during the load test stage carried out.

To conclude the description of the "Trasaghis footbridge" some photographs are incorporated (from Figure A.1.16 to Figure A.1.21) to show the current state of the manufact.



Figure A.1.16. Lateral view of the Trasaghis footbridge.



Figure A.1.17. Frontal view of the Trasaghis footbridge.



Figure A.1.18. Detail of the structural joint between the GLT curved beam and the oblique steel tubular element.



Figure A.1.19. View of the reinforced concrete foundation system.



Figure A.1.20. View from above of the walkway. Detail of the H-shape steel element connection with the GLT curved beam.



Figure A.1.21. Detail of the connection between the GLT curved beam and the concrete foundation system.

# A.2 Tesa footbridge

The footbridge reported in this section is located in Farra d'Alpago, an Alpine village close to the town of Belluno (Figure A.2.1). The bridge, built in 2005, connects the two sides of the outlet channel of the "Santa Croce" Lake, where a naturalist area is realized for jogging activities and so forth.

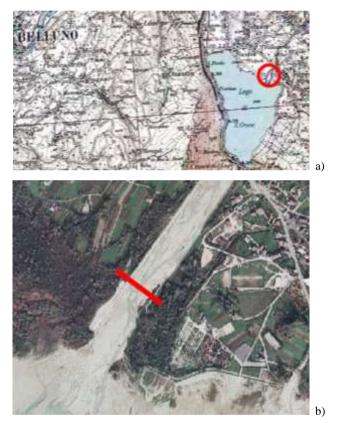


Figure A.2.1. a) Map of the location. The red circle indicates the bridge location. b) Satellite view of the area where the footbridge is located. The red line indicates the footbridge.

Again, to harmonize the structure with the surrounding naturalistic area, only ecofriendly materials (i.e. wood and steel) were adopted during the design stage, limiting the use of reinforced concrete only to the foundations (Figure A.2.2).



Figure A.2.2. Lateral view of the timber footbridge

From the geometric point of view the span is about 110 m and the double- beam deck shows a width of 3.20 m for the free crossing of the pedestrians (Figure A.2.3).

Two main materials are adopted for this structure:

- glued laminated timber (GLT) element of high strength: according to DIN1052 code [1];
- 2) steel elements of high strength: according to UNI EN10025 code [2].

As one can see from Figure A.2.2, the static scheme of the bridge is the classic cablestayed solution. In total, sixteen steel cables are adopted; in particular, eight "internalsecondary" cables of diameter 32 mm, and other eight "external-main" cables of diameter 44 mm (Figure A.2.4 clarifies the meaning of "internal" and "external").

The two antennas, located on both sides of the footbridge, are about 16 m high and realized by steel tubular elements of diameter 45.72 cm and thickness 1.25 cm. Particular care was paid to the design of the connection between the cables and the deck and the antennas (Figure A.2.5).

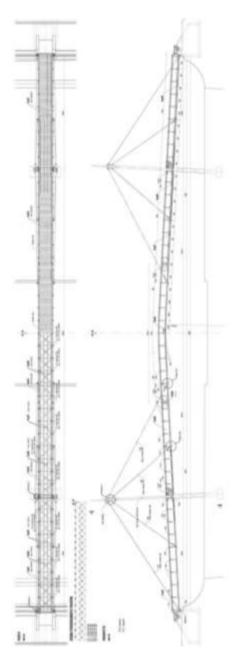


Figure A.2.3. Technical drawings of the footbridge.

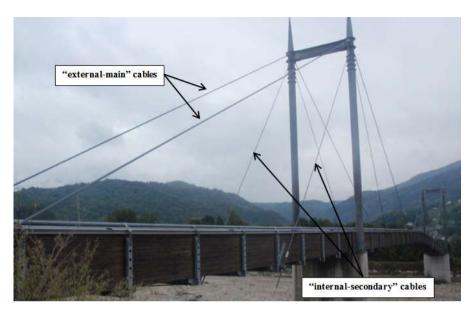


Figure A.2.4. Steel cables.



Figure A.2.5. Detail of the joint connection with the steel cables. Connection with antenna (left); connection with the deck (right).

From the static point of view the deck shows the classical "U-shape" cross section (Figure A.2.6). It is realized by linking two main curved GLT beams, located on the external side of the section, with transversal U-shape steel tubular elements (the vertical ones of dimension 100x120x5 mm, while the horizontal one 100x200x5 mm) of high strength S355J0 according to [2]. Moreover, these elements are connected by steel braces of circular cross-section that provide the lateral stiffness. The structural scheme

is completed mounting, on the "U-shape" elements, four longitudinal GLT rectangular beams (cross section 10x16.3 cm), used to carry the walking deck made of larch planking (thickness 4 cm). More details are reported in Figure A.2.7.

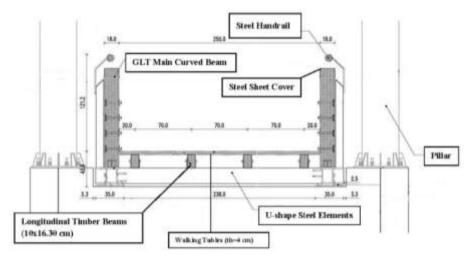


Figure A.2.6. Details of the deck cross-section resistant scheme.



Figure A.2.7. View from the bottom of the deck.

In order to guarantee the thermal deformation of the footbridge, at each ends of the two main GLT beam, *ad hoc* neoprene supports are posed (Figure A.2.8).



Figure A.2.8. Detail of the neoprene support.

The span length can be subdivided into three segments (each of them made of curved GLT-BS14 and GLT-BS16 beams – according to [1]) of length 22.5m, 65m (in the middle) and 22.5 m, respectively. *Ad hoc* structural joint connection (Figure A.2.9) were designed and realized at each ends of these parts to allow the final "structural connection" on field.



Figure A.2.9. Detail of the joints connection.

All these structural elements were realized in factory and mounted *in situ*, limiting in this way the activity on the field, i.e. making the construction process more automatized as possible (Figure A.2.10).



Figure A.2.10. Construction stage: assembling the central section of the footbridge. a) Before the assembly; b) during the stage.

Particular care was finally paid to the protection of the wood specimens; in particular *ad hoc* protective treatments against the attack of environmental agents (i.e. rain, snow, fungus, molds, and so forth) were used.

From the prescriptive point of view, the pedestrian bridge is regarded as category 3 according with the Italian code [3]-[4]. This means that during the design stage beyond the usual "dead" loads, other two "live" loads have to be considered in a static configuration; in particular:

- 3)  $q_{1.d}$  = isolated load of 10 kN over an area of 70 by 70 cm;
- 4)  $q_{1.e}$  = distributed load of 4.0 kN/m<sup>2</sup> resulting from a dense people gathering.

In addition<sup>21</sup> to the above loads, the effects of snow, wind and earthquake were taken into account by the designer as prescribed by the design code.

Once was finished the construction stage the static *proof test* required by the Italian regulation [3]-[4] was carried out. In addition to the dead loads due to the structural and non-structural elements of the footbridge, an additional loads provided by *ad hoc* devices was added until to reach the load provided by the regulations:  $q_{1.e} = 4.0 \text{ kN/m}^2$ . Once loaded the deck, the measurement of the vertical displacement of sensitive points along the footbridge due to the above loads was carried out. The maximum vertical displacement measured in the middle of the span was 14.5 cm (Figure A.2.11).



Figure A.2.11. Survey operations during the load test.

<sup>&</sup>lt;sup>21</sup> It is worth underlining that for this structure no particular analysis/assessment of comfort were required during the design stage.

To conclude the description of the "Tesa footbridge" some photographs are incorporated (from Figure A.2.12 to Figure A.2.18) to show the current state of the footbridge.



Figure A.2.12. Lateral view of the footbridge.



Figure A.2.13. Frontal view of the footbridge.



Figure A.2.14. Detail of the upper protection of the main GLT arcuate beams.



Figure A.2.15. Detail of the internal support made by reinforced concrete.



Figure A.2.16. Cross-section of the deck.





Figure A.2.17. Detail of the connection of the cable with the ground.



Figure A.2.18. View of the walking deck.

# A.3 References

- [1] DIN 1052: Design, Analysis and design of timber structures
- [2] UNI EN 10025: Hot rolled products of non-alloy steel for structural applications. Technical delivery conditions.
- [3] D.M. 16 gennaio 1996 Norme tecniche per le costruzioni in zona sismica [In Italian].
- [4] Norme tecniche per le costruzioni D.M. 14 Gennaio 2008 (NTC2008) [In Italian].

# Appendix B: A standard model for the single pedestrian walking and running

In this chapter a standard model, proposed in the literature for the dynamic behaviour of a single pedestrian walking or running, is presented.

#### **B.1** Dynamic behaviour of a single pedestrian

Different dynamic effects are induced by walking and running respectively. Indeed when the pedestrian is walking there is always one foot in contact with the ground, while during a running there is no contact between the alternate successive touching of one of the two feet (Figure B.1.1). Different persons walk with similar step frequencies [1], into the range 1.25 to 2.3 Hz, due to their similar physiological human constitution, but the step frequencies are influenced by the purpose of the movement and the traffic intensity (Table B.1.1).

The dynamic force induced, by a single pedestrian, can be represented by forces, due to the body mass movement, whose components in the three directions are denoted as putdown, rolling and push-off of the feet. The magnitudes of these forces depend on the step frequency and on the body weight. The lateral component is caused by the movement of the centre of gravity from one foot to the other. This kind of oscillation of the centre of gravity [2] introduces a dynamic force with half the walking frequency (Figure B.1.2). The longitudinal force (along the deck axis) characterises the retarding and the pushing walking period.

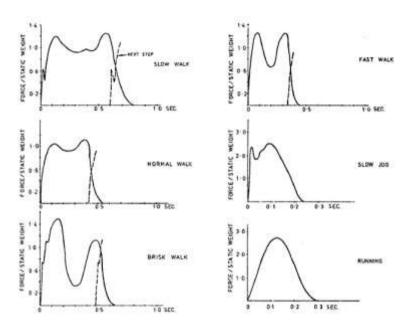


Figure B.1.1. Force trend due to different types of step [1].

Table B.1.1. Range of activities (top table). Typical values for step frequency  $(f_s)$ , velocity  $(v_s)$  and step length  $(l_s)$  (bottom table) [1].

Activity	Group Size N" of people	Range of activity rate. Hz (Footsteps/s)	
		Normal Range	Measured Range
Pedestrian Movements			
Walking	1, 2 and 4	1.6-2.2	1.0-3.0
Jogging	1, 2 and 4	22-32	1.6 - 4.0
Rhythmic Exercises		114	
Jumping	1, 4 and 8	2.0 - 3.0	1.4 - 4.0
Stride Jumps	1, 2 and 4	20-26	1.6-3.4
Running-on-the-spot	1, 2 and 4	22-32	1.4 - 4.0

	$f_{\rm r}$	$v_{z}$	$l_r$
	[Hz]	[m/s]	[m]
slow walking	1.7	1.0	0.60
normal waiking	2.0	1.5	0.75
fast walking	2.3	2.3	1.00
normal running	2.5	3.1	1.25
fast running	>3.2	5.5	1.75

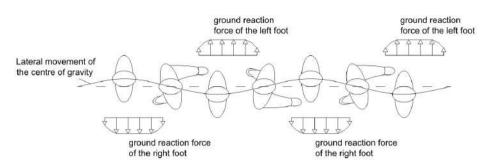


Figure B.1.2. Schematic representation of the lateral forces during the motion [2].

Time domain models are usually adopted for these forces [3]. One assumes that both human feet produce the same forces. The resulting periodic forces are represented by Fourier series as follows:

• vertical (gravity) direction (Figure B.1.3):

$$F_{\nu}(t) = P\left(1 + \sum_{n=1}^{k} \alpha_{n,\nu} \sin(2\pi n f_{\nu} t - \varphi_{n})\right);$$
(B.1)

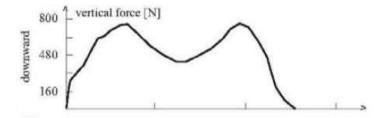


Figure B.1.3. Time history of the human induced force: vertical component [3].

• lateral (transversal) direction (Figure B.1.4):

$$F_{lat}(t) = P \sum_{n=1}^{k} \alpha_{n,\text{lat}} \sin(2\pi n f_{lat} t - \varphi_n) ; \qquad (B.2)$$

(B.3)

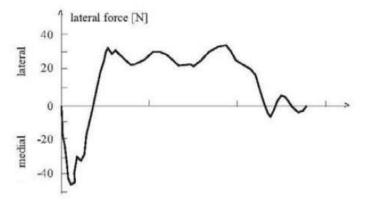


Figure B.1.4. Time history of the human induced force: lateral component [3].

• longitudinal (concordant to the deck axis) direction (Figure B.1.5):

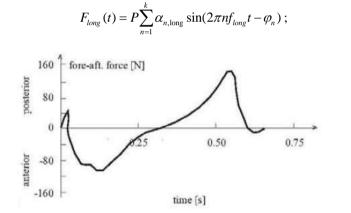


Figure B.1.5. Time history of the human induced force: longitudinal component [3].

where:

- $\alpha_{n,v}, \alpha_{n,\text{lat}}, \alpha_{n,\text{long}}$ : numerical coefficients corresponding to the *n*th harmonic for vertical, lateral and longitudinal direction respectively;
- *P* : static load of the pedestrian;
- *f*: frequency component of repetitive loading (i.e. frequency of the motion);
- $\varphi_n$ : phase angle of the *n*th harmonic;

• *k* : number of harmonics that characterize the forcing function in the frequency range of interest.

The usual value for the static load of the pedestrian is P = 700 N, while the number of the harmonics to be accounted depends on the forces under study. For the vertical component it is reasonable to assume k = 1 (Figure B.1.6), whereas for the two horizontal components it is better to consider k = 3 or 4 (Figure B.1.7).

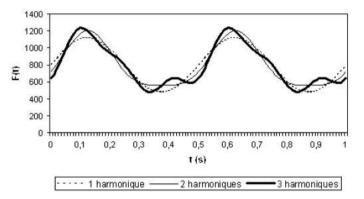
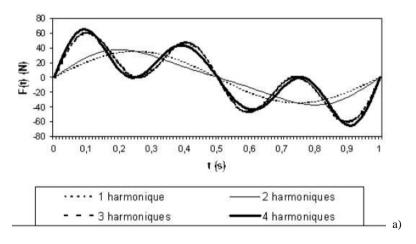


Figure B.1.6. Walking force: vertical component. Differences between three harmonics representation for walking step frequency *f*=2Hz [3].



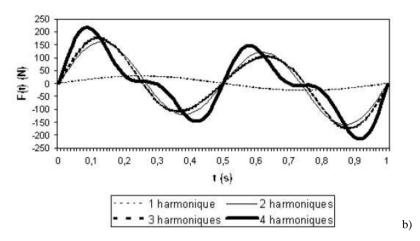


Figure B.1.7. Walking force. a) lateral component; b) longitudinal component. Differences between four harmonics representation for walking step frequency *f*=2Hz [3].

The model of the pedestrian running is more complicated because the running depends more on the individual way of running and the type of shoes. These forces are characterised by a lift-off phase, during which no foot is in contact with the ground. Usually only the vertical component is taken into consideration because it has been observed as the horizontal components, both lateral and longitudinal, have a low "impact" on the dynamic response of the bridge when compared with the vertical one [3].

The vertical component is then modelled by Fourier transform as follows:

$$F_{r,vert}(t) = P + \sum_{n=1}^{k} P_n \sin(2\pi n f_r t) \quad \text{for} \quad (j-1)T_m \le t \le \left(j - \frac{1}{2}\right)T_m$$
  
$$F_{r,vert}(t) = 0 \qquad \qquad \text{for} \quad \left(j - \frac{1}{2}\right)T_m < t \le jT_m$$
(B.4)

where:

- *P* : static load of the pedestrian;
- *j*: the step number, *j*=1,2,..., *m*;
- *f<sub>r</sub>*: running frequency;
- $T_m = \frac{1}{f_r}$ : running period.

In this case, the phase shifts are assumed to be negligible. Three harmonics are usually adopted for the load representation (Figure B.1.8) and the amplitudes of the these harmonics are assumed as follows:  $P_1 = 1.6 P_0$ ;  $P_2 = 0.7 P_0$ ;  $P_3 = 0.2P_0$ .

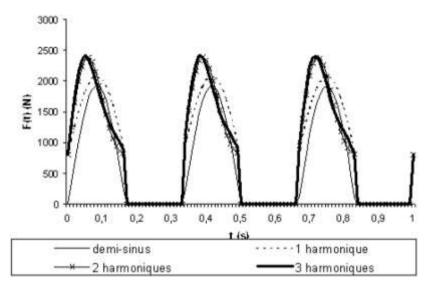


Figure B.1.8. Running force. vertical component. Differences between three harmonics representation for running step frequency *f*=3Hz [3].

Different value of the Fourier coefficients for walking and running have been proposed by various authors and a review in presented in Table B.1.2 [4].

Author(s)	Fourier coefficients / Phase angles	Comment	Type of activity and load direction
Blanchard et al.	<i>a</i> <sub>1</sub> = 0,257		Walking – vertical
Bachmann & Ammann	$a_1 = 0,4 - 0,5; a_2 = a_3 = 0,1$	for <i>f<sub>p</sub></i> = 2,0 - 2,4 Hz	Walking – vertical
Schulze	$a_1 = 0,37; a_2 = 0,10;$ $a_3 = 0,12; a_4 = 0,04;$ $a_5 = 0,015$	for $f_p = 2.0$ Hz	Walking - vertical
Bachmann et al.	$ \begin{aligned} & a_1 = 0, 4/0, 5; \ a_2 = a_3 = 0, 1 \\ & a_1 = a_2 = a_3 = 0, 1 \\ & a_{1/2} = 0, 1; \ a_1 = 0, 2; \ a_2 = 0, 1 \\ & a_1 = 1, 6; \ a_2 = 0, 7; \ a_3 = 0, 3 \\ & \varphi_2 = \varphi_3 = n/2 \end{aligned} $	$f_p = 2,0/2,4$ Hz $f_p = 2,0$ Hz $f_p = 2,0$ Hz $f_p = 2,0$ Hz $f_p = 2,0 - 3,0$ Hz	Walking – vertical Walking – lateral Walking – longitudinal Running – vertical Walking – vertical & lateral
Kerr	$a_1, a_2 = 0,07; a_3 = 0,2$	a <sub>1</sub> is frequency dependant	Walking - vertical
Young	$\begin{array}{l} a_1 = 0,37 \ (f_\rho - 0,95) \leq 0,5 \\ a_2 = 0,054 + 0,0088 \ f_\rho \\ a_3 = 0,026 + 0,015 \ f_\rho \\ a_4 = 0,01 + 0,0204 \ f_\rho \end{array}$	Mean values for Fourier coefficients	Walking - vertical
Charles & Hoorpah	$a_1 = 0.4$ $a_1 = 0.05$ $a_1 = 0.2$		Walking – vertical Walking – lateral Walking - longitudinal
EC5, DIN1074	$a_1 = 0,4; a_2 = 0,2$ $a_1 = a_2 = 0,1$ $a_1 = 1,2$		Walking – vertical Walking – lateral Jogging – vertical
Synpex findings	$\begin{array}{l} a_1 = 0,0115 f_s^2 + 0,2803 \ f_s - \\ 0,2902 \\ \varphi_1 = 0 \\ a_2 = 0,0669 f_s^2 + 0,1067 \ f_s - \\ 0,0417 \\ \varphi_2 = -99,76 f_s^2 + 478,92 \ f_s - \\ 387,8 \ [\circ] \\ a_3 = 0,0247 \ f_s^2 + 0,1149 \ f_s - \\ 0,1518 \\ \text{If } f_s < 2,0 \ \text{Hz} \\ \varphi_2 = - \\ 1431,35 \ f_s + 819,65 \ f_s^2 - \\ 1431,35 \ f_s + 811,93 \ [\circ] \\ \text{If } f_s > 2,0 \ \text{Hz} \\ \varphi_p = 813,12 \ f_s^3 - \\ 8505,9 \ [\circ] \\ a_4 = -0,0039 \ f_s^2 + 0,0285 \ f_s - \\ 846 = -0,0039 \ f_s^2 - 0,0285 \ f_s - \\ 0,0082 \\ \varphi_8 = 34,19 \ f_s - 65,14 \ [\circ] \end{array}$	Fourier coefficients and phase angles of step-by-step load model which represents mean human ground reaction forces	Walking - vertical

Table B.1.2. Different Fourier coefficients for walking and running [4].

### **B.2** References

- [1] Guidelines for the design of footbridges. Fib bulletin 32, November 2005.
- [2] Heinemeyer, C., Butz, C., Keil, A., Mark, S., Goldack, A., Trometer, S., Lukić, M., Chabrolin, B., Lemaire, A., Martin, P., Cunha, Á., Caetano, E., 2009. Design of Lightweight Footbridges for Human Induced Vibrations. Background document in support to the implementation, harmonization and further development of the Eurocodes, JRC European Commission, Luxembourg.
- [3] Footbridges: assessment of vibrational behaviour of footbridges under pedestrian loading Technical guide, 2006 Sétra-AFGC.
- [4] Živanović, S. et al., 2005. Vibration serviceability of footbridges under humaninduced excitation: a literature review. Journal of Sound and Vibration 279, pp. 1-79.

# Appendix C: Overview of some existing codes and guidelines

In this chapter more details on the guidelines introduced in *Chapter 2* about the comfort evaluation of footbridges are given.

#### C.1 ISO 10137

ISO 10137 [1] provides a series of principles for the prediction of the vibrational characteristics of a structure during the design stage. To assess the effects of the vibrations, it is important to specify to the type of vibration source, its location and the receiving element. The assessment of the response is evaluated by a numerical model. The document gives the designer the choice of the comfort of level to be considered, but states that pedestrian bridges must be so that the amplitude of the vibrations do not cause discomfort to users. In the absence of more definitive data, the code suggests the following maximum levels of vibration:

- in the vertical direction (z-axis): the vibrations should not exceed those obtained by a multiplicative factor of 60 in the selected reference system to the relevant base curve represented in Figure C.1.1. When one or more persons are at rest on the walkway the multiplicative factor of 30 should be applied.
- horizontal vibrations induced by pedestrian traffic or wind should not exceed 60 times the base curve for the horizontal direction (x and y- axis), see Figure C.1.2.

For the calculation of root-mean-square values of the acceleration, an averaging time of  $I \ s$  is recommended. Furthermore the following scenarios have to be considered during the calculation:

• one person walking across the walkway and another (the receiver) standing at mid-span;

- an average pedestrian flow based on a daily occurrence rate, e.g. a group size of 8 to 15 people, depending on the length and the width of the walkway;
- the presence of streams of pedestrians (significantly more than 15 persons);
- occasional festive or choreographic events (when relevant).

The document also suggests the dynamic forces produced by the pedestrians that has to be accounted in the design phase. The load induced by a singular pedestrian can be evaluated as:

• vertical direction: 
$$F_{\nu}(t) = Q\left(1 + \sum_{n=1}^{k} \alpha_{n,\nu} \sin\left(2\pi n f_{\nu} t + \varphi_{n,\nu}\right)\right),$$
 (C.1)

• transversal direction: 
$$F_h(t) = Q\left(1 + \sum_{n=1}^k \alpha_{n,h} \sin\left(2\pi n f_h t + \varphi_{n,h}\right)\right).$$
 (C.2)

The dynamic action of a group of *N* people is evaluated simply multiplying the above forces by the coefficient C(N) (see Table C.1.1); that is:

$$F_N(t) = F(t) * C(N).$$
(C.3)

The details about the above formulas, coefficients and so forth... are not here reported for brevity.

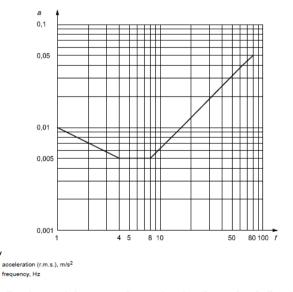


Figure C.1.1. Building vibration z-axis base curve for acceleration (foot-to-head vibration direction) [1].

Ke

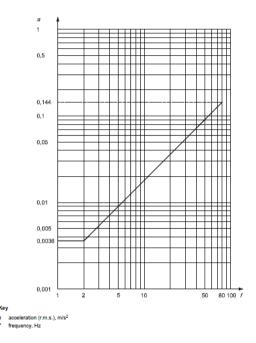


Figure C.1.2. Building vibration x- and y-axis base curve for acceleration (side-to-side and back-tochest vibration direction) [1].

Coordination	1st harmonic	2nd harmonic	3rd harmonic
high	0,80	0,67	0,50
medium	0,67	0,50	0,40
lów	0,50	0,40	0,30

#### C.2 Fib-buletin 32

The guidelines [2] provide an available tool for the design of the footbridges, in particular with reference to the assessment of the maximum acceleration to be compared with the comfort limit value. For the dynamic analysis it is important to define the density of pedestrians (expressed as [pers/m<sup>2</sup>], which in turn influences the speed of pedestrians. Based on these guidelines the density is calculated as:

$$q = \frac{\lambda}{v_s \cdot b_{eff}}, \qquad (C.4)$$

$$q = \frac{N_r}{L \cdot b_{eff}} , \qquad (C.5)$$

where  $\lambda$  is the rate of pedestrian arrival [pers/s],  $v_s$  the velocity of pace [m/s],  $b_{eff}$  the effective width [m],  $N_r$  the number of pedestrians on bridge deck and L the length of bridge [m]. It was observed in [3] that in a pedestrian flow with a density varying between 0.3 and 0.6 pers/m<sup>2</sup>, pedestrians are able to move undisturbed (Figure C.2.1). If the density increases, the single pedestrian is no longer able to move undisturbed but she/he is forced to modify her/his frequency and speed. In general the speed decreases with increasing density as reported in Figure C.2.2.

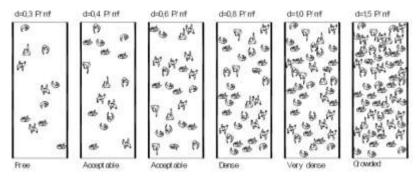


Figure C.2.1. Different scenarios of pedestrian densities [2].

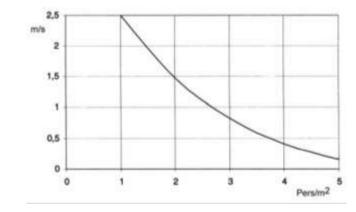


Figure C.2.2. Velocity versus density of pedestrian [2].

The first calculation method proposed by [4] and reported within this guideline is based on the resonant response of a *1*-DOF oscillator, and determines the maximum vertical acceleration resulting from the passage of a pedestrian who walks or runs with a frequency equal to the natural frequency of the bridge. The maximum acceleration can be evaluated as:

$$a = \phi \left( 4\pi^2 f^2 y \alpha \right), \tag{C.6}$$

where *f* is the vertical natural frequency of the bridge [Hz], *y* the static deflection at mid-span for a force of 700 N [m],  $\alpha$  the Fourier coefficient of the relevant harmonic of the walking or running rate,  $\phi$  the dynamic amplification factor (Figure C.2.3).

Another model is based on the experience by [5]. The working hypothesis of this method are:

- 1) single-degree-of-freedom (SDF) oscillator with stationary excitation;
- 2) step frequency  $f_s$  = natural frequency of the bridge f;
- 3) coefficient 0.6 due to changing excitation point.

In the above assumptions the maximum acceleration in  $[m/s^2]$  is defined as:

$$a = 0.6 \frac{\alpha G}{0.5M} \frac{\pi}{\delta} \left( 1 - e^{-n\delta} \right), \tag{C.7}$$

where G is the dead load of an ideal pedestrian (0.7 kN), M the mass in [t] of the equivalent 1-DOF oscillator for single span,  $\delta$  the logarithmic decrement and n the number of cycles per span (span over step length).

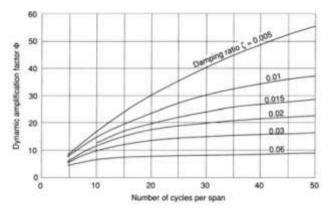


Figure C.2.3. Dynamic amplification factor for resonant response due to sinusoidal force moving across simple span [2].

The guidelines give moreover indications about vandalism loading, or repetitive jumping in order to excite the natural frequency of a bridge, can lead to very high values

of acceleration. Are suggested maximum limit of acceleration for footbridges of 0.7 g to 0.8 g. As already observed in section 1.2 this acceleration should be used during the design stage of the structure but not as a threshold for comfort. To prevent the lock-in effects the guideline suggests a threshold in lateral accelerations of 0.08  $m/s^2$  for a frequency of 1 Hz.

# C.3 BS 5400

The British Standard BS5400, BD 29/04 and BD 37/01 [6] state that all the footbridges should satisfy the following minimum requirements:

- in the vertical direction the natural frequencies of the structure must have a value greater than 5 Hz;
- in the transverse horizontal direction the minimum frequency is fixed at 1.5 Hz.

If the fundamental frequency of vertical vibration, said  $f_0$ , however, is less than or equal to 5 Hz, the maximum vertical acceleration  $a_v$  (expressed as m/s<sup>2</sup>) of any part of the bridge has to be limited to:

$$a_{\nu} \le 0.5\sqrt{f_0}$$
 . (C.8)

The maximum vertical acceleration may be calculated either by simplified method or by rigorous method. For complex structures, the maximum vertical acceleration must be calculated assuming that the dynamic load induced by a single pedestrian is that due to a concetrated force *F*, which moves through the span of the bridge at a constant speed  $v_c = 0.9 f_0$  [m/s], where:

$$F = 180\sin\left(2\pi f_0 t\right). \tag{C.9}$$

If the fundamental frequency of the horizontal vibration is less than 1.5 Hz, great attention has to be paid to avoid the possibility to activate the lateral movement of the deck (the so called *lock-in* effects). The codes, however, do not propose any method to evaluate the maximum horizontal (transversal) acceleration.

# C.4 Sétra-AFGC

These guidelines [7] present the main aspects that one has to consider during the design of the footbridge to avoid resonant phenomena in order to improve, or guarantee, an adequate level of comfort. Moreover these guidelines can be used in order to evaluate the comfort of existing bridges. The methodology of working is summarized in the flowchart in Figure C.3.1.

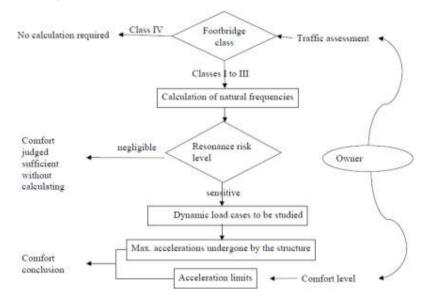


Figure C.3.1. Methodology flowchart proposed by Sétra-AFGC [7].

Operationally the proposed method can be summarized in the following five steps:

- 6) *step 1*: determination of the footbridge class;
- 7) *step 2*: choice of comfort level;
- 8) *step 3*: evaluation of the dynamic footbridge properties (if needed);
- 9) *step 4*: dynamic evaluation (if needed);
- 10) *step 5*: modification of the structural design to fulfil the comfort requirement (if needed).

About the classification of the footbridges, based on the expected (or measured) level of pedestrian traffic, it possible to determine four classes:

- Class IV: seldom used footbridge, built to link sparsely populated areas or to ensure continuity of the pedestrian footpath across motorway or express lane areas.
- Class III: footbridge for standard use, that may occasionally be crossed by large groups of people but that will never be loaded throughout its bearing area.
- Class II: urban footbridge linking up populated areas, subjected to heavy traffic and that may occasionally be loaded throughout its bearing area.
- Class I: urban footbridge linking up high pedestrian density areas (for instance, nearby presence of a rail or underground station) or that is frequently used by dense crowds (demonstrations, tourists, etc.), subjected to very heavy traffic.

The owner has to determine the footbridge class. It is always possible to select a higher class to ensure/increase the comfort when needed. Class IV footbridges are considered not to require any calculation to check the comfort performance. Anyway for very light footbridges belonging to Class IV, at least Class III should be considered to ensure a minimum level of comfort for the users.

The three levels of comfort can be defined as follows:

- *maximum comfort*: accelerations are "imperceptible" to the users;
- *average comfort*: accelerations are merely perceptible to the users;
- *minimum comfort*: under seldom loading configurations, the accelerations undergone are perceived by the users, but do not become intolerable.

The choice of the comfort level is normally influenced by the attitude of the persons<sup>22</sup> using the footbridge and by its level of importance according to *step 1*. The level of comfort is evaluated in reference to the acceleration undergone by the bridge using different dynamic load cases. Due to the subjective nature of the comfort concept, the guidelines present the comfortable limits in terms of four acceleration thresholds, see Table C.3.1 and Table C.3.2, for vertical and horizontal accelerations respectively. The first three ranges correspond to the maximum, mean and minimum comfort levels described in the previous section; while the 4th range corresponds to an uncomfortable situation.

<sup>&</sup>lt;sup>22</sup> In this choice the designer should be more demanding on behalf of particularly sensitive users as schoolchildren, disabled people and so forth..., and more tolerant in case of all those footbridges rarely used by pedestrians.

For the footbridges belonging to the Class as from I to III it is necessary to evaluate the dynamic features of the bridge (i.e. natural vibration frequency). These frequencies for the vertical, transverse horizontal and longitudinal horizontal direction, have to be determined in two different cases: empty footbridge (no pedestrian over the deck) and footbridge loaded by pedestrians (700 N/m<sup>2</sup>) throughout its bearing area. The ranges in which these frequencies are situated, see Table C.3.3 and Table C.3.4, make possible the assessment of resonance risk. Four ranges can be established: range 1 corresponds to the maximum risk of resonance, while range 4 corresponds to a negligible risk of resonance. In cases where the risk of resonance is considered negligible after the above check, comfort level is automatically considered sufficient.

Table C.3.1.Acceleration ranges (in m/s<sup>2</sup>) for vertical vibrations proposed by Sétra-AFGC [7].

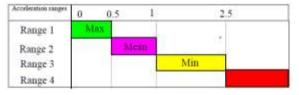
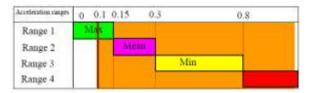
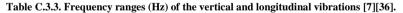
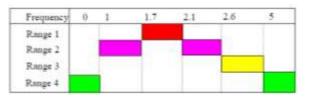
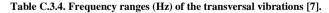


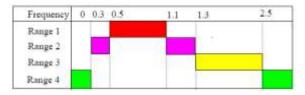
 Table C.3.2. Acceleration ranges (in m/s²) for horizontal vibrations proposed by Sétra-AFGC. The acceleration is limited in any case to 0.10 m/s² to avoid lock-in effect [7].











If the above evaluation shows the possibility of resonance risk a more detailed analysis must to be carried out (*step 4*). In this case three different cases of dynamic loads have to be considered:

#### • Case 1: sparse and dense crowds.

This case regards the study of the pedestrian bridges belonging to Class II and III. The number of pedestrians involved, within the hypothesis crowd uniformly distributed over the deck, is equal to:

$$N = S * d , \qquad (C.10)$$

in which *S* indicates the deck area, and *d* pedestrian density. For Class III footbridges the density *d* of pedestrian must be taken equal to 0.5 pedestrians/m<sup>2</sup>; while d = 0.8 pedestrians/m<sup>2</sup> for Class II footbridges. The equivalent number of pedestrians  $N_{eq}$  (all in phase and at the same frequency) able to produce the same effects on structure as random pedestrian is:

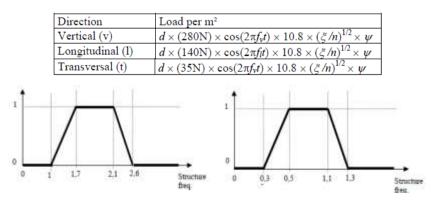
$$N_{eq} = 10.8\sqrt{N\xi}$$
, (C.11)

where  $\xi$  is the critical damping ratio (Table C.3.5).

Table C.3.5. Typical critical damping ratio [7].

Type	Critical damping ratio
Reinforced concrete	1.3%
Pre-stressed concrete	1%
Mixed	0.6%
Steel	0.4%
Timber	1%

Table C.3.6. Case 1 - Loads per unit of area proposed by Sétra-AFGC. Below: factor  $\psi$  for vertical and longitudinal vibrations on the left, and for lateral vibrations on the right [7].



The loads summarized in Table C.3.6, have to be applied to the whole of footbridge, and the direction of the loads must agree with the direction of the mode under study in order to maximize the response in terms of vibration/accelerations of the bridge (Figure C.3.2).

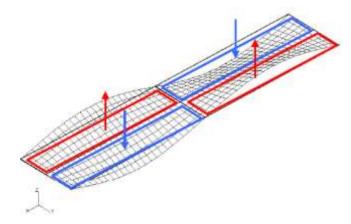


Figure C.3.2. Example of loads application: mode shape versus load direction [7].

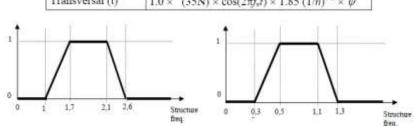
#### • Case 2: very dense crowds.

This case regards the study of Class I footbridges. In this application the density of pedestrian is assumed to be d = 1.0 pedestrians/m<sup>2</sup>. The equivalent number of pedestrians  $N_{eq}$  is:

$$N_{eq} = 1.85\sqrt{N}$$
 . (C.12)

The forces per unit of area to be applied in this situation are summarized in Table C.3.7.

 Table C.3.7. Case 2 - Loads per unit of area proposed by Sétra-AFGC. Below: factor  $\psi$  for vertical and longitudinal vibrations on the left, and for lateral vibrations on the right [7].



#### • Case 3: effect of the second harmonic of the crowd.

This case has to be taken into account only for Class I and II footbridges. It is similar to the two cases just presented, but in this case the second harmonic of the stresses caused by the pedestrian walking must also be accounted. The pedestrian density is assumed d = 1.0 pedestrians/m<sup>2</sup> and d = 0.8 pedestrians/m<sup>2</sup> for Class I and II respectively. Once again the crowd is considered to be uniformly distributed, but the individual force exerted by a pedestrian is reduced to 70N vertically, 7N transversally and 35N longitudinally. The correction factor  $\psi$  is presented in Figure C.3.3.

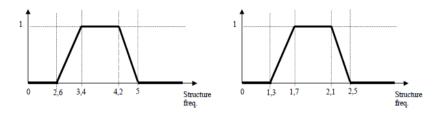


Figure C.3.3. factor  $\psi$  for vertical and longitudinal vibrations on the left, and for lateral vibrations on the right [7].

If the above calculation shows that the comfort level are not fulfilled, it is necessary to review the design of the footbridge (changing the natural frequency of the structure, or its stiffness), if we are in the design stage, or introducing *ad hoc* devices to reduce the vibration.

### C.5 The comfort level in existing recommendations

To conclude this part related to the comfort for the footbridges (and civil structures in general), in Table C.3.1, Table C.3.2 and Figure C.3.1 a summary of the recommendations provided by the main international codes in terms of frequencies and acceleration limits, respectively, are presented. The synthesis is taken from reference [7].

100 3631	1.9- 11	X = Nundamental natural frequency of the bridge
ALEC Guide 11	0.5	
Eurocode 1	$Min \begin{cases} 0.50 \sqrt{f_{A}} \\ 0.70 \end{cases}$	for /= 1 to 3 Hz for /= 55 Hz, check dependent on case it on /+ 5 Hz, no check necessary
Difu-Fachbandit 102	$0.5 \cdot \sqrt{f_j}$ , sut.	Sec 7: 5 5 Hz; 7: n fundamental natural tequency of the unloaded bridge
VDI 2057	0.6 • $\sqrt{f_1}$ .vet. 0.214 .net.	E = fundamental natural Requirecy of the bridge
SEA	0.39	
85 5400	0.5. 17,	A + fundamental natural Requirecy of the bridge
Ontario Bridge Cade ON/T83	$0.25 \cdot f_1^{\pm in}$	7. = fundamental matural frequency of the origin
Europode 5 (ENV 1985-2)	0.7	
Bachmann (40)	0.5 - 1.0	
Japanese Footbridge Devign Code (1979)	1,0	
Laberal acceleration $\sigma_{\rm Lines}$ [HeV]		
Eurocode 1	$3 dim \begin{cases} 0.14 \sqrt{f_0} \\ 0.13 \end{cases}$	fau f = 0.5 to 1.5 Hz for f = 1.5-2.5 Hz sheck dependent on case from f = 2.5 Hz no-check neutrality
Eurocode 5 (EPIV 1995-2)	0.2	for f < 2.5 Hz (for standing individuals)

# Table C.3.1. Overwiev of the limit value of the acceleration according to the main international codes [7].

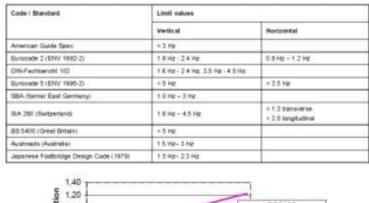


Table C.3.2. Overwiev of the critical frequencies according to the main international codes [7].

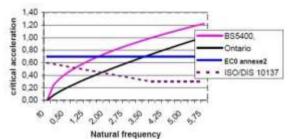


Figure C.3.1. Vertical critical accelerations (m/s<sup>2</sup>) versus natural frequency (Hz) by various codes [7].

# C.6 References

- [1] ISO 10137, Bases for design of structures Serviceability of buildings and walkways against vibration, 2007.
- [2] Guidelines for the design of footbridges. Fib bulletin 32, November 2005.
- [3] Oeding, 1961. Verkehrsanalyse Fußgänger, Stadt Stuttgart Planungsamt, Abt. Verkehr (in German).
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- [5] Grundmann H., Kreuzinger H., Schneider M., 1993. Schwingungsuntersuchungen f
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- [6] BS5400, Part 2, Appendix C, Vibration Serviceability Requirements for Foot and Cycle Track Bridges. British Standards Institution, 1978.
- [7] Footbridges: assessment of vibrational behaviour of footbridges under pedestrian loading Technical guide, 2006 Sétra-AFGC.

# Appendix D: Crowd Load Model (CLM): experimental data set

In this chapter more details about the experimental data used for the validation of the crowd load model (see section 4.2) are given.

# D.1 Time-Frequency analysis of the signals: theoretical background

The experimental campaign carried out on the "Tesa" footbridge (see section 3.1.1) is supported to provide the required information for the CLM model validation.

To better represent the data in order to reach a better resolution in terms of robustness and efficiency, the analysis is performed following the time-frequency procedure (the so called Short Time Fast Fourier Transform – STFFT) proposed by Choi-Williams (CW) [1]. This choice allows to identify the frequency variation over the time that can be associated to the interaction between the users (i.e. pedestrians motion over the deck) and the footbridge. From another point of view this approach identifies exactly the time at which one particular mode/frequency is activated. Furthermore, it is able to show visually the shift of the frequencies from lower to higher values; that is the transfer of energy from lower to higher modes, can be detected.

Following the procedure proposed by Cohen [2], let t denote the time and  $\omega$  the pulsation, the time-frequency distribution can be written as:

$$P(t,\omega) = \frac{1}{4\pi^2} \int \int \int_{-\infty}^{+\infty} \left[ e^{-i\theta(t-u)} \Phi(\theta,\tau) \right] e^{-i\tau\omega} R_{t,1}(t,\tau) du d\tau d\theta \quad (D.1)$$

where:

- *u* and  $\tau$  replace *t*,  $\theta$  replace  $\omega$  when integrating;
- $\Phi(\theta,\tau) = e^{-\frac{\theta^2 \tau^2}{s}}$  with  $(s \in R)$ , is the kernel function of CW distribution;

•  $R_{t,1}(t,\tau) = x(u+\tau/2)x^*(u-\tau/2)$  is the autocorrelation function in which x(t) is the input signal, *t* the time and  $u = t + \tau/2$ .

Introducing now the Fourier transform W(t-u) for the above kernel function in equation (D.1):

$$P(t,\omega) = \frac{1}{4\pi^2} \int_{-\infty}^{+\infty} W(t-u) e^{-i\tau\omega} R_{t,1}(t,\tau) du d\tau; \qquad (D.2)$$

As an example the time-frequency analysis is performed for the test carried out on "Tesa2 footbridge, and labelled as *test B3* (see section 3.1.1). The main results for the data acquired by sensor WSa2 (see Figure 3.1.3) along the transversal (Y) and gravity (Z) axis are summarized in Figure D.1.1 and Figure D.1.2 respectively.

During this test, only one frequency clearly appears at 1.35 Hz, and it is constant over the entire duration of the data acquisition. Another frequency appears at 5.00 Hz in the time period between 50 and 70 sec for the right-to-left crossing, and in the time period from 100 to 160 sec for the left-to-right crossing. This plot shows many vertical lines; in particular, one can observe that in the time period between 125 and 140 sec, there is a consistent frequency shift, due to the overlapping of the waves associated to the movement of the pedestrians from the sides toward the mid-span. Also along the Z axis only one frequency clearly appears at 2.00 Hz, and it is activated in the time period between 50 and 80 sec for the right-to-left crossing, and between 100 and 160 sec for the left-to-right crossing. Moreover one more frequency of 6.00 Hz seems to be "slightly" activated between 50 and 70 sec for the right-to-left crossing, and between 130 and 150 sec for the left-to-right crossing.

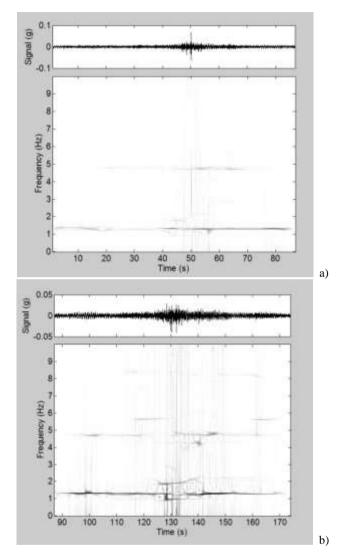


Figure D.1.1. Six persons walking. Time-frequency plots from the data recorded by sensor WSa2 along transversal (*Y*) axis. a) data from right-to-left crossing; b) data from left-to-right crossing.

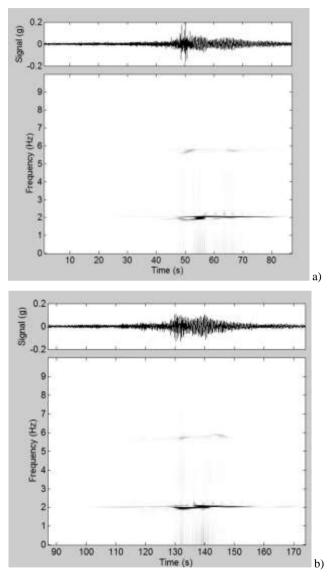


Figure D.1.2. Six persons walking. Time-frequency plots from the data recorded by sensor WSa2 along gravity (Z) axis. a) data from right-to-left crossing; b) data from left-to-right crossing.

# **D.2** Experimental data analysis

To validate the proposed CLM model, in necessary to identify a set of experimental data that will be compared with the numerical simulation. By the way, in this section, the data acquired during the *tests B* presented in section 3.1.1 are analyzed and discussed. To give an easier interpretation of these data, each record is classified into two different sets: crossing from the right to the left side of the footbridge (labelled as RtoL), and from the left to the ride side (labelled as RtoL) ([3] and [4]).

To have a clear idea about the acquired data, the numerical analysis of the data are performed following the time-frequency technique (STFFT) presented in the previous section.

Here below the results achieved by the analysis are discussed.

#### test B1

This test consists of collecting records of the accelerations of the footbridge due to a round trip along the deck. The mass of the person is 85 Kg. Further details about this test are summarized in Table D.2.1. The results achieved for the measurements taken by sensor WSa2 along the transversal Y and vertical (gravity) Z axes are plotted in Figure D.2.1 and Figure D.2.2 respectively.

Туре	ID
Day/Time	November 7,2013 / 2.48 pm
Test type	walking (round trip)
Air temperature	15°C
Pedestrian(s) num.	1 person
Total Pedestrian(s) mass	85 Kg
Total time duration	≈2:30 min
<i>RtoL</i> time duration	≈1:15 min
<i>LtoR</i> time duration	≈1:15 min
Average velocity of the pedestrian(s)	1.45 m/s

Table D.2.1. Walking (round trip) of one person.

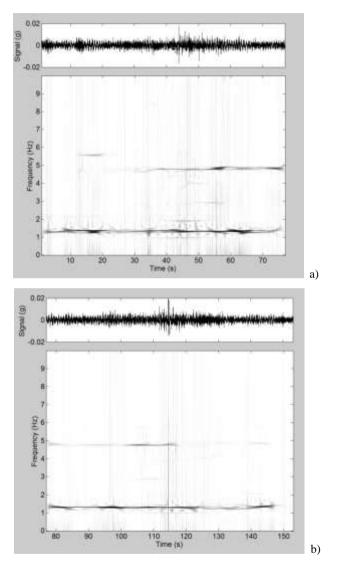


Figure D.2.1. Time-frequency plot from the data taken by sensorWSa2 along Y transversal axis during *test Ia.* a) *RtoL* data ; b) *LoR* data

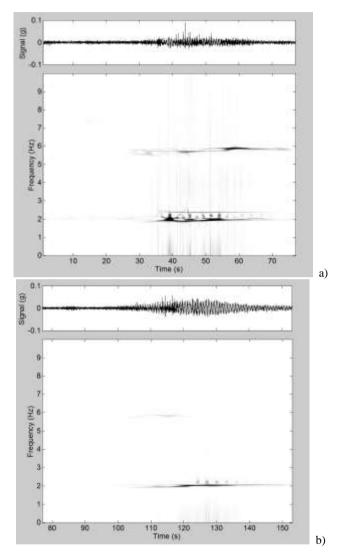


Figure D.2.2. Time-frequency plots from the data taken by sensor WSa2 along Z gravity axis during *test Ia*. a) *RtoL* data ; b) *LtoR* data

The following three main frequencies (average values) appear along *Y* axis:

- 1.35 Hz: constant over the time;
- 5.00 Hz: it activates only during the time period from 30 to 70 sec for the *RtoL*, and from 80 to 120 sec for the *LtoR*. This means that this particular

frequency is detected when the pedestrian is located about 15-20 m before the position of the sensor WSa2;

• 5.80 Hz: it activates only during the time period from 10 to 32 sec for the *RtoL*.

Several vertical lines (mostly located between 35 and 60 sec), that represent the shifts in terms of frequencies, appear during the *RtoL* crossing, while they disappear during the *LtoR* crossing. These shifts can be related to noise into the signal due to the overlapping of the waves associated to the movement of the pedestrian from the sides towards the mid-span.

Instead, along Z axis two main frequencies (average values) appear:

- 2.00 Hz: it activates during the time periods from 35 to 70 sec for the *RtoL*, and from 105 to 150 sec for the *LtoR*. This means that this particular frequency is detected while the pedestrian is located about 10 m before the position the sensor WSa2 and about 40 m after it;
- 6.00 Hz: it activates only during the time period from 30 to 70 sec for the *RtoL*. Once again, this means that this particular frequency is detected while the pedestrian is located about 10 m before the position the sensor WSa2, and about 40 m after it. During the *LtoR* crossing, this frequency almost disappears.

One can observe that, also for the results along the Z axis, the time-frequency plot for the *LtoR* crossing is more "clear/clean" with respect to the *RtoL* crossing. Moreover, the time-frequency plots along the Z axis show, in general, a clear trend compared with the ones along the transversal Y axis.

#### - test B2

The records of a round trip of four persons along the deck are considered. The average mass is 75 Kg. The pedestrians proceed in pairs by keeping a distance of 1 m between each other in the longitudinal direction of the footbridge, and a distance of about 0.5-0.7 from the closest lateral edge of the footbridge (Figure D.2.3). More details are summarized in Table D.2.2.

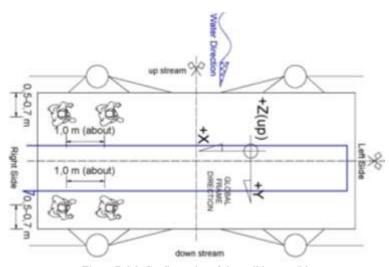


Figure D.2.3. Configuration of the walking test B2.

Type	ID
Day/Time	November 7,2013 / 2.55 pm
Test type	walking (round trip)
Air temperature	15°C
Pedestrian(s) num.	4 persons
Total Pedestrian(s) mass	300 Kg
Total time duration	≈2:45 min
<i>RtoL</i> time duration	≈1:23 min
<i>LtoR</i> time duration	≈1:20 min
Average velocity of the pedestrian(s)	1.33 m/s

Table D.2.2. Walking (round trip) of four persons.

The results achieved for the measurements taken by sensor WSa2 along the transversal *Y* and vertical (gravity) *Z* axes are plotted in Figure D.2.4 and Figure D.2.5 respectively.

Along Y axis two main frequencies (average values) appear:

- 1.35 Hz: constant over the time;
- 5.00 Hz: it activates during the time period from 40 to 70 sec for the *RtoL*, and from 80 to 140 sec for the *LtoR*. This means that this particular frequency is

detected when the pedestrian is located about 15-20 m before the position of the sensor WSa2.

One can observe, moreover, that several vertical lines (mostly located between 46 and 60 sec and between 120 and 130 sec), which represent the frequency shifts, appear during both the *RtoL* and the *LtoR* crossings. Furthermore one more frequency of about 2.00 Hz appears at about 130 sec up to the end of the record. All these facts can be related, as previously observed, to the overlapping of the waves associated to the movement of the pedestrians from the sides towards the mid-span of the bridge.

Along Z axis the results appear are more "clear/clean", and in particular only two main frequencies (average values) are highlighted:

- 2.00 Hz: it activates during the time period from 35 to 70 sec for the *RtoL*; and from 110 to 160 sec for the *LtoR*. This mean that this particular frequency is detected while the pedestrian is located about 10 m before the position of sensor WSa2, and about 40 m after it;
- 6.00 Hz: it activates during the time period from 30 to 70 sec for the *RtoL* only. This means that this particular frequency is detected while the pedestrian is located about 10 m before the position of sensor WSa2, and about 40 m after it. During the *LtoR* crossing this frequency almost disappears.

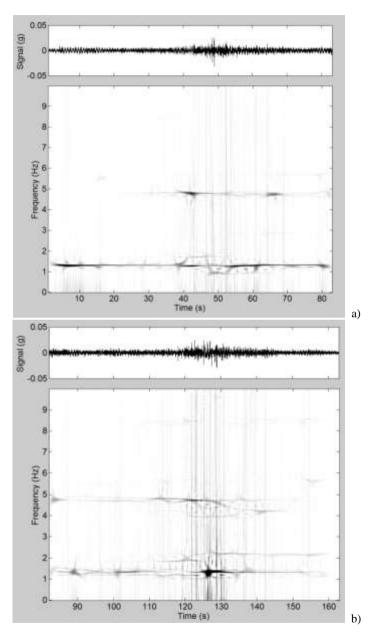


Figure D.2.4. Time-frequency plots from the data recorded by sensor WSa2 along the Y transversal axis during *test Ib.* a) *RtoL* data ; b) *LtoR* data.

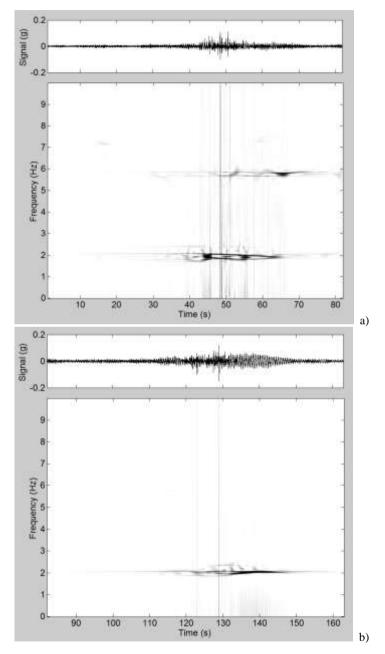


Figure D.2.5. Time-frequency plots from the data recorded by sensor WSa2 along Z gravity axis during *test Ib.* a) *RtoL* data ; b) *LtoR* data.

#### - test B3

In this test, the records of a round trip along the deck of six persons are considered. The average mass is 80 Kg. The pedestrians proceed in pairs by keeping a distance of 1 m between each other in the longitudinal direction of the footbridge, and a distance of about 0.5-0.7 from the closest lateral edge of the footbridge (Figure D.2.6). Further details about this test are summarized in Table D.2.3.

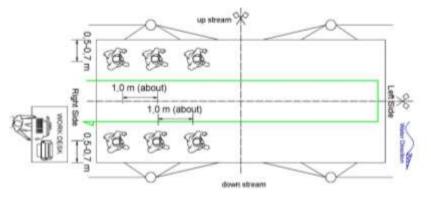


Figure D.2.6. Configuration of the walking test B3.

Type	ID
Day/Time	November 7,2013 / 3.00 pm
Test type	walking (round trip)
Air temperature	15°C
Pedestrian(s) num.	6 persons
Total Pedestrian(s) mass	480 Kg
Total time duration	≈2:42 min
<i>RtoL</i> time duration	≈1:21 min
LtoR time duration	≈1:21 min
Average velocity of the pedestrian(s)	1.35 m/s

Table D.2.3. Walking (round trip) of six persons.

During this test, only one frequency clearly appears at 1.35 Hz along *Y*, and it is constant over the entire duration of the data acquisition. Another frequency appears at 5.00 Hz in the time period between 50 and 70 sec for the *RtoL* crossing, and in the time period from 100 to 160 sec for the *LtoR* crossing. Once again, the *LtoR* plot shows many vertical lines; in particular, one can observe that in the time period between 125 and 140

sec, there is a consistent frequency shift, due to the overlapping of the waves associated to the movement of the pedestrians from the sides toward the mid-span (Figure D.2.7). Also along the Z axis only one frequency clearly appears at 2.00 Hz, and it is activated in the time period between 50 and 80 sec for the *RtoL* crossing, and between 100 and 160 sec for the *LtoR* crossing. Moreover one more frequency of 6.00 Hz seems to be "slightly" activated between 50 and 70 sec for the *RtoL* crossing, and between 130 and 150 sec for the *LtoR* crossing (Figure D.2.8).

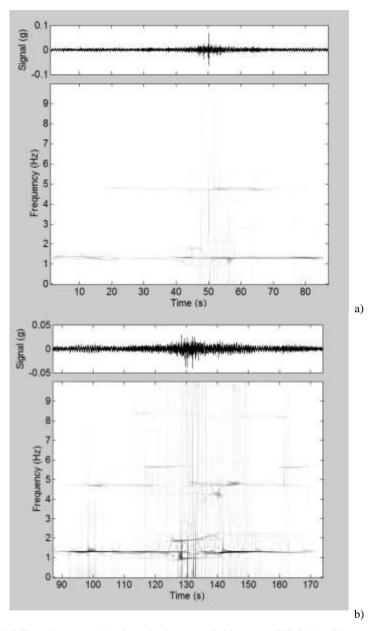


Figure D.2.7. Time-frequency plots from the data recorded by sensor WSa2 along Y transversal axis during *test Ic*. a) *RtoL* data ; b) *LtoR* data.

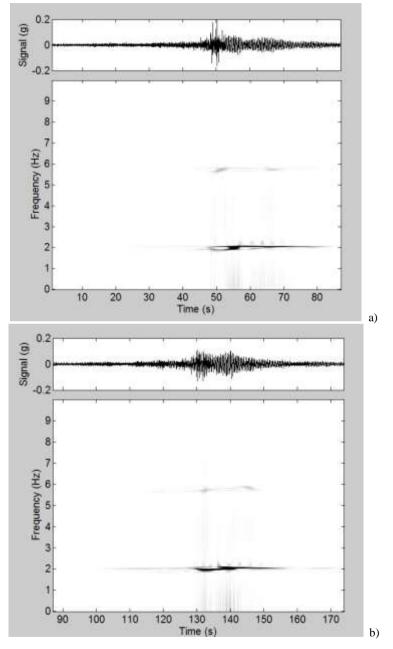


Figure D.2.8. Time-frequency plots from the data recorded by sensor WSa2 along Z gravity axis during *test Ic.* a) *RtoL* data ; b) *LtoR* data.

#### - test B4

During this test, the footbridge response under a running person along the deck (round trip) is recorded. The average mass is 85 Kg. Further details about this test are summarized in Table D.2.4.

Туре	ID
Day/Time	November 7,2013 / 3.15 pm
Test type	running (round trip)
Air temperature	15°C
Pedestrian(s) num.	1 person
Total Pedestrian(s) mass	85 Kg
Total time duration	≈0:46 min
<i>RtoL</i> time duration	≈0:23 min
<i>LtoR</i> time duration	≈0:23 min
Average velocity of the pedestrian(s)	4.78 m/s

Table D.2.4. Running (round trip) of one person.

In Figure D.2.9, the time-frequency plots from the data acquired by sensor WSa2 along the transversal *Y* axis are reported. In this test, only one frequency clearly appears at 1.35 Hz during the *LtoR* crossing. During the *RtoL* crossing, only one vertical line appears at about 35 sec, and it can be associated to one "jump" of the pedestrian in the proximity of sensor WSa2. Moreover, one can observe that during the *LtoR* crossing some frequency shifts appear between 70 and 85 sec. Once again, these shifts are associated to the overlapping of the waves associated with the motion of the pedestrian.

In Figure D.2.10, the plot for the data acquired by WSa2 along the vertical (gravity) Z axis is reported. Also in this direction only one frequency clearly appears at 2.00 Hz during the *LtoR* crossing. During the *RtoL* crossing a frequency shift appears at about 35 sec, and can be associated to one "jump" of the pedestrian in the proximity of sensor WSa2, as observed for the *Y* axis.

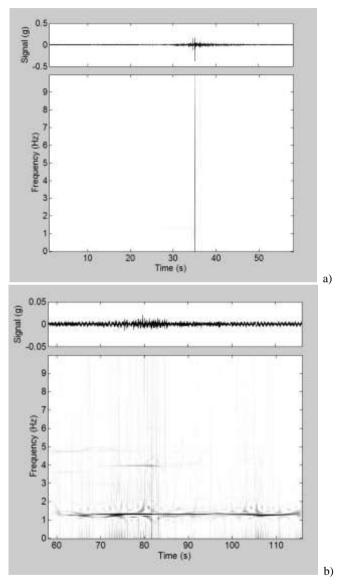


Figure D.2.9. Time-frequency plots from the data recorded by sensor WSa2 along Y transversal axis during *test II*. a) *RtoL* data ; b) *LtoR* data.

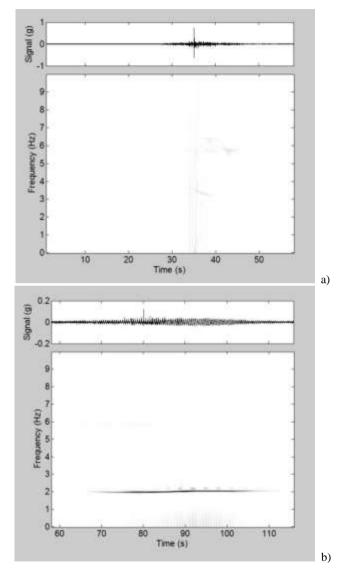


Figure D.2.10. Time-frequency plot - WSa2 along Z gravity axis - test II. a) RtoL data ; b) LtoR data.

As a general remark of these analysis, it appears as for the data along Y show several frequencies are highlighted compared to the signal along Z axis. Another aspect that can be highlighted is this: during the *LtoR* walking "some" frequencies shifts in Y and in Z directions disappear.

Further investigation regards the singular value decomposition (SVD) of the spectral density matrix. The results are shown in Figure D.2.11; in particular a zoom between 1.00 Hz and 5.00 Hz is plotted.

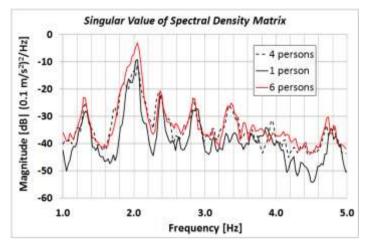


Figure D.2.11. Zoom between 1.00 Hz and 5.00 Hz of the singular value of spectral density matrix. Comparison between acquisitions along transversal Y axis from WSa1 during the test B1 (solid black line) and test B2 (dashed black line) and test B3 (solid red line).

One can see that the trend of the singular value (SV) related to the *test B3* is (almost) always "higher" than the one related to the *test B1* and *test B2*. Higher values of SV indicate that many frequencies can be activated, and so it will be more difficult to identify the real value of the frequency.

### D.3 References

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- [2] L. Cohen, 1995. Time-frequency Analysis. Electrical engineering signal processing Prentice Hall Signal Processing series (1st edition 1995).
- [3] Bortoluzzi D., Casciati S., Faravelli L., 2014. Testing the effects of walking and running on an existing timber pedestrian bridge. On the Proceeding of the 9th International Conference on Structural Dynamics, EURODYN 2014 Porto, Portugal, 30 June - 2 July 2014.

[4] S. Casciati, F. Casciati, L. Faravelli and D. Bortoluzzi, 2014. Modelling The Human Induced Vibrations In A Cable-Stayed Pedestrian Timber Bridge. On the Proceeding of the 7th European Workshop on Structural Health Monitoring – EWSHM2014, July 8-11, 2014. La Cité, Nantes, France.

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Valdenogher, dicembre 2014.

Daniele Bortoluzzi

"Without deviation from the norm, progress is not possible." (Frank Zappa)